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**A-Atlas-103**

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Charts of Shockwave, Expansion Wave and  
Skin Friction Data for Hypersonic Speeds

1.0 Summary

This report presents curves of shockwave, expansion wave and boundary layer data plotted for use at Mach numbers up to 20. Normal and oblique shockwave parameters, conical shock parameters, expansion wave parameters, some aerodynamic coefficients and compressible laminar and turbulent boundary layer skin friction formulas are given for Mach numbers up to at least 20. The use of these curves is briefly described in the text.

2.0 Symbols

$C_D$  - wave drag coefficient =  $\frac{\text{Wave Drag}}{qS}$

$C_f$  - local skin friction coefficient =  $\frac{\tau}{q}$

$C_{f_r}$  - mean skin friction coefficient =  $\frac{\text{Drag}}{qS}$

$C_N$  - normal force coefficient =  $\frac{\text{Normal Force}}{qS}$

$C_p$  - coefficient of specific heat at constant pressure BTU/lb.°R

$C_x$  - axial drag coefficient =  $\frac{\text{Axial Drag}}{qS}$

$g$  - acceleration due to gravity ft./sec.<sup>2</sup>

$h$  - heat transfer coefficient BTU/ft.<sup>2</sup>sq.ft.°R

$H$  - total pressure lbs./ft.<sup>2</sup>

$J$  - mechanical equivalent of heat ft.-lb./BTU

$k$  - thermal conductivity BTU/ft. hr.°R

$l$  - wetted length ft.

$l'$  - effective starting length ft.

$M$  - Mach number

$N$  - Nusselt number =  $\frac{h l}{k}$

$p$  - static pressure lb./ft.<sup>2</sup>

$Pr$  - Prandtl number =  $\frac{\mu C_p}{k}$

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$Q$  - heat flux  
BTU/hr. ft.<sup>2</sup>  
 $q$  - dynamic pressure  
lb./ft.<sup>2</sup>  
 $Re$  - Reynolds number =  $\frac{\rho u l}{\mu}$   
 $S$  - reference area  
ft.<sup>2</sup>  
 $T$  - static temperature  
°R  
 $T'$  - reference temperature  
°R  
 $T_i$  - recovery or inherent temperature  
°R  
 $u$  - velocity  
ft./sec.  
 $x$  - axial length  
ft.

Greek

$\alpha$  - angle of attack  
degrees  
 $\beta$  - shockwave angle  
degrees  
 $\gamma$  - ratio of specific heat  
 $\delta$  - flow deflection angle, boundary layer thickness  
degrees, ft.  
 $\theta$  - boundary layer momentum thickness  
ft.  
 $\theta_v$  - cone semi-vertex angle  
degrees  
 $\theta_w$  - cone shockwave half angle  
degrees  
 $\mu$  - coefficient of viscosity  
lb. sec./ft.<sup>2</sup>  
 $\nu$  - Prandtl-Meyer angle  
degrees  
 $\rho$  - density  
lb. sec.<sup>2</sup>/ft.<sup>4</sup>  
 $\tau$  - shear stress  
lbs./ft.<sup>2</sup>

Subscripts

1 - initial or free stream  
2 - after the shock  
S - surface of the cone  
∞ - sea level



$W$  - at the wall

$\omega$  - local

$i$  - incompressible or location index

### 3.0 Discussion and Results

The need for a combined collection of basic supersonic aerodynamic data has long been evident to Convair Aerodynamicists working with high velocity bodies. The existing charts and tables (Reference 1, 2, 3) are either limited in some phase of Mach number and flow deflection angle or the data are not presented graphically (Reference 4). It was believed also that some convenient reference should be provided for the skin friction and heat transfer formulas associated with the reference temperature method explained in Reference 9. This report fulfills these needs.

The sections following will deal with the separate types of curves presented.

#### 3.1 Inviscid Flow Parameters

The curves in this section are separated into one, two and three dimensional flow. The flow parameters presented, all versus initial Mach number, are Mach number after the shock, static pressure ratio, static temperature ratio, density ratio, velocity ratio, dynamic pressure ratio, Reynolds number ratio, total head ratio and shockwave angle. All ratios are quantities after the wave divided by the value of the same quantity before the wave.

Three coefficients are given for both unyawed and yawed cones.  $C_N$ ,  $\frac{dC_N}{d\alpha}$  and  $C_x$  are given for cones in Newtonian flow (Reference 4, 5, 6).

The Reynolds numbers presented in Figure 1.6, 2.8 and 3.8 were found from

$$\frac{Re_2}{Re_1} = \frac{\rho_2 u_2}{\rho_1 u_1} \left( \frac{T_1}{T_2} \right)^{0.7}$$

where an exponential viscosity - temperature relation

$$\frac{\mu_2}{\mu_1} = \left( \frac{T_2}{T_1} \right)^{0.7}$$

was assumed.

The data for these curves were obtained from Reference 1 through Reference 6 or were computed for high Mach number-deflection angle combinations when they were not tabulated.

### 3.2 Viscous Flow Parameters By The Reference Temperature Method

The curves presented in this section are primarily for use with the reference temperature method of calculating heat transfer and skin friction. Figures 4.1 - 4.7 include: local and mean incompressible flow skin friction coefficients for laminar, transition and turbulent flow vs. Reynolds number (Reference 7); reference temperature ratio  $T'/T$  vs. Mach number; and reference temperature  $(T')^{1/4}$  vs.  $T'$ . For use in the heat balance equation, the inherent (recovery) temperature rise  $\Delta T_i$  is given versus velocity. The theoretical laminar boundary layer stability curve (Reference 8) is shown in Figure 4.5 to indicate the type of boundary layer flow which might be expected for various  $M$ ,  $T_w/T$  and  $Re$  combinations. Figures 4.6 and 4.7 present the NACA-NBS variation of  $C_p$  with  $T$  and variation of  $\mu$  with  $T$  as given by the Sutherland viscosity rule.

The reference temperature method and its associated equations will be presented below in outline form. A complete derivation of these equations may be found in Reference 9, which also contains a critical evaluation of the method. The concept of an "effective length", needed for the application of this method to bodies other than flat plates, is discussed by Sieff (Reference 10) and Romig (Reference 11). Therefore the following sections are primarily on the use of the method in obtaining the heat transfer and skin friction.

The reference temperature method is based on the assumption that the use of a characteristic temperature in the compressible equations for drag and heat transfer will eliminate their outward dependence on temperature and Mach number, i.e., convert them to the incompressible, constant-property equation. Under this assumption the  $T'$  method is thus described as: if all temperature-dependant properties in the incompressible equations for heat transfer and drag are based on the reference temperature  $T'$  then the incompressible equations will yield the compressible friction or heat transfer coefficients. Use of this method therefore obviates solving the compressible laminar and turbulent boundary layer equations. The equations for the reference temperature are

$$\begin{aligned} T' &= T_\infty \left[ 0.42 + 0.032M_\infty^2 + 0.58 \frac{T_w}{T_\infty} \right], M_\infty \leq 5.5 \\ T' &= T_\infty \left[ 0.70 + 0.023M_\infty^2 + 0.58 \frac{T_w}{T_\infty} \right], M_\infty \geq 5.5 \end{aligned} \quad (1)$$

where the ratio  $T'/T_\infty$  may be found in Figure 4.2 vs. Mach number for integral values of  $1 \leq T_w/T_\infty \leq 4$ . The eq. (1) are valid for both laminar and turbulent flow.

### 3.2.1 Basic Equations for Two-Dimensional Flow

The compressible skin friction may be obtained either as the ratio of compressible to incompressible,  $C_{f_c}/C_{f_i}$ , or explicitly as a function of the Reynolds number based on  $T'$ . For practical considerations explicit relationships will be developed first for the local and mean compressible friction.

Assume all air properties in the incompressible drag equations are based on  $T'$  so that according to the assumption stated in 3.2,

$$(\text{compressible}) C_{f_\infty} q_\infty \equiv \tau \equiv (\text{incompressible}) C_{f_i}' q'$$

where the primes denote that air properties are based on  $T'$ .

or

$$C_{f_\infty} = C_{f_i}' \frac{\rho'}{\rho_\infty}$$

Using the perfect gas equation at constant pressure,

$$C_{f_\infty} = \frac{T_\infty}{T'} C_{f_i}' \quad (2)$$

where  $C_{f_i}'$  is given either by the Blasius formula

$$\text{LAMINAR } C_{f_i}' = \frac{0.664}{\sqrt{\frac{\rho' u_\infty l}{\mu'}}} \quad (3)$$

or the Karman equation

$$\text{TURBULENT } (C_{f_i}')^{-1/2} = 1.7 + 4.15 \log \frac{\rho' u_\infty l}{\mu'} C_{f_i}' \quad (4)$$

These laminar and turbulent incompressible friction coefficients (Eq. 3 & 4) are plotted in Figure 4.1 versus Reynolds number. The Reynolds number to be used with Eq. (3) & (4) and hence in evaluating Eq. (2) is a function of  $T'$ . The expression for  $Re_{T'}$  is given as

$$Re_{T'} = \frac{\rho' u_\infty l}{\mu'}$$

or, using the power law for viscosity,  $\mu \propto (T)^{0.7}$  and using ambient conditions as a reference,

$$Re_{T'} = \left( \frac{T_{\infty}}{T'} \right)^{1.7} Re_{\infty} \quad (5)$$

If the perfect gas law is used and sea level ( $T = 518.4^{\circ}R$ ) conditions are used in the power law,

$$Re_{T'} = 12.44 \times 10^4 \frac{p_{\infty} u_{\infty} l}{(T')^{1.7}} \quad (6)$$

Thus it is only necessary to enter Figure 4.1 with  $Re_{T'}$ , computed by Eq. (5) or Eq. (6), locate the desired  $C_{f_i}'$  and compute  $C_{f_{\infty}}$  by Eq. (2). For convenience in computing  $Re_{T'}$ , the function  $(T')^{1.7}$  is plotted versus  $T'$  in Figure 4.3.

The mean skin friction coefficients may be obtained in much the same way from the drag equation. For both laminar and turbulent flow

$$C_{F_{\infty}} = \frac{T_{\infty}}{T'} C_{F_i}' \quad (7)$$

where  $C_{F_i}'$  is given by the Blasius formula

$$\text{LAMINAR } C_{F_i}' = \frac{1.328}{\sqrt{Re_{T'}}} \quad (8)$$

and the Prandtl-Schlichting equation

$$\text{TURBULENT } C_{F_i}' = \frac{0.455}{(\log_{10} Re_{T'})^{2.58}} \quad (9)$$

Eqs. (8) and (9) are also plotted versus Reynolds number in Figure 4.1. The same procedure is used in evaluating the mean coefficient as was used for the local.

The compressible heat transfer rate can be obtained from the definition of the heat equation

$$Q = h (T_i - T_w) \quad (10)$$

where the heat transfer coefficient is defined as

$$h = \frac{Nu k}{L} \quad (11)$$

If the air properties in  $h$  are based on  $T'$  and the following incompressible relationships for Nusselt number are used,

$$\text{LAMINAR } Nu_m = 0.332 \sqrt{Pr'} \sqrt{Re_T'} \quad (12)$$

and

$$\text{TURBULENT } Nu_m = 0.0296 (Re_T')^{0.8} \quad (13)$$

then after considerable algebraic manipulation the following heat transfer equations are found

$$\text{LAMINAR } h = 0.0074 \left[ \frac{P_{\infty} U_{\infty}}{L} \right]^{0.5} \quad (14)$$

$$\text{TURBULENT } h = \frac{0.02498 (P_{\infty} U_{\infty})^{0.8}}{L^{0.2} (T')^{0.51}} \quad (15)$$

It was assumed in Eq. (14) and (15) that the air properties varied in the following way

$$\frac{\mu}{\mu_{\infty}} = \left( \frac{T}{T_{\infty}} \right)^{0.7}, \quad \frac{k}{k_{\infty}} = \left( \frac{T}{T_{\infty}} \right)^{0.85}, \quad Pr = 0.72 \quad \text{and} \quad p = pRT \quad (15a)$$

The validity of Eq. (14) in particular and the exponents in the power laws were substantiated in the investigations of Reference (9). It was found that these heat transfer equations are correct as long as the boundary layer air is not dissociated.

Curves of  $\Delta T_1 = T_1 - T_1$ , where  $T_1$  is the free stream static temperature, are plotted versus free stream velocity in Figure 4.4a - 4.4c for use in eq. (10).

Thus to compute the compressible laminar or turbulent heat, it is only necessary to find the reference temperature and solve eq. (14) or (15). These values of  $h$  are used along with the temperatures obtained from Figure 4.4a - 4.4c in eq. (10) to find the heat rate  $Q$ .

### 3.2.2 Two Dimensional Flow With Discontinuities

The equations (1) - (15) written above apply in strict sense only to flat plate flow. Any condition which would tend to make the external flow discontinuous at a point, such as transition from laminar to turbulent flow, or passing through a shockwave or expansion, renders the equations inapplicable.

In order to use eq. (1) - (15) it is necessary, therefore, to somehow refer the boundary layer characteristics at the point of discontinuity to a flat plate boundary layer with identical characteristics but with no flow discontinuities. This can be accomplished by forcing, as it were, the momentum loss of the boundary layer flow to remain constant across any discontinuity. This momentum-loss method will give an effective starting length for the boundary layer characteristics to the point of discontinuity.

The momentum loss equations are discussed at length in Reference (10) and (11) and only a brief outline of them are given here. If the momentum loss through the boundary layer is defined as

$$\int_0^{\delta} \rho u (u_{\infty} - u) dy$$

then across any point of discontinuity it is assumed that

$$\int_0^{\delta_1} \rho u (u_1 - u) dy = \int_0^{\delta_2} \rho u (u_2 - u) dy \quad (16)$$

This is equivalent to stating that.

$$\theta_1 q_1 = \theta_2 q_2 \quad (17)$$

where  $\theta$  is the boundary layer momentum thickness and  $q$  the stream dynamic pressure. The laminar or turbulent equations for  $\theta$  allow (17) to be solved for  $\ell_2$ , the effective starting length. For convenience in nomenclature  $\ell_2$  is designated as  $\ell'$  hereafter.

Therefore let

$$\theta_{LAM} = \frac{1.328}{\left(\frac{\rho_0 u_0 \ell}{\mu_0}\right)^{1/2}} \left(\frac{C_F}{C_{Fi}}\right)_{LAM} \ell \quad (18)$$

and

$$\theta_{TURB} = \frac{0.072}{\left(\frac{P_{\infty} u_{\infty}}{\mu_{\infty}}\right)^{0.2} \left(\frac{C_F}{C_{F_i}}\right)_{TURB}} L \quad (19)$$

where  $L$  is the length along the body to the point of discontinuity. The turbulent expression for  $\theta$  is based on the  $1/7$ th power law for velocity. It is adequate for  $5 \times 10^5 \leq Re \leq 10^7$ . The more exact solution (Eq. (9)) would not be amenable for computation of  $L'$ . These equations giving  $\theta$  can be simplified by making the same assumptions used in the heat equations (14) and (15) in the skin friction derivation to get

$$\left(\frac{C_F}{C_{F_i}}\right)_{LAM} = \left(\frac{T_{\infty}}{T_i}\right)^{0.15} \quad \& \quad \left(\frac{C_F}{C_{F_i}}\right)_{TURB} = \left(\frac{T_{\infty}}{T_i}\right)^{0.66} \quad (20)$$

Substitution of (20) into (18) and (19) and use of Eq. (15a) gives

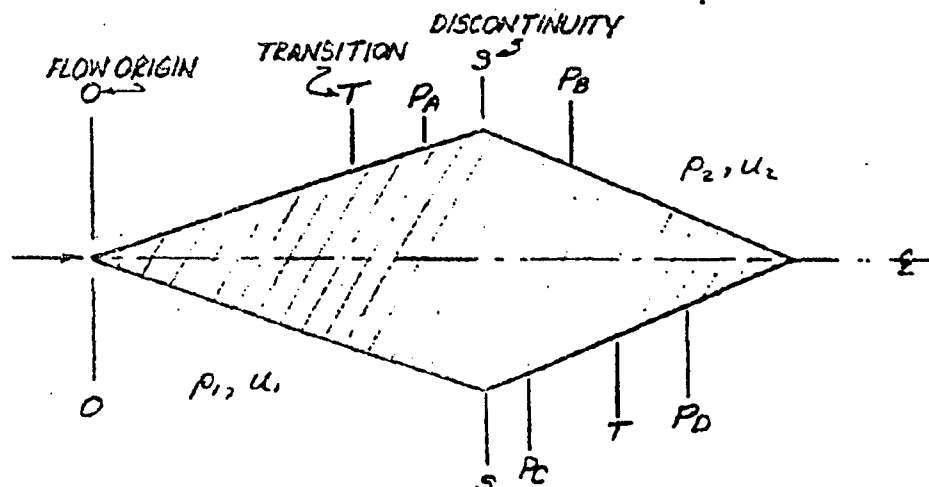
$$\theta_{LAM} = 0.003765 \left[ \frac{P_{\infty} u_{\infty}}{(T_{\infty})^{1.7}} \right]^{-0.5} \left( \frac{T_{\infty}}{T_i} \right)^{0.15} L^{0.5} \quad (21)$$

and

$$\theta_{TURB} = 0.006892 \left[ \frac{P_{\infty} u_{\infty}}{(T_{\infty})^{1.7}} \right]^{-0.2} \left( \frac{T_{\infty}}{T_i} \right)^{0.66} L^{0.6} \quad (22)$$

These equations can then be used in (17) to obtain  $L'$ .

Two types of flow discontinuities can be encountered in two dimensional flow. These are (a) transition from laminar to turbulent flow and (b) shockwave or an expansion. The equations for  $L'$  for combinations of these cases are obtained from Eq. (17) - (22) and are summarized in the following table for flow on an infinite double-wedge airfoil, a typical two-dimensional body.



The following effective starting lengths are given to the points  $P$  of interest

$$\text{TO } P_A \quad L'_A = \overline{TP}_A + 0.4696 \left[ \frac{p_1 u_1}{(T_1')^{1.7}} \right]^{-0.375} (\overline{OT})^{0.625} \quad (23)$$

$$\text{TO } P_B \quad L'_B = \overline{SP}_B + (\overline{SP}_A + L'_A) \frac{p_1}{p_2} \left( \frac{u_1}{u_2} \right)^{1.5} \left( \frac{T_2'}{T_1'} \right)^{0.825} \quad (24)$$

$$\text{TO } P_C \quad L'_C = \overline{SP}_C + \overline{OS} \frac{p_1}{p_2} \left( \frac{u_1}{u_2} \right)^3 \left( \frac{T_2'}{T_1'} \right)^{0.3} \quad (25)$$

$$\text{TO } P_D \quad L'_D = \overline{TP}_D + (L'_C + \overline{PC}T)^{0.625} \cdot 0.4696 \left[ \frac{p_2 u_2}{(T_2')^{1.7}} \right]^{-0.375} \quad (26)$$

Eq. (23) - (26) may then be used to evaluate the  $L'$ , which in turn can be used in Eq. (2) - (15) along with local conditions to evaluate the desired coefficients. Since Eq. (1) is independent of length this method does not involve iteration.



### 3.2.3 Three Dimensional Flow

Since the boundary layer at any point on a cone is much thinner than the boundary layer of a flat plate which has otherwise the same boundary layer characteristics as the cone, the Eq. (2) - (15) do not apply to conical flow unless some correction is made in order to correlate the boundary layer thicknesses. This correction can be made by working directly with the three-dimensional boundary layer equations in the laminar case. Hantschke and Wendt (Reference 12) found that the conical equations transformed directly into the two dimensional equations if the factor  $1/3$  were inserted in the cone length. Therefore, for laminar flow, if

$$\text{LAMINAR: } l_{\text{cone}} = \frac{1}{3} l_{\text{FLAT PLATE}} \quad (27)$$

then Eq. (2) - (25) (with slight modification in the mean friction coefficient) can be used. Van Driest (Reference 13) made the same type of investigation of the three-dimensional turbulent equations and found that the effective starting length for a cone in fully turbulent flow is

$$\text{TURBULENT } l_{\text{cone}} = \frac{1}{2} l_{\text{FLAT PLATE}} \quad (28)$$

Summarized below, for convenience, are eq. (2) - (15) for conical flow. The length used is the actual cone wetted length.

$$\text{laminar } Re_{T_1} = 4.147 \times 10^4 \frac{\rho_s u_s l}{(\tau')^{1.7}} \quad (29)$$

$$\text{turbulent } Re_{T_1} = 6.22 \times 10^4 \frac{\rho_s u_s l}{(\tau')^{1.7}} \quad (30)$$

$$\text{Local skin friction (unchanged) } \tau_{f_w} = \left( \frac{\tau_w}{\tau'} \right) \tau'_{f_i}$$

$$\text{mean laminar } C_{F_w} = \frac{2}{3} \cot \theta_v \left( \frac{\tau_w}{\tau'} \right) C'_{F_i} \quad (31)$$

referred to base area of cone

$$\text{mean turbulent } C_{F_w} = 1.02 \cot \theta_v \left( \frac{\tau_w}{\tau'} \right) C'_{F_i} \quad (32)$$

referred to base area of cone  
(using the  $1/5$  or  $1/7$  power law for  
gives identical results)

$$\text{laminar } K = 0.1281 \left( \frac{\rho_s u_s}{l} \right)^{0.5} \quad (33)$$

$$\text{and turbulent } h = \frac{0.02866 (p_s u_s)^{0.8}}{(L)^{0.2} (T')^{0.51}} \quad (34)$$

Eq. (29) - (34) are then solved in the same manner as the equations (2) - (15) for two dimensional flow. —

### 3.2.4 Three Dimensional Flow with Discontinuities

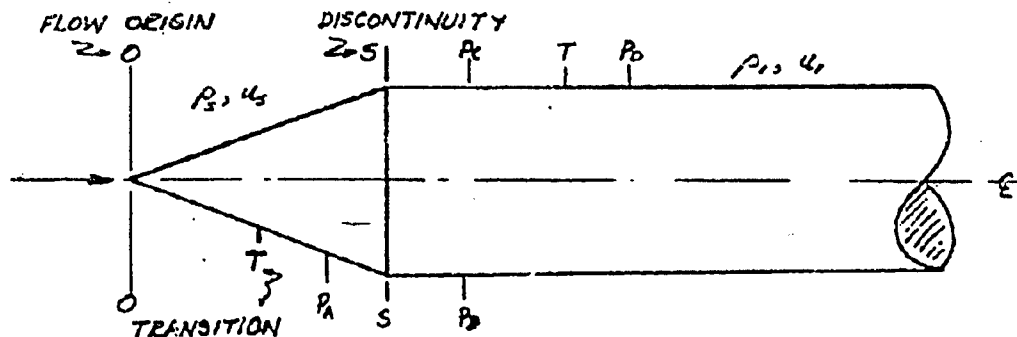
The constant momentum-loss method of section 3.2.2 can be applied to the case of three-dimensional flow. The only changes which need be made are in the equations for the momentum thickness where the factors 1/3 and 1/2 must be inserted in the laminar and turbulent  $\delta$ , respectively. Making this change we have

$$\text{LAMINAR } \theta_{\text{cone}} = 0.002174 \left( \frac{p_s u_s}{(T_s)^{1.7}} \right)^{-0.5} \left( \frac{T_s}{T'} \right)^{0.15} L^{0.5} \quad (35)$$

$$\text{TURBULENT } \theta_{\text{cone}} = 0.003959 \left( \frac{p_s u_s}{(T_s)^{1.7}} \right)^{-0.2} \left( \frac{T_s}{T'} \right)^{0.66} L^{0.8} \quad (36)$$

These equations are used in Eq. (17) to find  $L'$ , the effective length, which is then used in Eq. (2) - (15) to obtain friction and heat transfer.

The types of discontinuities usually encountered in conical flow would be found, say, on a cone-cylinder. The equations for various starting lengths found from Eq. (17), (35) and (36) are summarized for that body in the following table:



ASSUMED PRESSURE GRADIENT ON THE CYLINDER

$$\text{AT } P_A: \quad l'_A = \frac{TP_A}{2} + 0.4727(\overline{OT})^{0.625} \left[ \frac{P_2 u_2}{(T_2')^{1.7}} \right]^{-0.375} \quad (37)$$

$$\text{AT } P_B: \quad l'_B = \overline{SP}_B + 0.57435 \left( l'_A + \frac{\overline{SP}_A}{2} \right) \frac{P_2}{P_1} \left( \frac{u_2}{u_1} \right)^{1.5} \left( \frac{T_1'}{T_2'} \right)^{0.825} \quad (38)$$

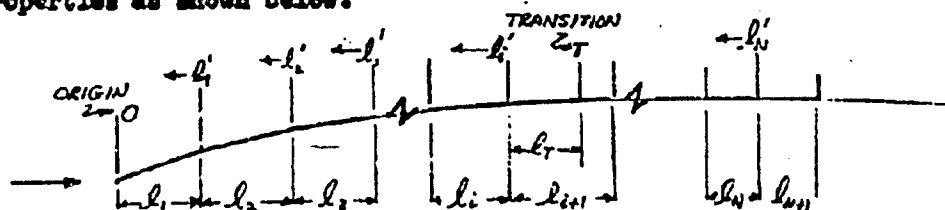
$$\text{AT } P_C: \quad l'_C = \overline{SP}_C + 0.57735 \frac{\overline{OS}}{3} \frac{P_2}{P_1} \left( \frac{u_2}{u_1} \right)^3 \left( \frac{T_1'}{T_2'} \right)^{0.3} \quad (39)$$

$$\text{AT } P_D: \quad l'_D = \overline{TP}_D + 0.4696 \left[ l'_C + \overline{TP}_C \right]^{0.625} \left[ \frac{P_2 u_2}{(T_2')^{1.7}} \right]^{-0.375} \quad (40)$$

to be used in Eq. (2) - (15) to find the friction and heat transfer.

3.2.5 Flow With Streamwise Gradients

In section 3.2.4 the flow on the cylinder portion of the body investigated was assumed to be at constant pressure in order to obtain the simple equations (38), (39) and (40). For the case where a streamwise gradient is to be considered the body must be broken into small segments of constant flow properties. If this is done then the equations (17), (21) and (22) can be applied to each section and equivalent starting lengths can be found for any point on the body. For flow of such a type consider a semi-infinite biconvex airfoil broken into small segments of constant stream properties as shown below:



Each segment has the stream properties  $p_1, u_1; p_2, u_2; p_3, u_3; \dots$  etc.

To derive the general equation for the effective length along the body at a point preceded by completely laminar flow, apply Eq. (21) and (22) to Eq. (17). For segment  $L_1$ ,

$$Q_1 g_1 = Q_2 g_2$$

So

$$0.003765 \left[ \frac{p_1 u_1}{(T_1)^{1.7}} \right]^{0.5} \left( \frac{T_1}{T_1'} \right)^{0.15} L_1^{0.5} g_1 = 0.003765 \left[ \frac{p_2 u_2}{(T_2)^{1.7}} \right]^{0.5} \left( \frac{T_2}{T_2'} \right)^{0.15} L_1'^{0.5} g_2$$

or

$$(L_1')^{0.5} = L_1^{0.5} \left( \frac{p_2 u_2}{p_1 u_1} \right)^{0.5} \left( \frac{T_1}{T_2} \right)^{0.15} \left( \frac{T_1'}{T_2'} \right)^{0.15} \frac{g_1}{g_2}$$

$$L_1' = L_1 \left( \frac{p_2 u_2}{p_1 u_1} \right) \left( \frac{T_1}{T_2} \right)^2 \left( \frac{T_1'}{T_2'} \right)^{0.3} \left( \frac{g_1}{g_2} \right)^2$$

Apply the perfect gas law to the  $g$  ratio

$$\frac{g_1}{g_2} = \frac{\rho_1 u_1^2}{\rho_2 u_2^2} = \frac{p_1 u_1^3 T_2}{p_2 u_2^3 T_1}$$

and insert in the above equation to get

$$L_1' = L_1 \left( \frac{p_2 u_2}{p_1 u_1} \right) \left( \frac{p_2 u_2}{p_1 u_1} \right) \left( \frac{T_2}{T_1} \right)^{0.3} \left( \frac{T_1}{T_2} \right)^2 \left( \frac{T_1'}{T_2'} \right)^{0.3}$$

$$\text{or } L_1' = L_1 \frac{p_2}{p_1} \left( \frac{u_1}{u_2} \right)^3 \left( \frac{T_1'}{T_1} \right)^{0.3}$$

This procedure can be repeated for each segment, i.e.,

$$l'_i = (l_2 + l'_1) \frac{p_2}{p_1} \left( \frac{u_2}{u_1} \right)^3 \left( \frac{T_2'}{T_1'} \right)^{0.3}$$

or

$$l'_i = l_2 \frac{p_2}{p_1} \left( \frac{u_2}{u_1} \right)^3 \left( \frac{T_2'}{T_1'} \right)^{0.3} + l_1 \frac{p_1}{p_2} \left( \frac{u_1}{u_2} \right)^3 \left( \frac{T_1'}{T_2'} \right)^{0.3}$$

and in general

$$l'_i = \sum_{k=1}^{k=i} l_k \frac{p_k}{p_i} \left( \frac{u_k}{u_i} \right)^3 \left( \frac{T_k'}{T_i'} \right)^{0.3} \quad (41)$$

Let transition occur at some point T in segment  $l_{i+1}$  at some distance  $l_T$  from the beginning of the segment. Then the effective length to T is

$$l'_T = (l_T + l'_i)^{0.625} \cdot 0.4696 \left[ \frac{p_{i+1} u_{i+1}}{(T'_{i+1})^{1.7}} \right]^{-0.375} \quad (42)$$

and for any general turbulent section thereafter

$$l'_N = \sum_{k=i+2}^{k=N} l_k \frac{p_k}{p_N} \left( \frac{u_k}{u_N} \right)^{1.5} \left( \frac{T'_k}{T'_N} \right)^{0.825} + (l'_T + l_{i+1} - l_T) \frac{p_{i+1}}{p_N} \left( \frac{u_{i+1}}{u_N} \right)^{1.5} \left( \frac{T'_{i+1}}{T'_N} \right)^{0.825} \quad (43)$$

These generalized formulas for  $l'$  may then be used along with local conditions to evaluate the friction and heat transfer at any point from Eq. (2) - (15).

#### 4.0 Conclusions

A collection has been made of aerodynamic flow characteristics for one, two and three dimensional hypersonic flow. The Mach number range is from  $0 \leq M \leq 20$ . The flow parameters are shown as functions of initial Mach number and deflection angle in sections 1, 2, 3; Figures 1.0 through 3.16.

Equations for compressible skin friction and heat transfer using the reference temperature method were presented for two and three dimensional flow, flow with discontinuities and flow with streamwise gradient. Figures necessary in using the formulae are in section 4, Figure 4.1 - 4.7.

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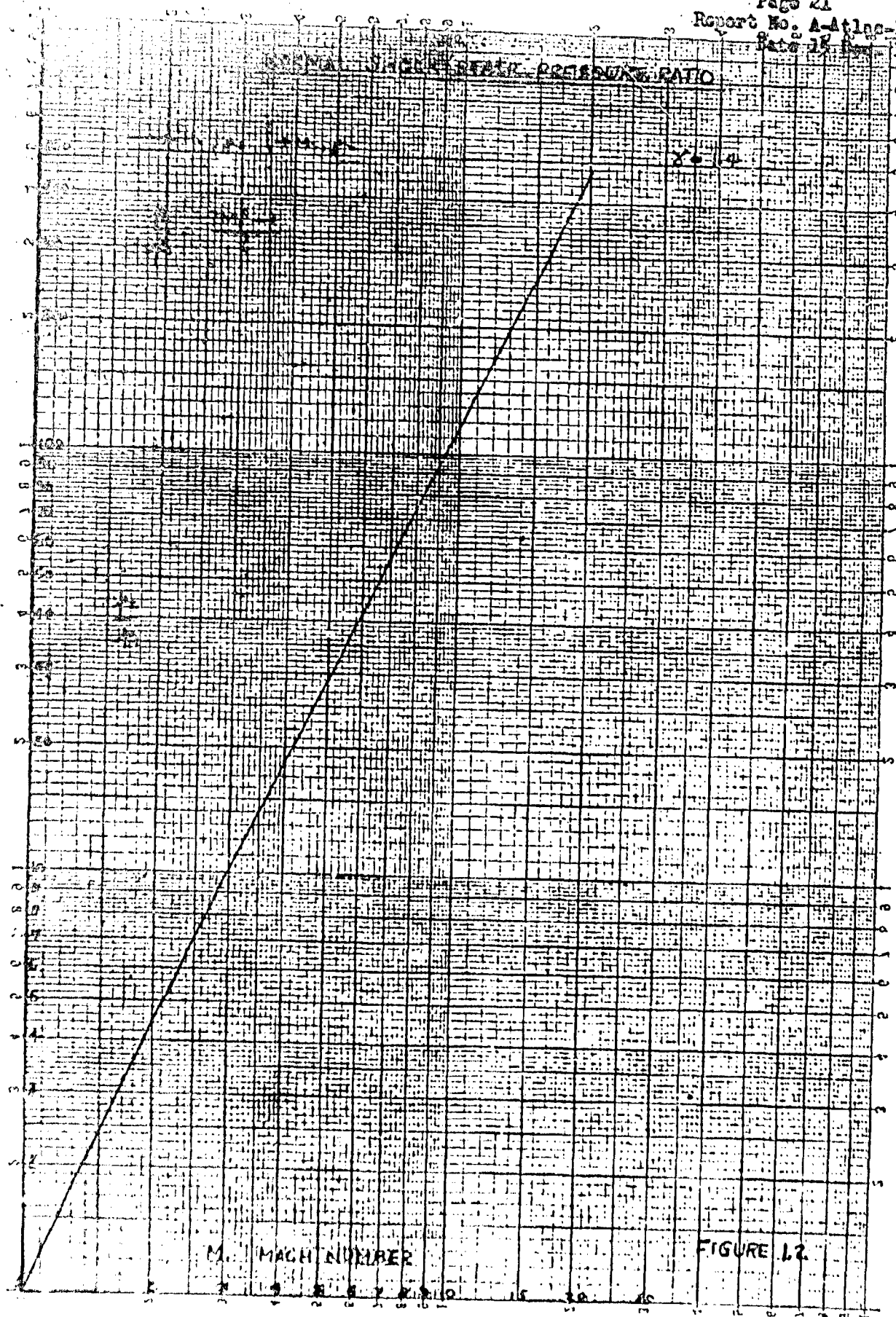
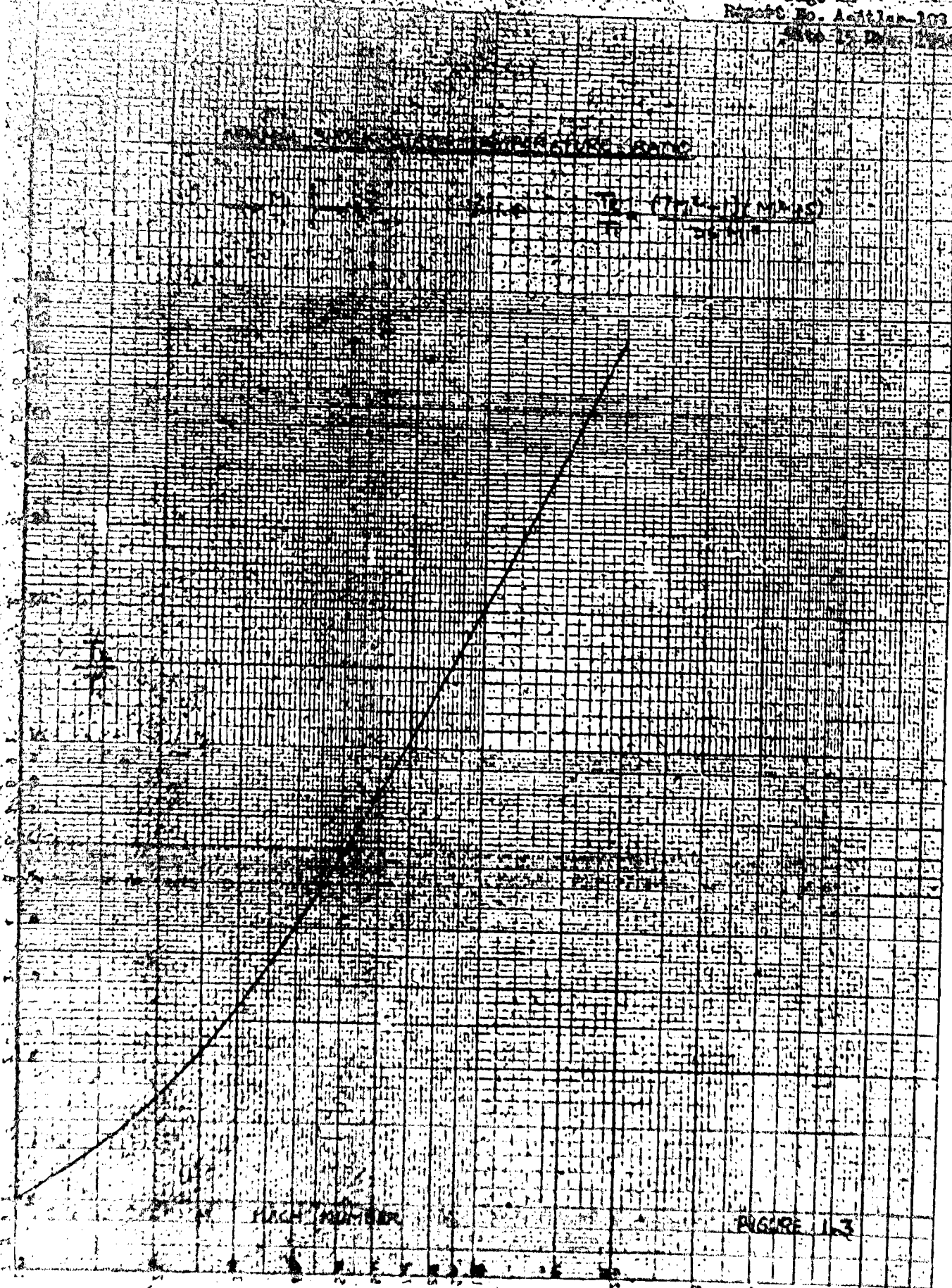


FIGURE 12







HIGH FREQUENCY

FIGURE 1-3

SUPPLEMENT  
A-ATLAS-103

# NORMAL SHOCK DENSITY RATIO FOR AIR AT DISSOCIATIONAL EQUILIBRIUM

DISSOCIATED AIR PROPERTIES FROM CONVARD REPORT ZA-7-01

$M_1$   
 $P_1$   
 $T_1$   
 $\rho_1$

$\rightarrow$

$P_2$   
 $\rho_2$

$T_1 = 400^\circ R$

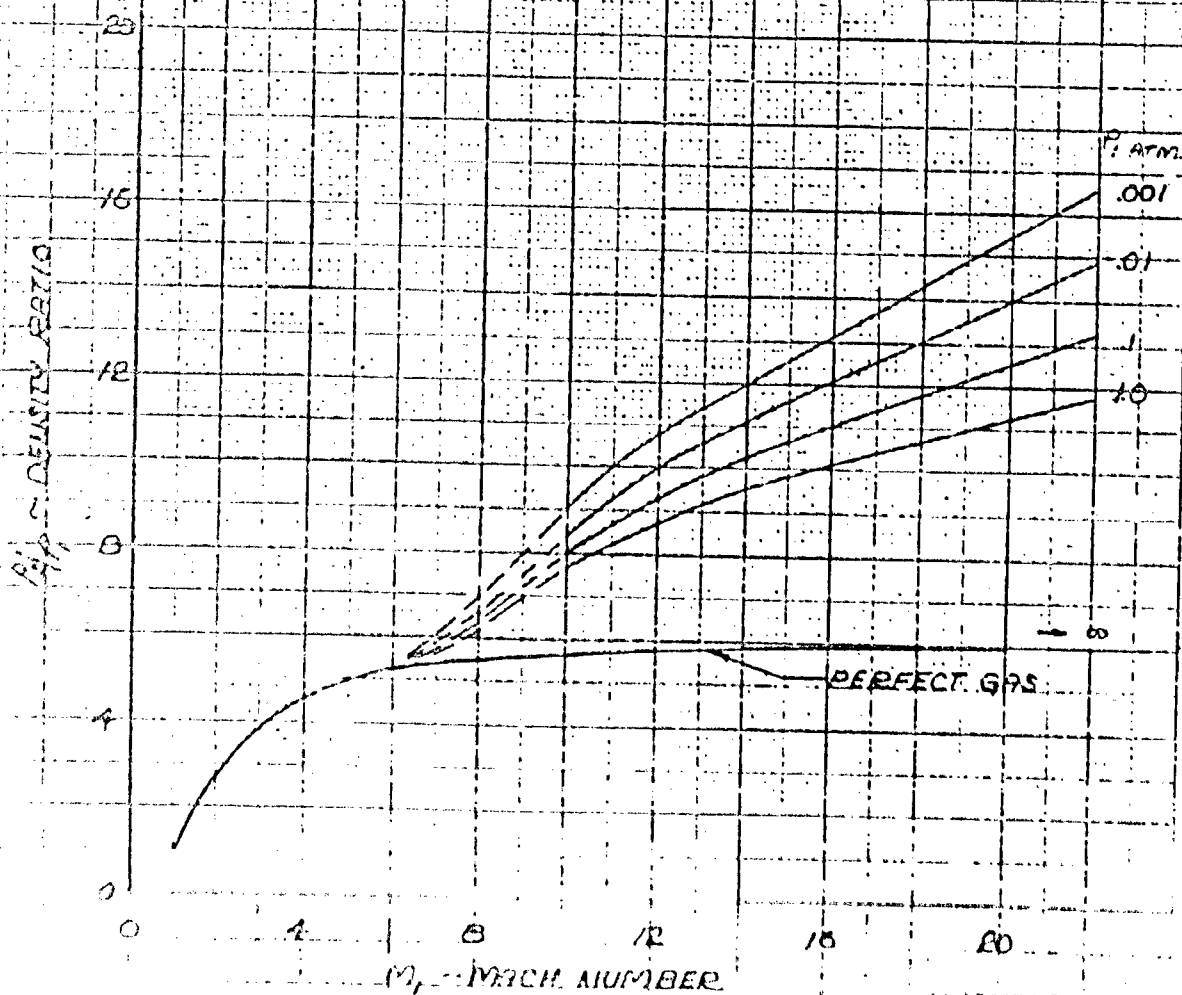
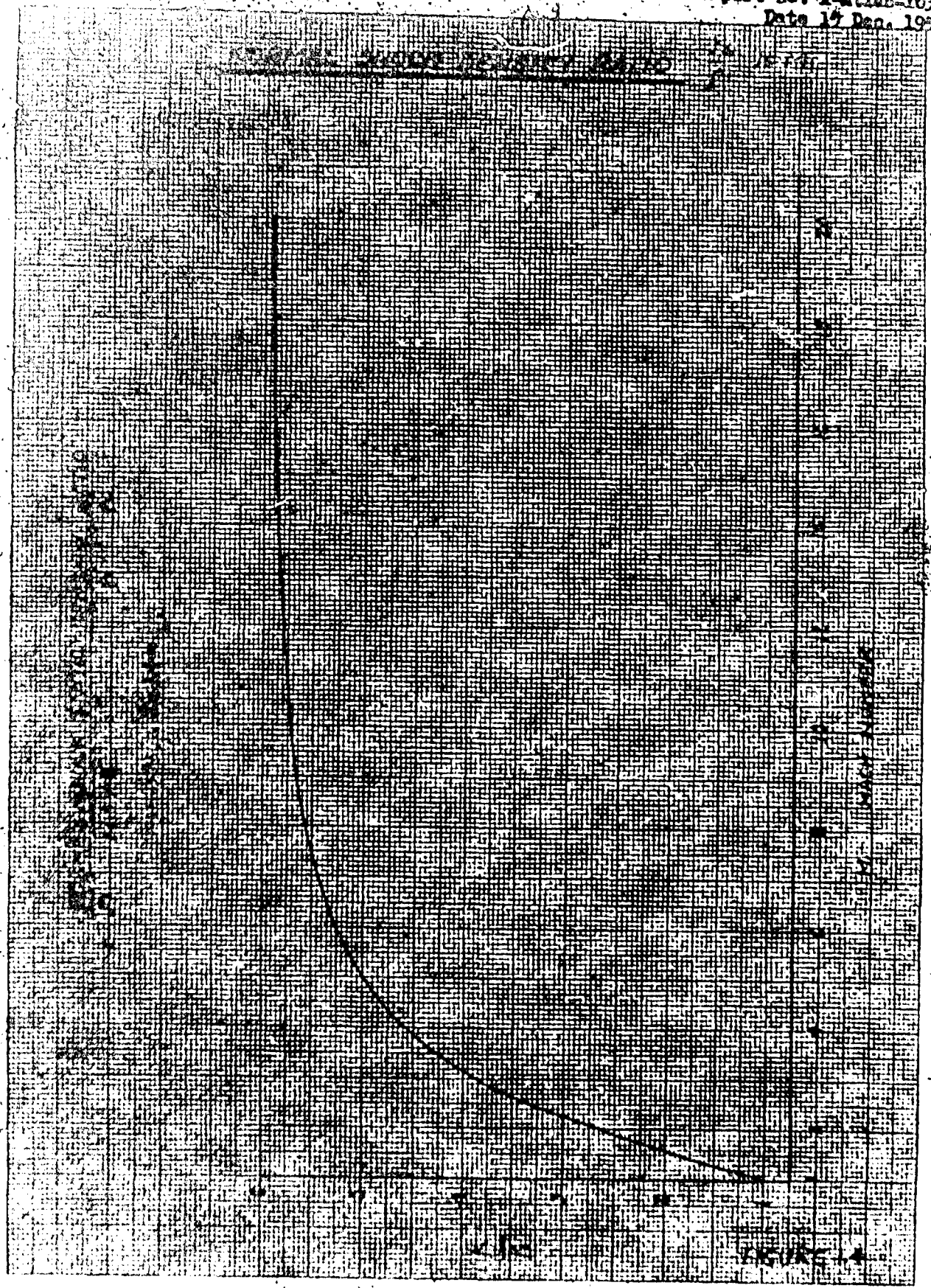
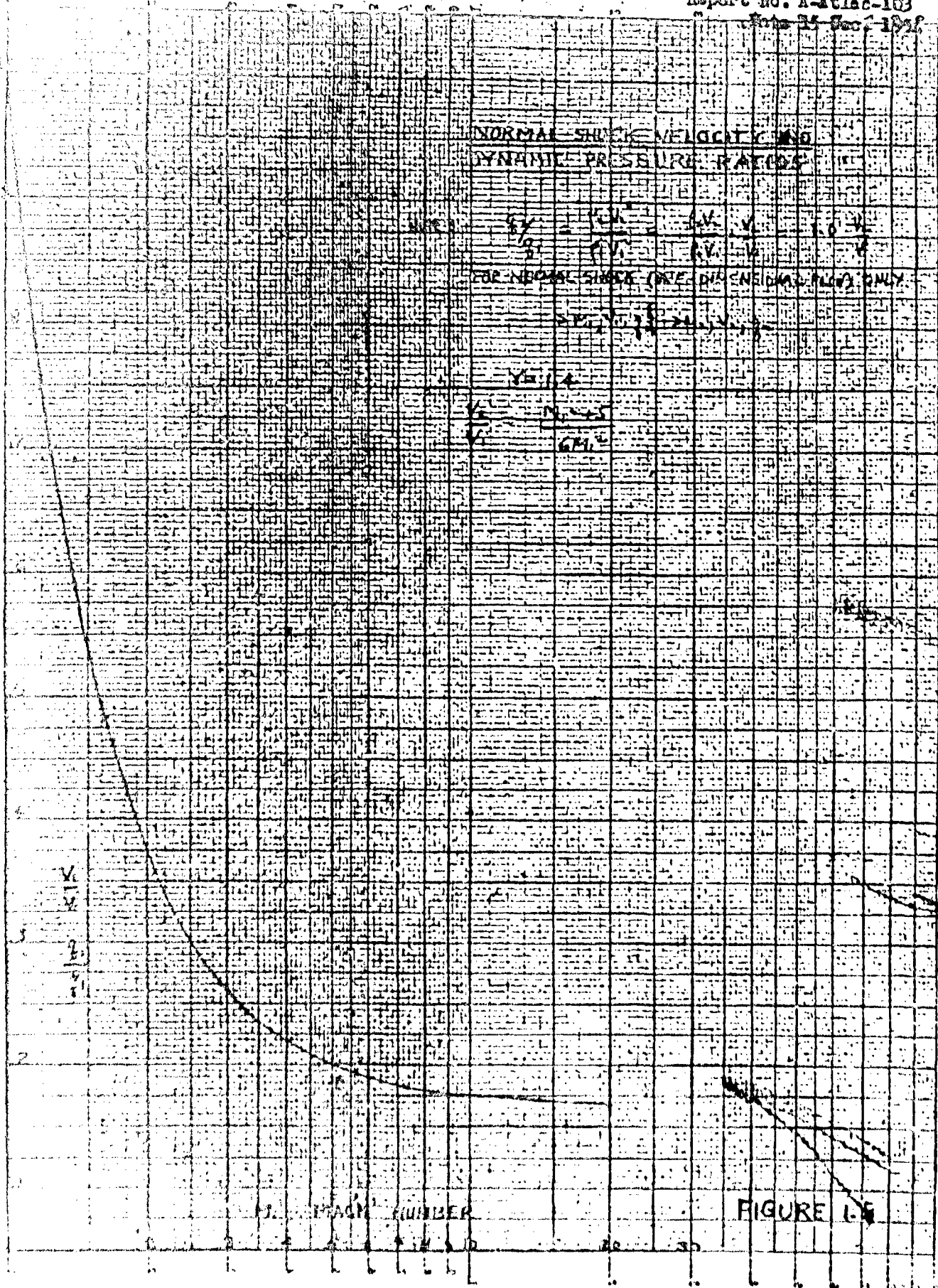
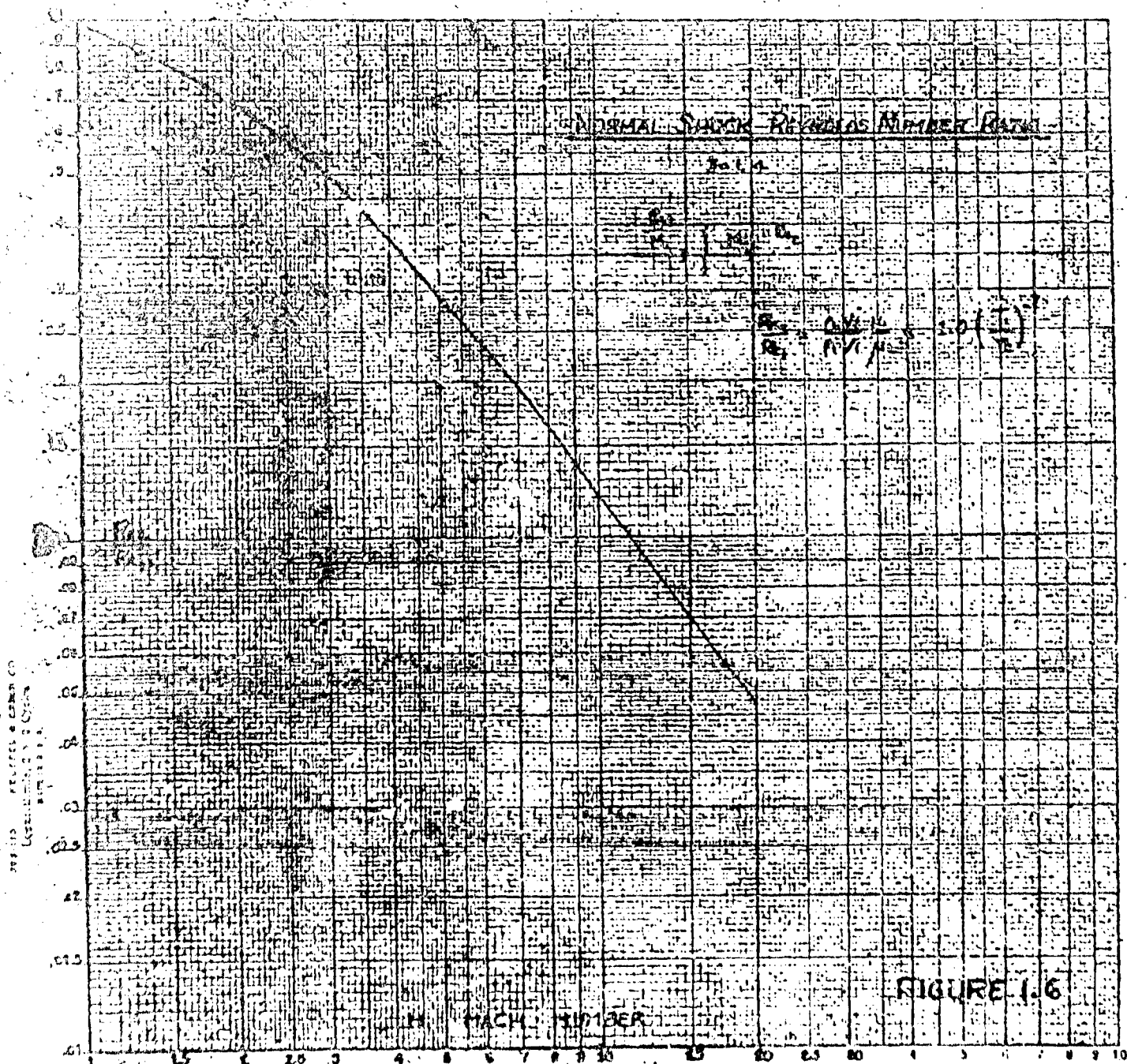


FIGURE 1.4A









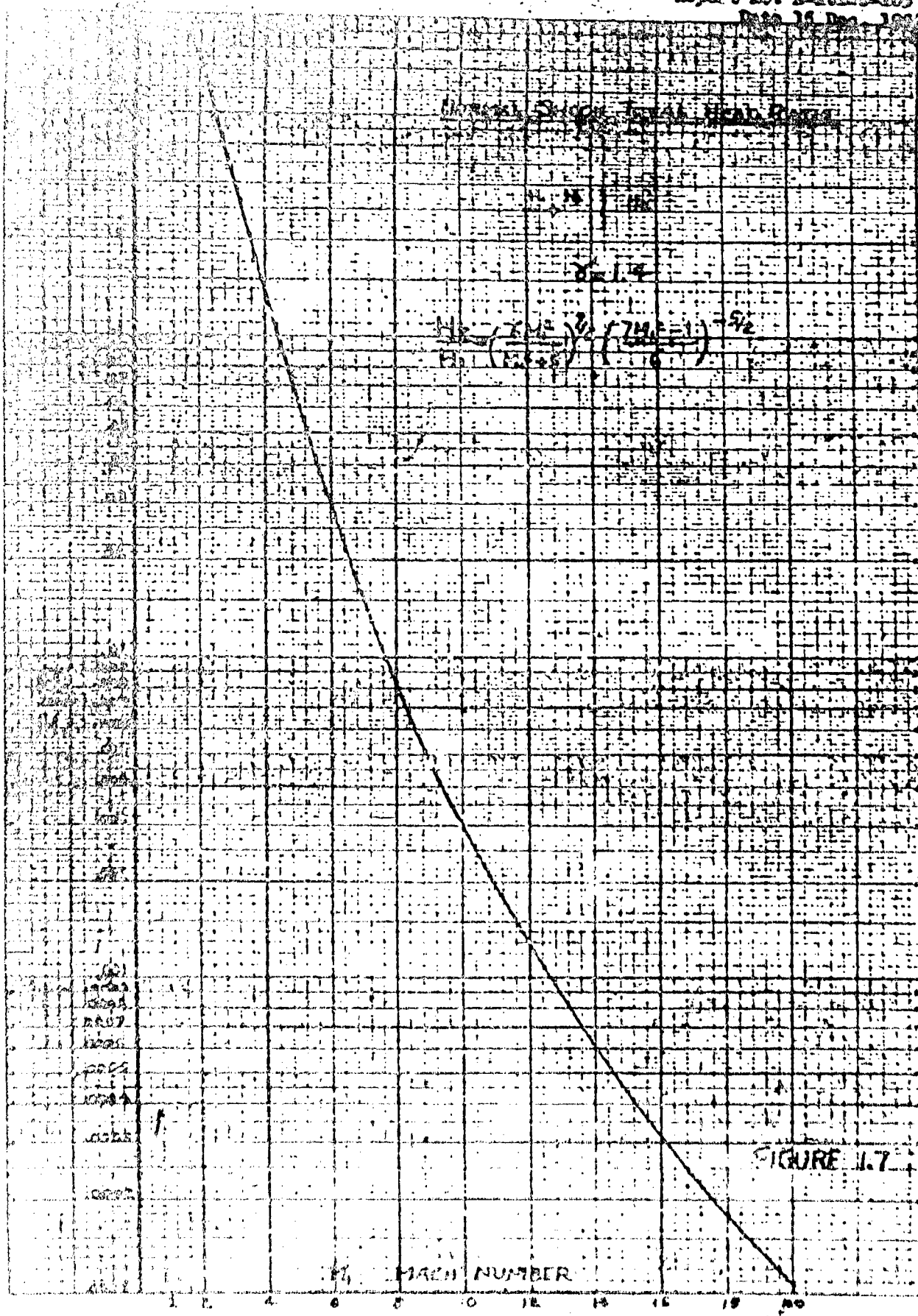


FIGURE 1.7

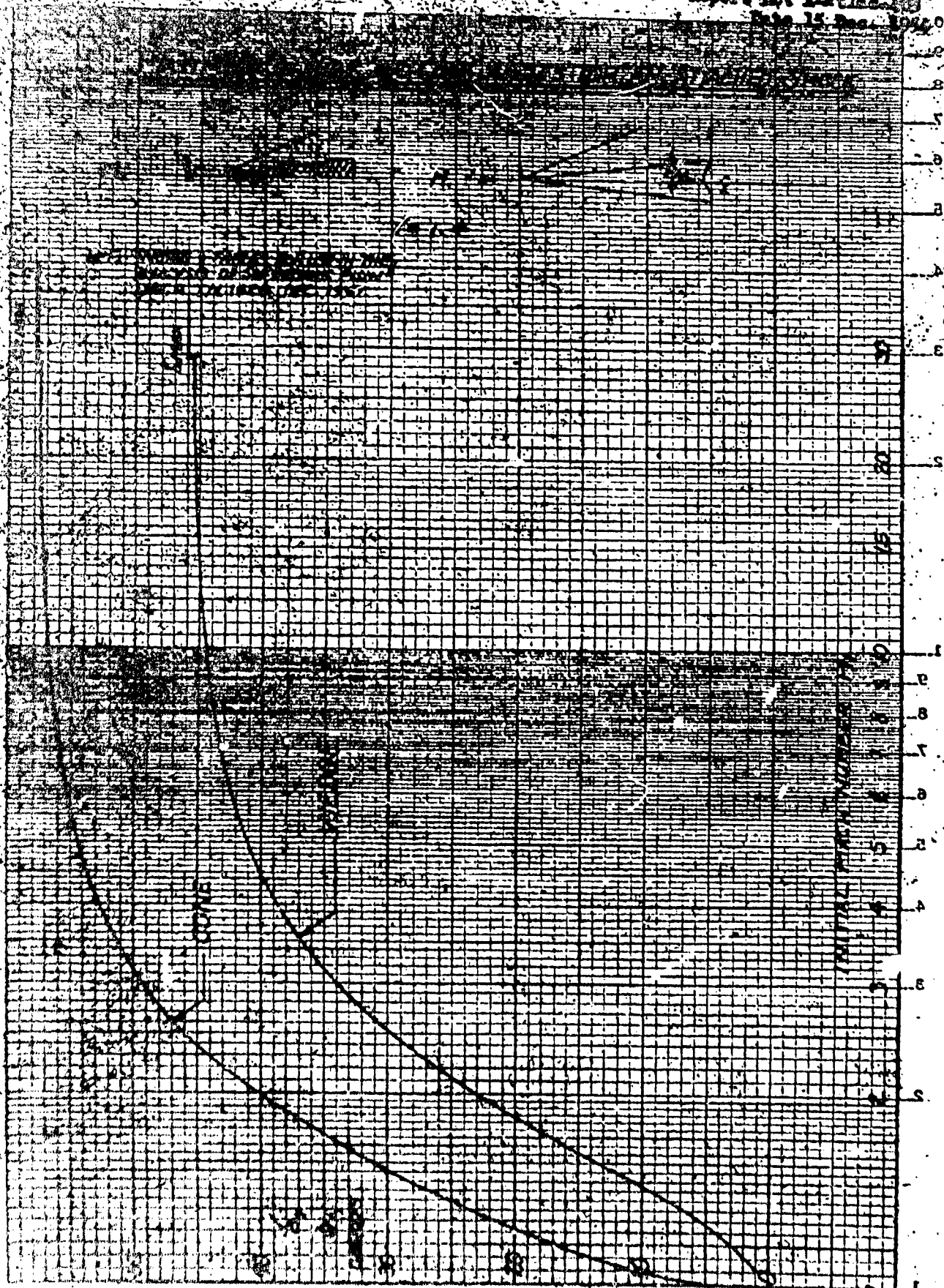
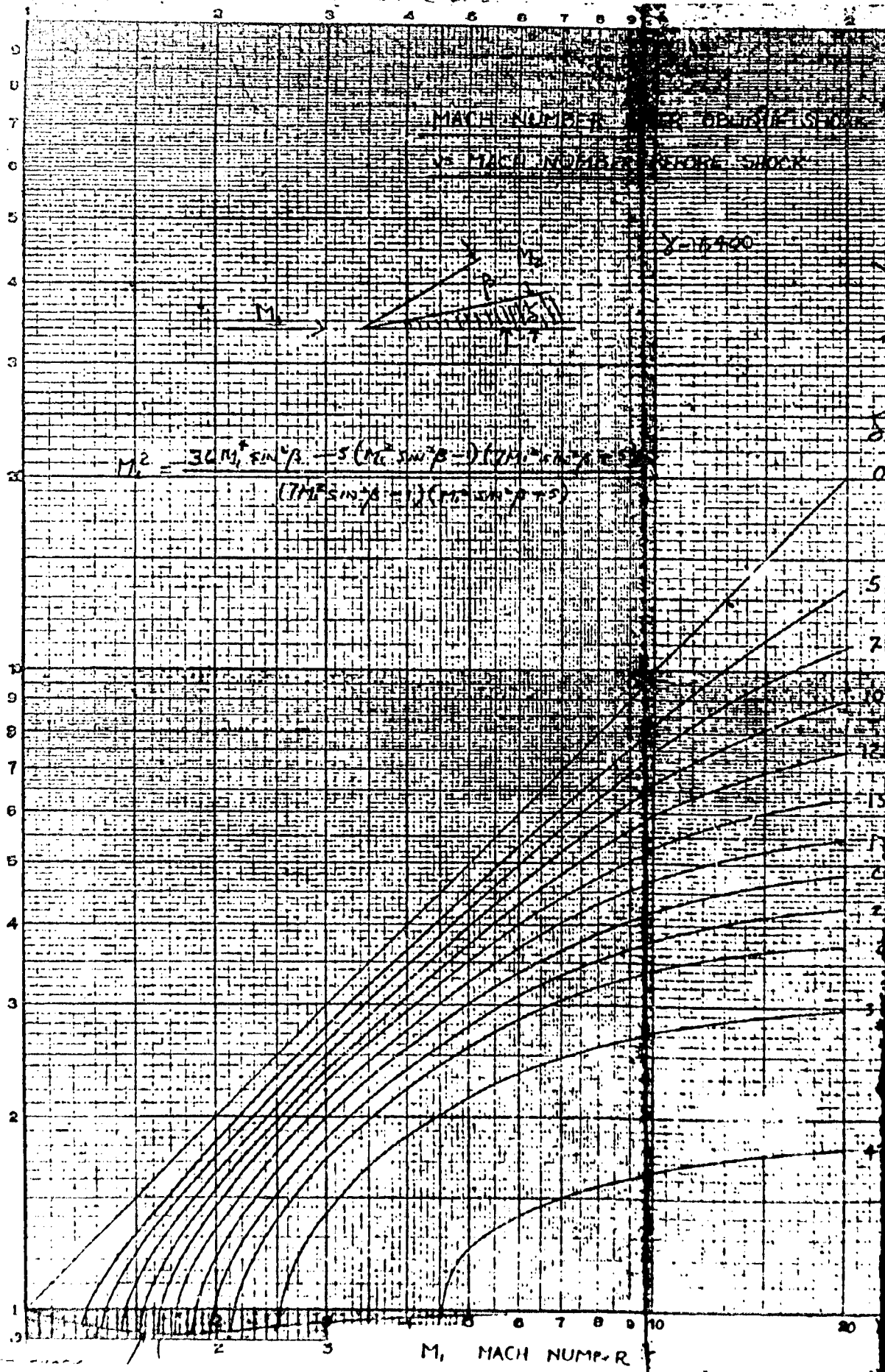


FIGURE 2.0





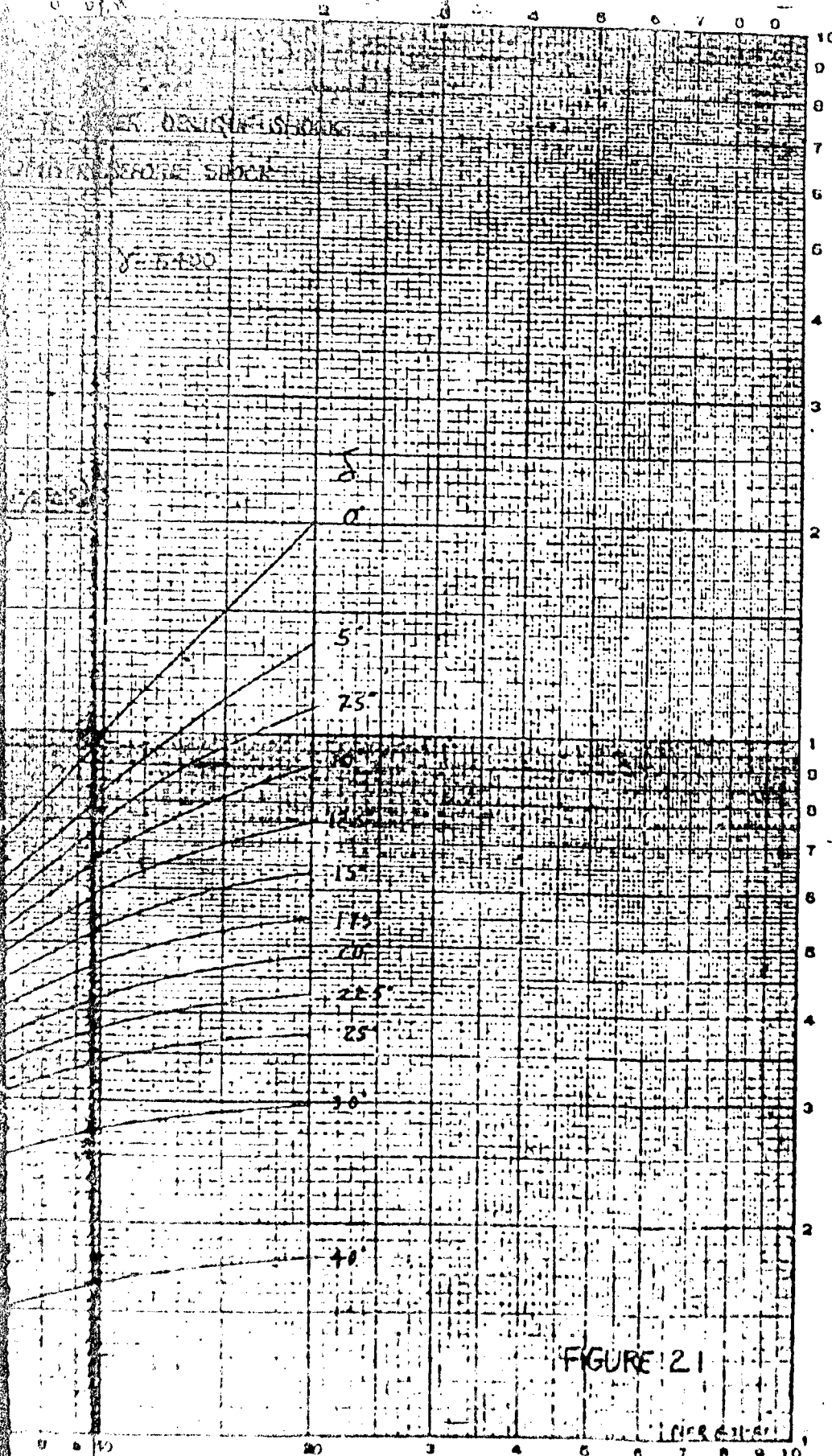
$M_z$  vs  $M_1$

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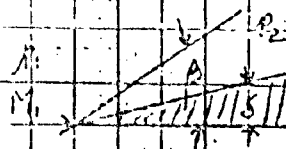
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FIGURE 2.1

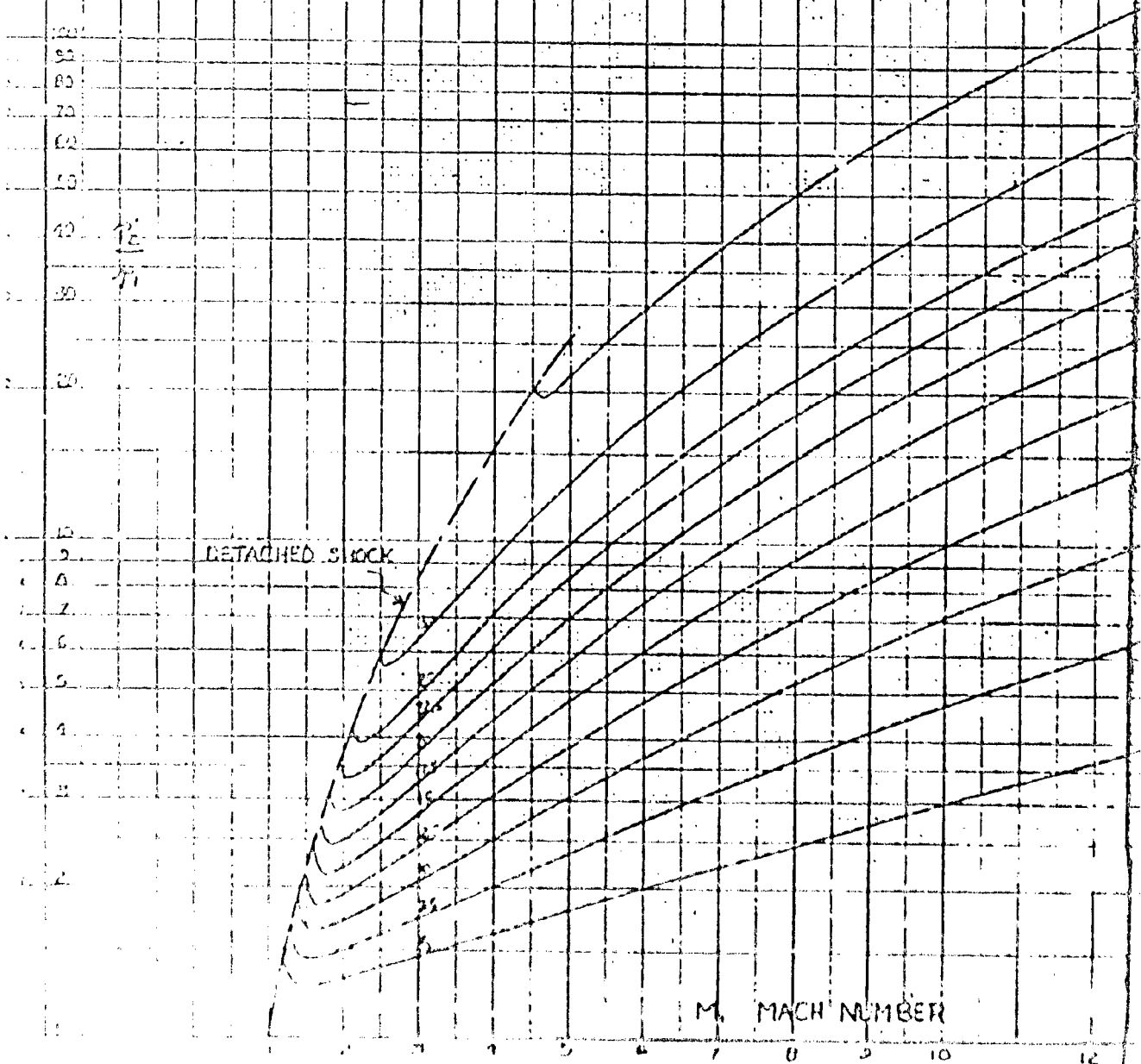


# OBlique SHOCK STATIC PRESSURE $P_2$ VS. MACH - NUMBER

$\gamma = 1.400$



$$\frac{P_2}{P_1} = \frac{7M_1^2 \sin^2 \theta - 1}{5}$$



A

$P_2/P_1$  vs  $M_1$

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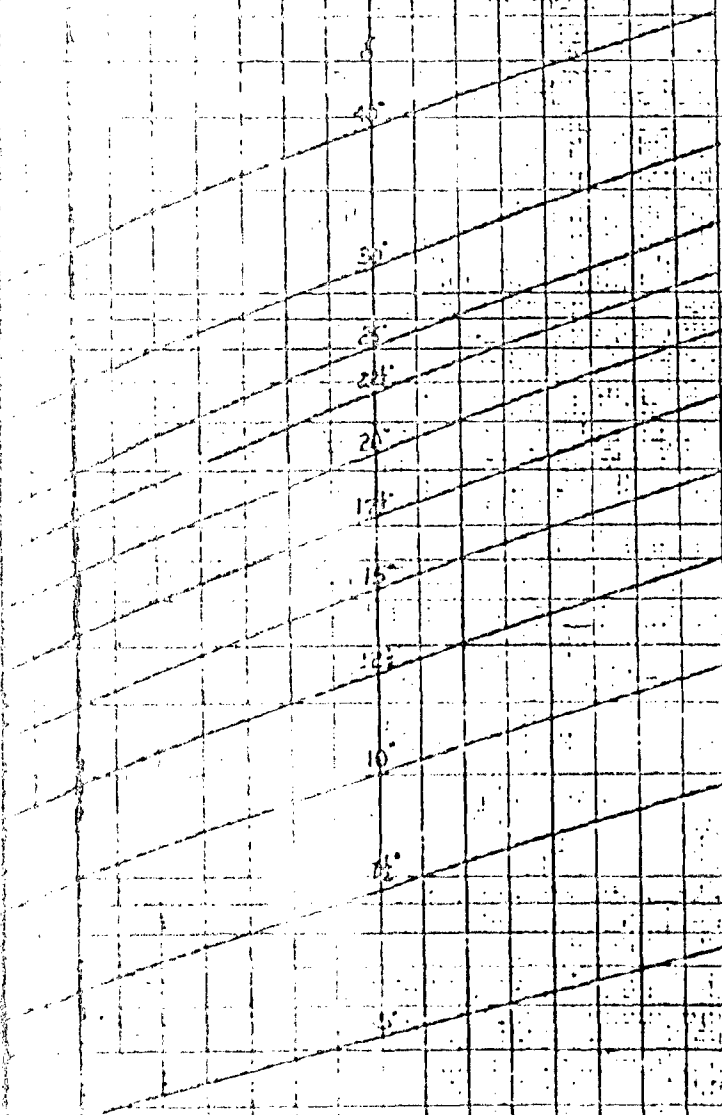
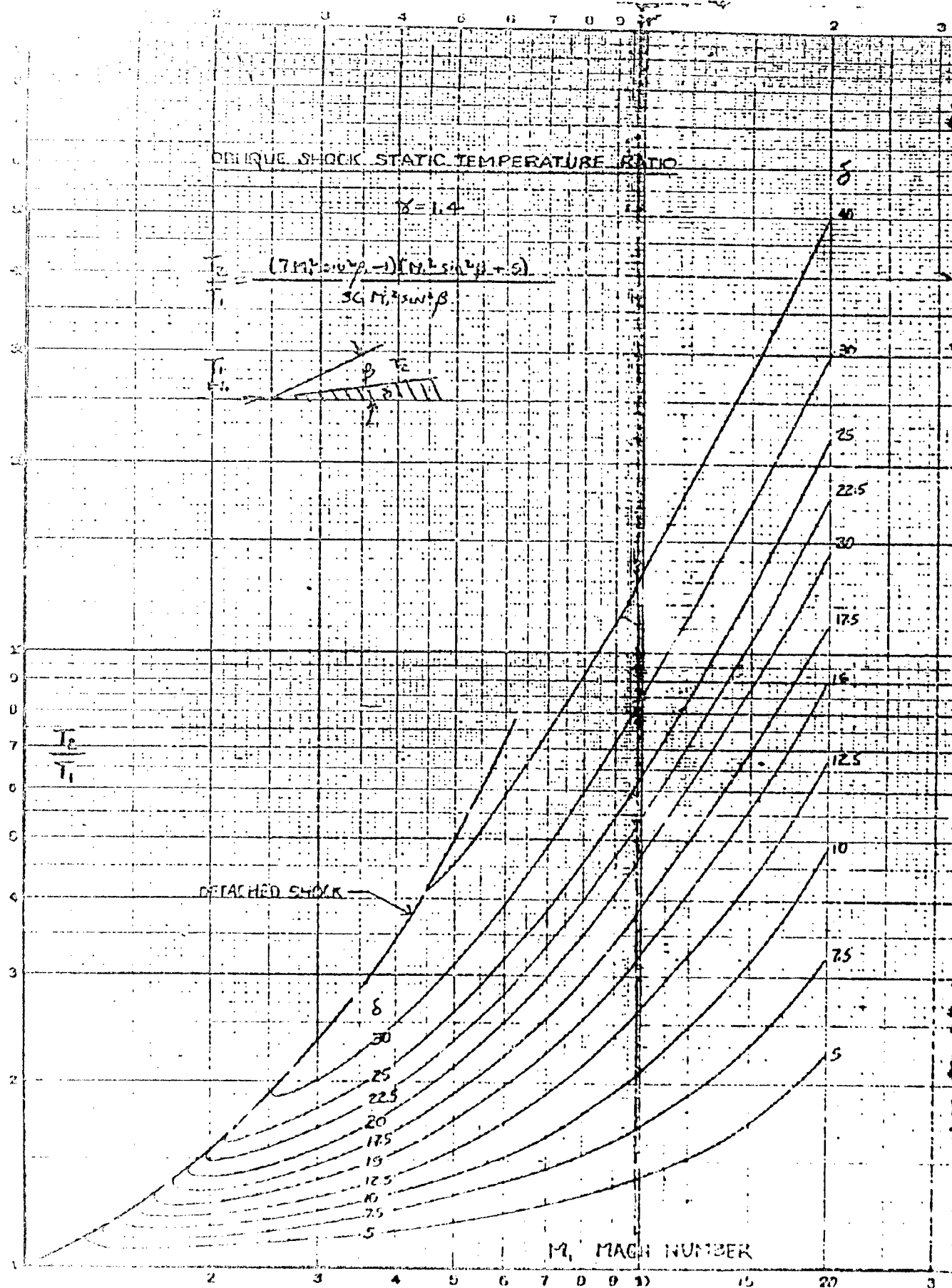


FIGURE 2.2

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B





$$\frac{T_c}{T_1} \text{ vs } M_1$$

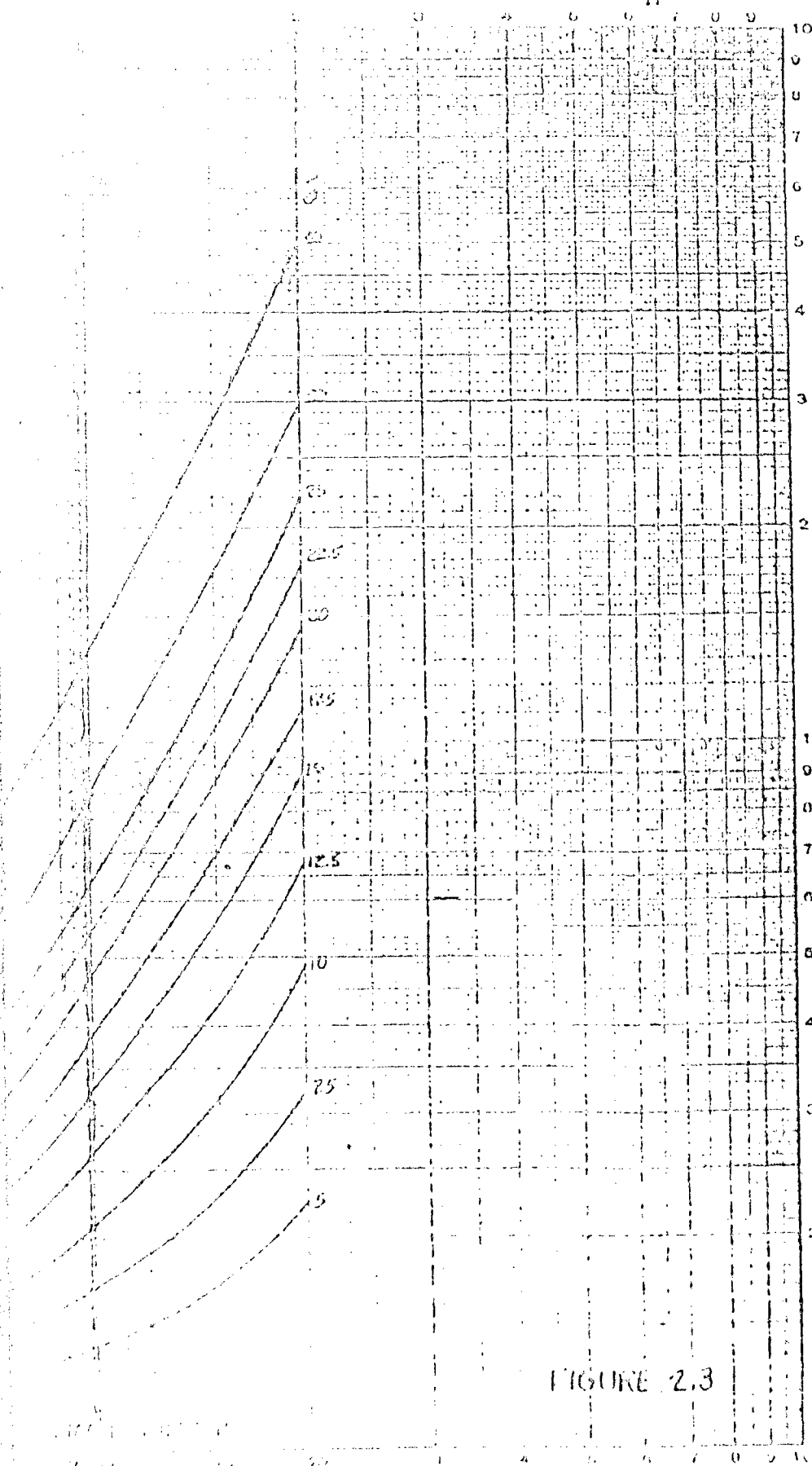


FIGURE 2.3

FIGURE 2.3

B



# OBLIQUE SHOCK WAVE ANGLE

vs. MACH NUMBER

$\gamma = 1.400$

$\beta$  DEGREES

90

80

70

60

50

40

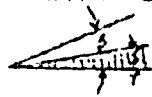
30

20

10

0

$M_1$



DETACHED SHOCK

$\delta$  DEGREES

40

30

25

22.5

20

17.5

15

12.5

10

7.5

5

0

$M_1$  MACH NUMBER

A

$\beta$  vs  $M_1$

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Date 15 Dec. 1954

ANGLE

DATA FROM NACA TN 1373

AND  $\tan \delta = \frac{M_1^2 \sin 2\beta - 2 \cot \beta}{2 + M_1^2 (\gamma + \cos 2\beta)}$

$\delta$  DEGREES

40

30

25

22.5

20

17.5

15

12.5

10

7.5

5

0

10

11

12

13

14

15

16

17

18

$\infty$

FIGURE 2.5

APR 5-7-57

B

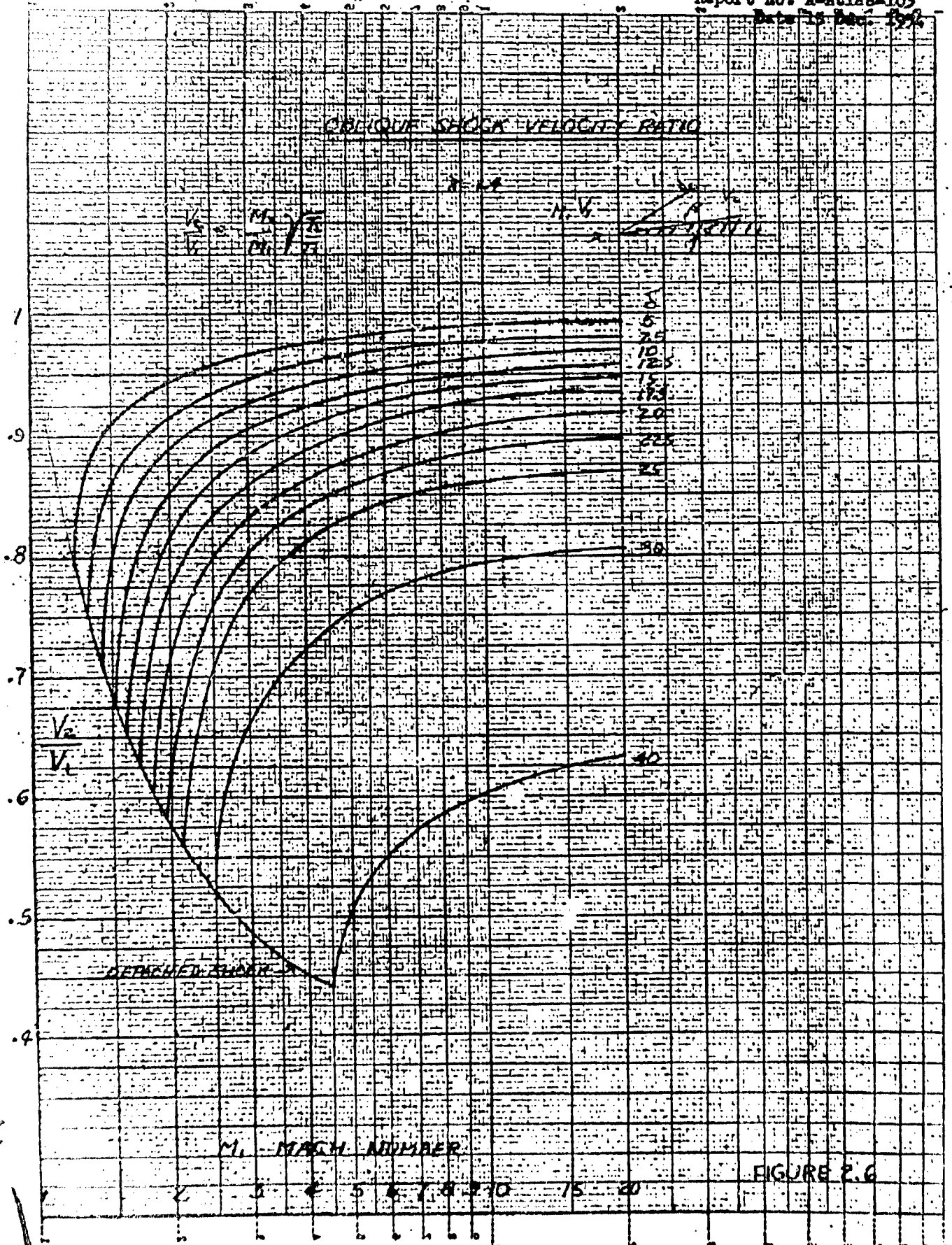
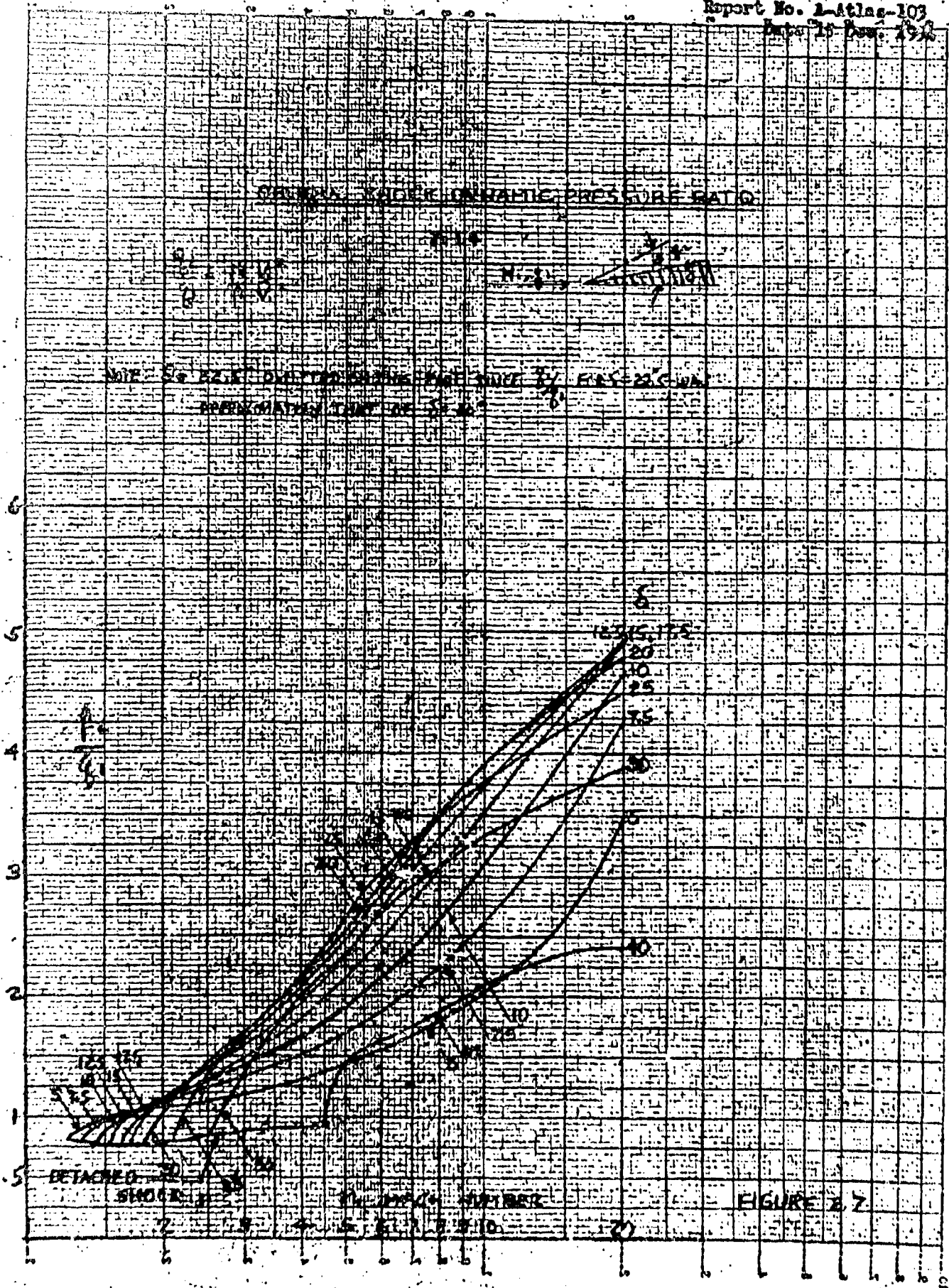


FIGURE 2.6



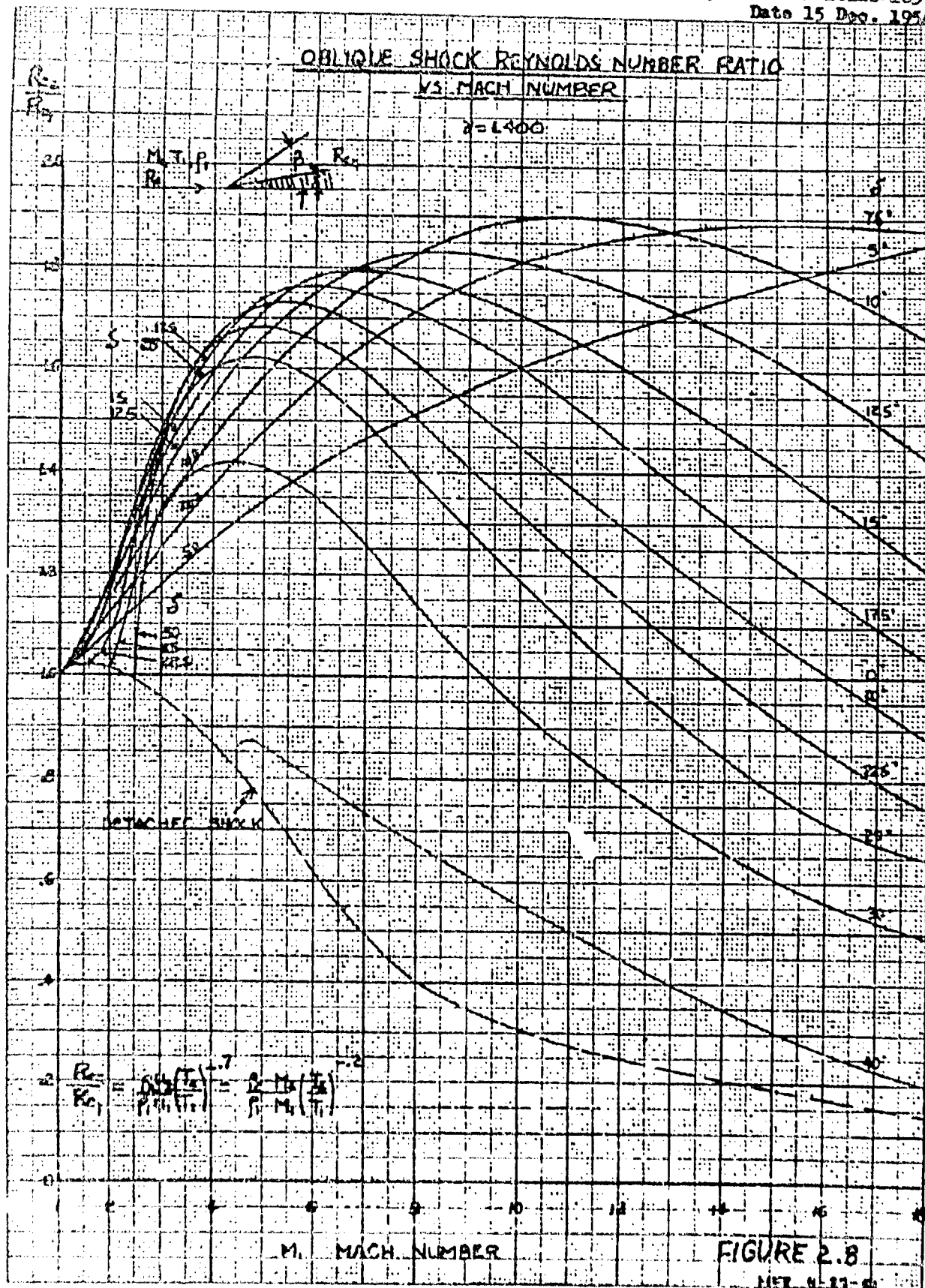
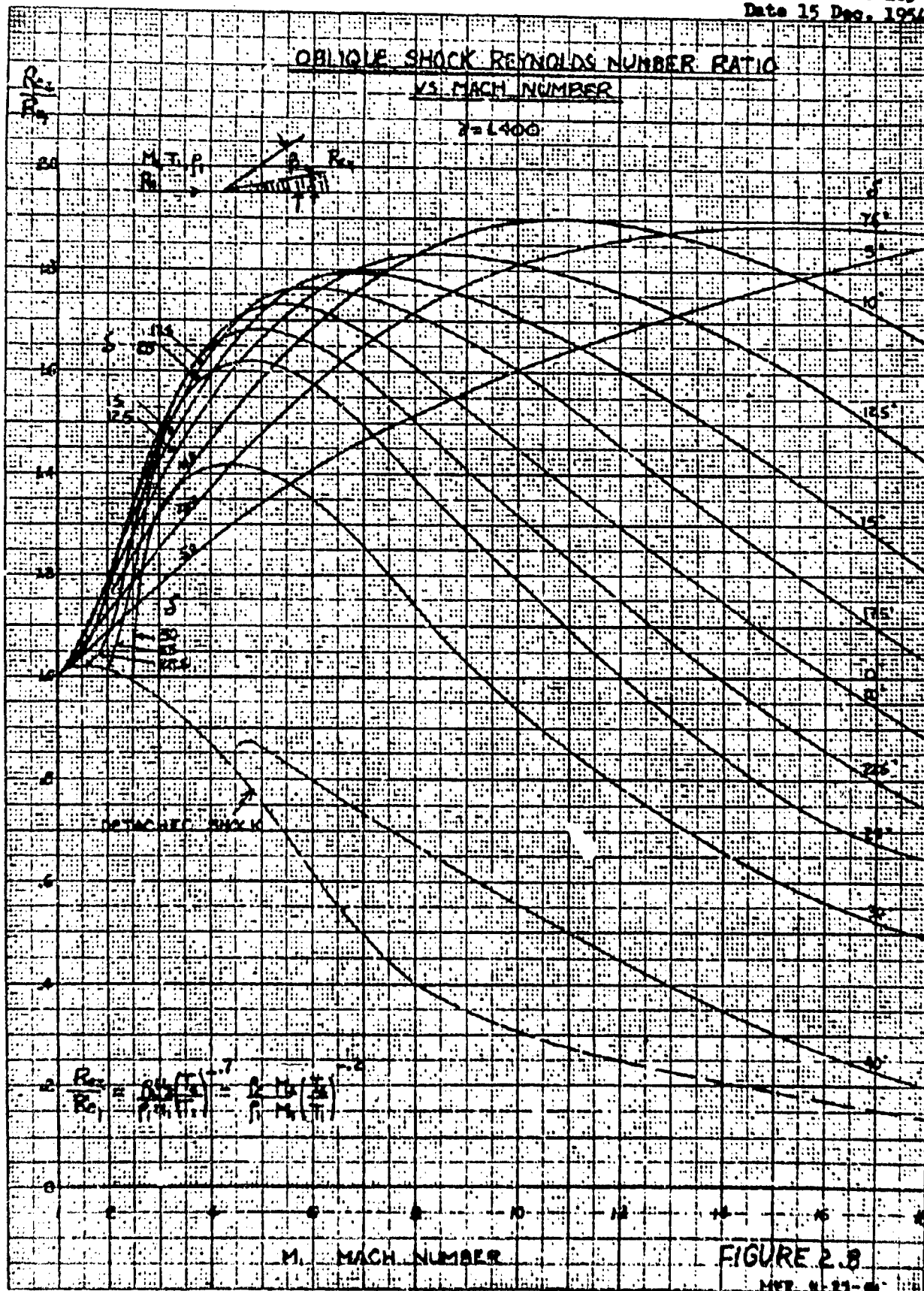


FIGURE 2.8

REF. 4-21-6





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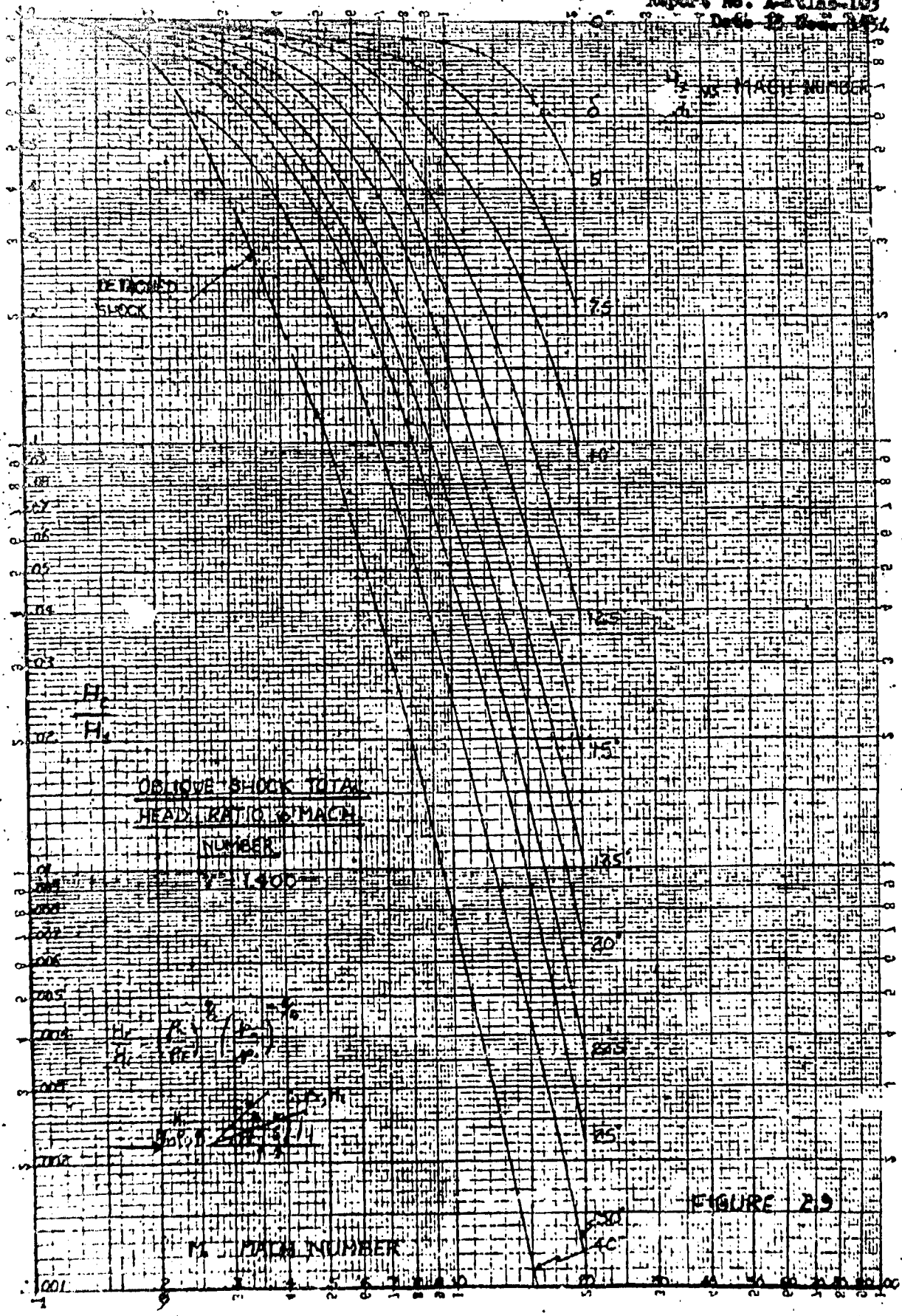
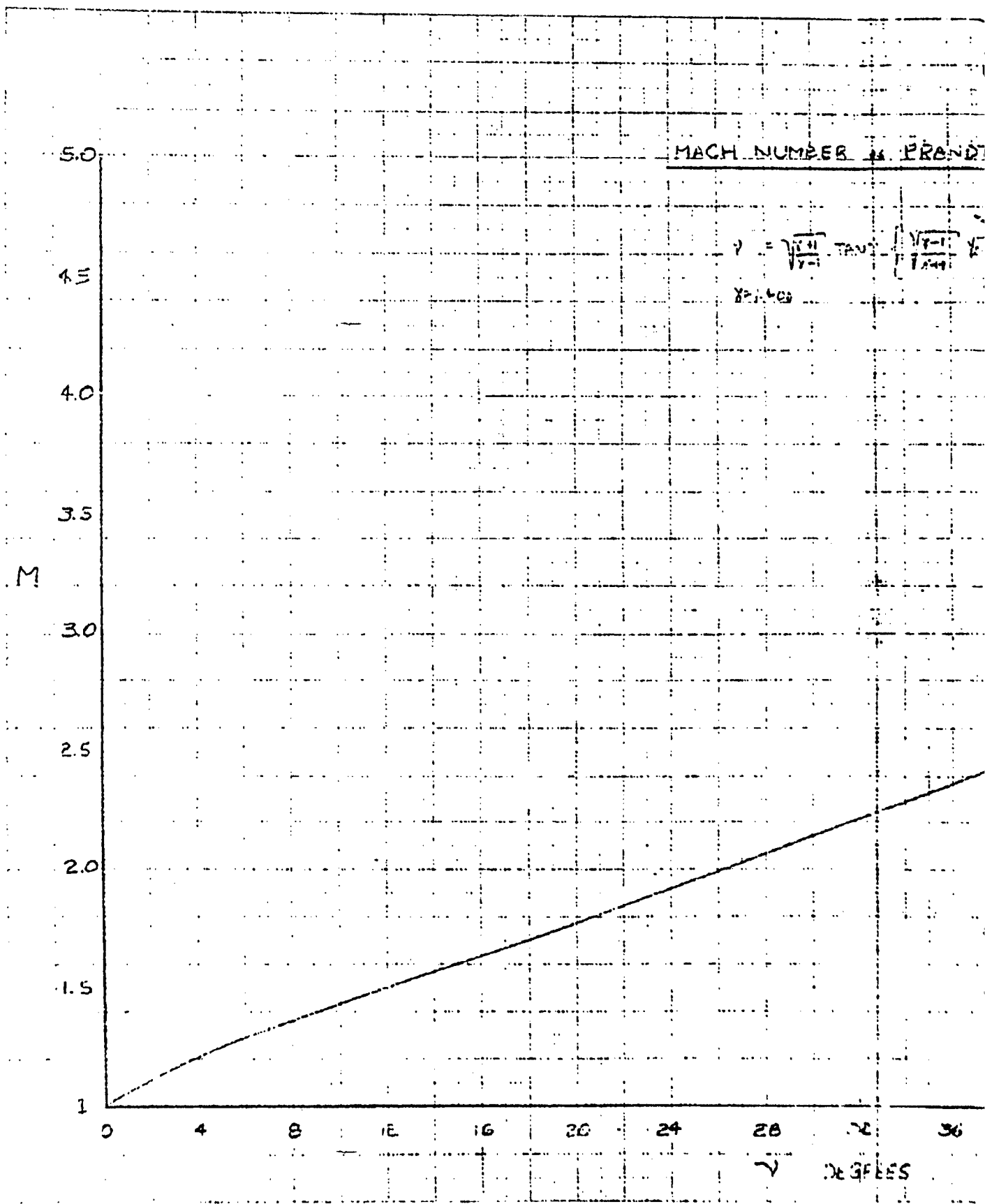


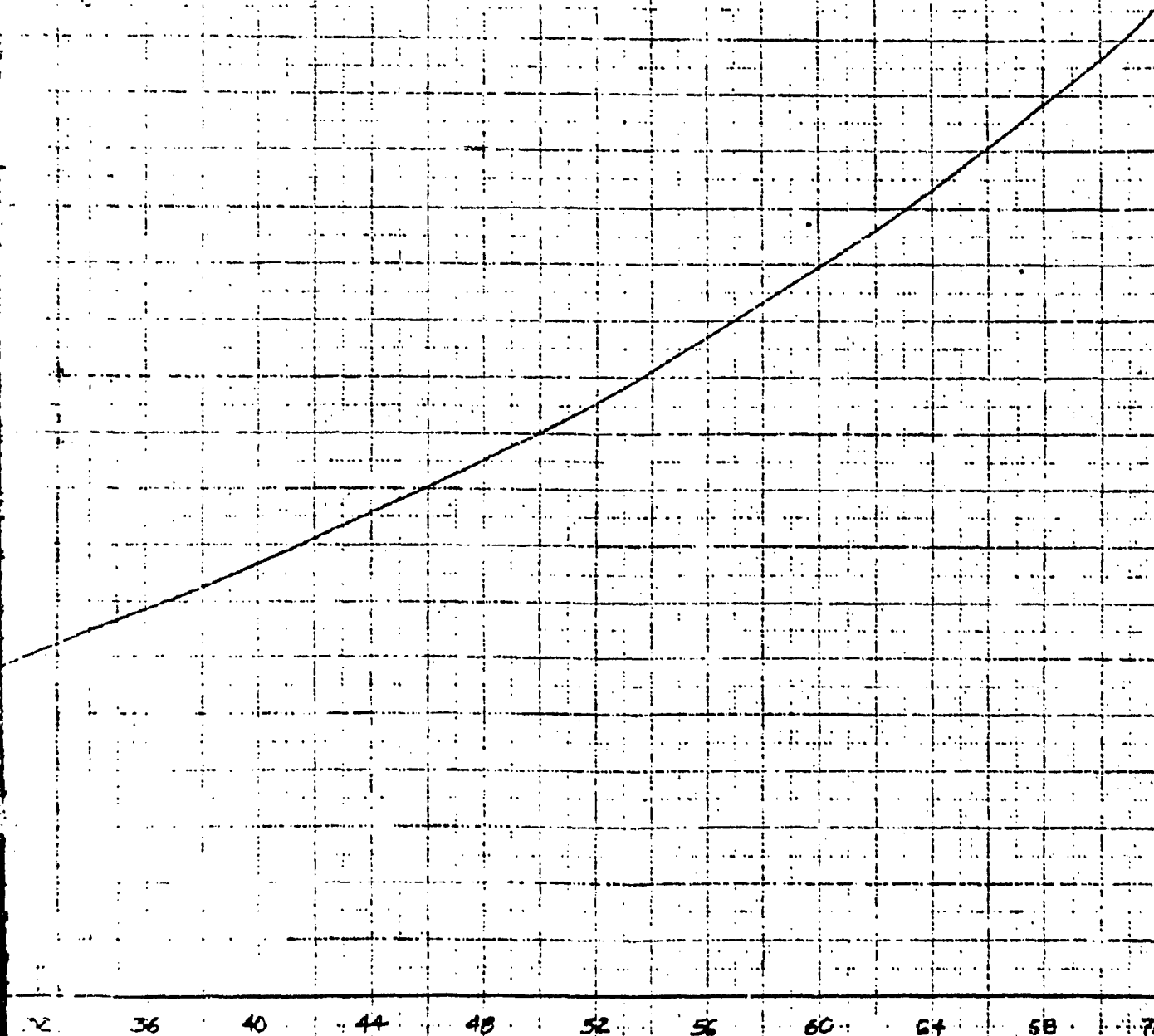
FIGURE 29



A

FB. 4 PRANDTL-MEYER ANGLE

$$\tan \left\{ \frac{\sqrt{M^2-1}}{1+\sqrt{M^2-1}} \right\} = \tan \frac{\theta}{2}$$



DEGREES

FIGURE 2.10p.

UNFA 2-15-54

B

PRANDTL-MEYER ANGLE

$$\left\{ \frac{V_2 - 1}{V_1 - 1} \right\}^{\frac{\gamma}{\gamma-1}} = \tan^2 \frac{\theta}{2}$$

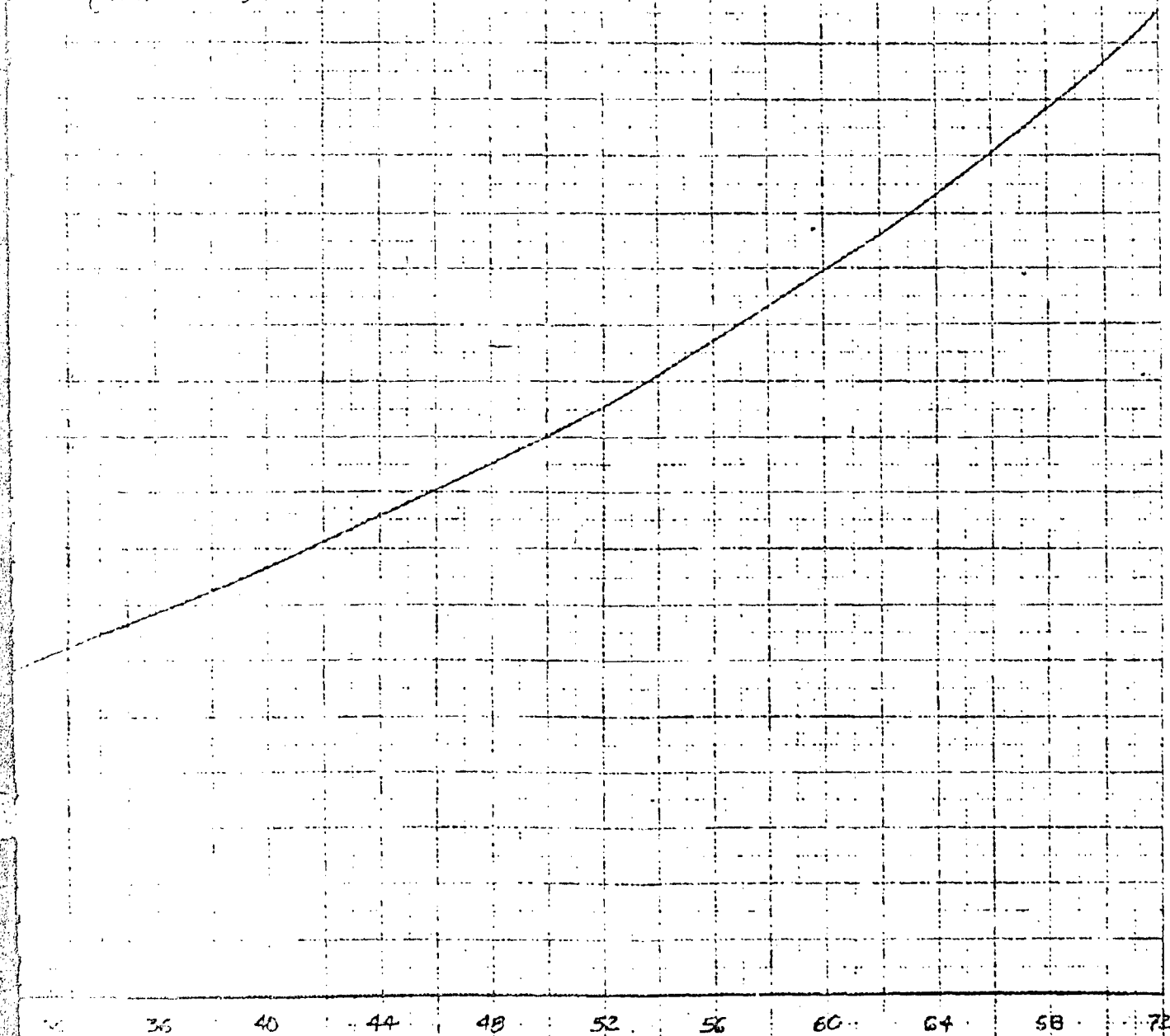


FIGURE 2.10a

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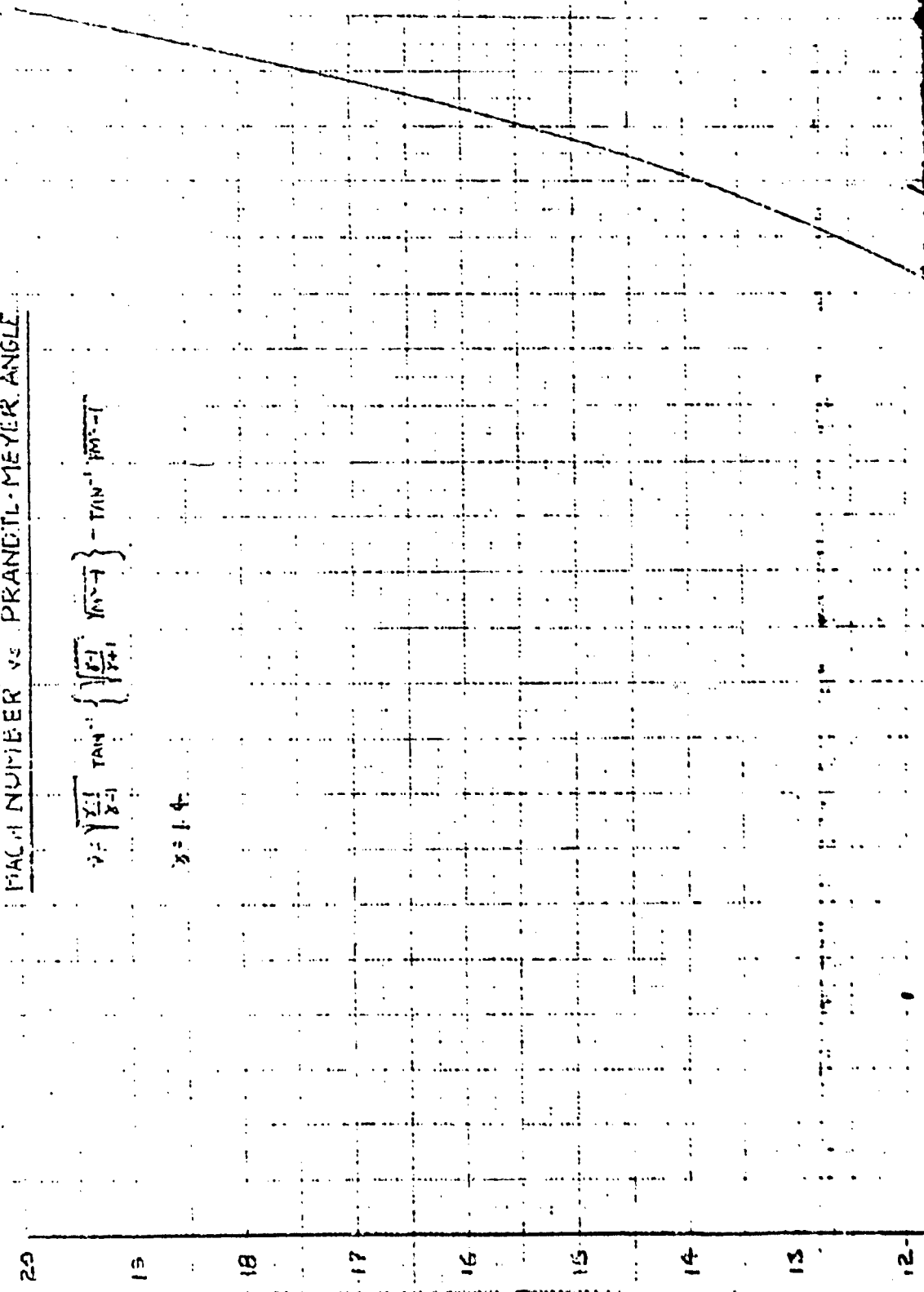
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A

PRANDTL NUMBER VS PRANDTL-MEYER ANGLE

$$N = \sqrt{\frac{\gamma-1}{\gamma+1}} \tan^{-1} \left\{ \sqrt{\frac{\gamma+1}{\gamma-1}} M \sqrt{1-M^2} \right\} - \tan^{-1} M \sqrt{1-M^2}$$

$\gamma = 1.4$



M

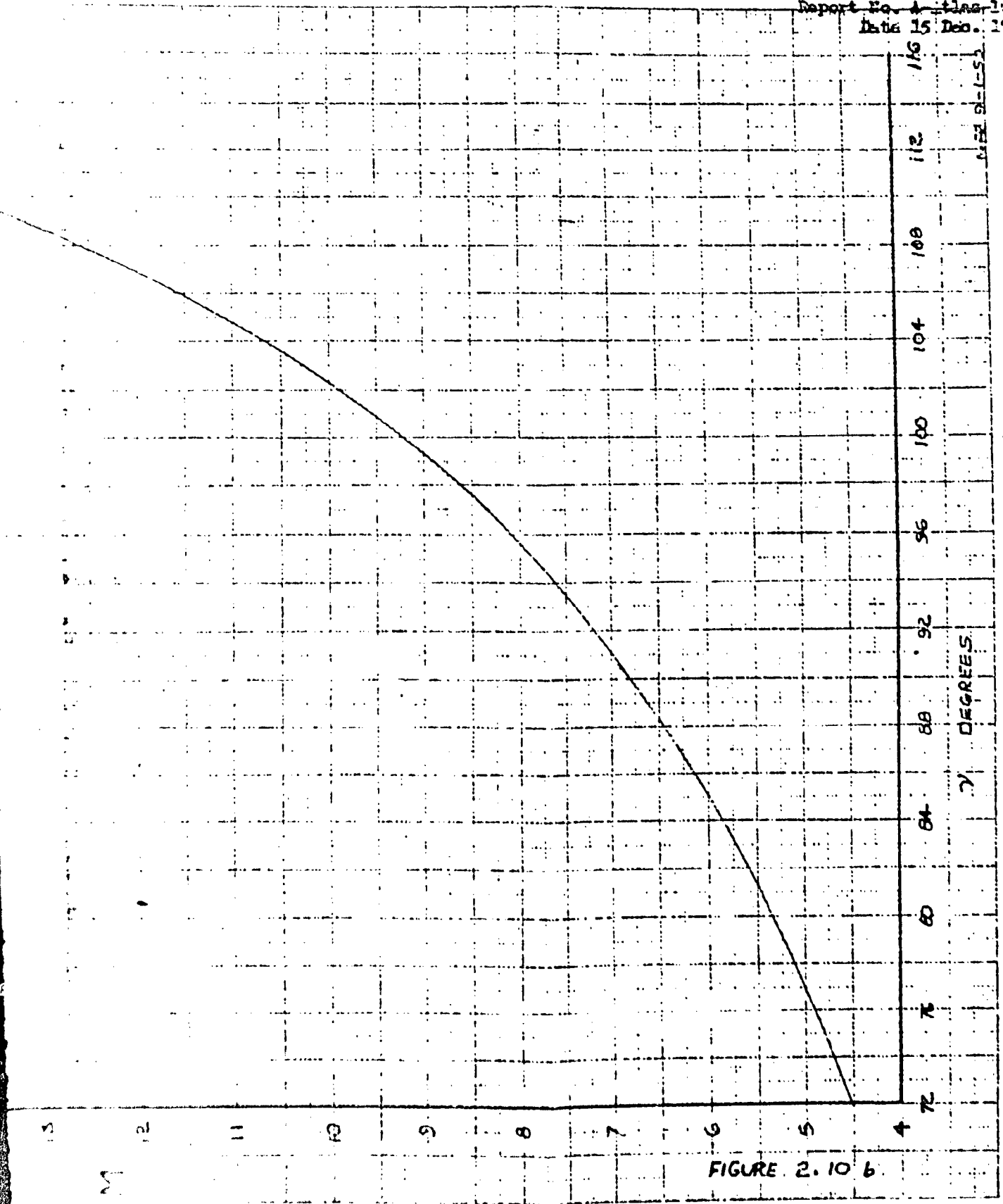


FIGURE 2.10 b

B

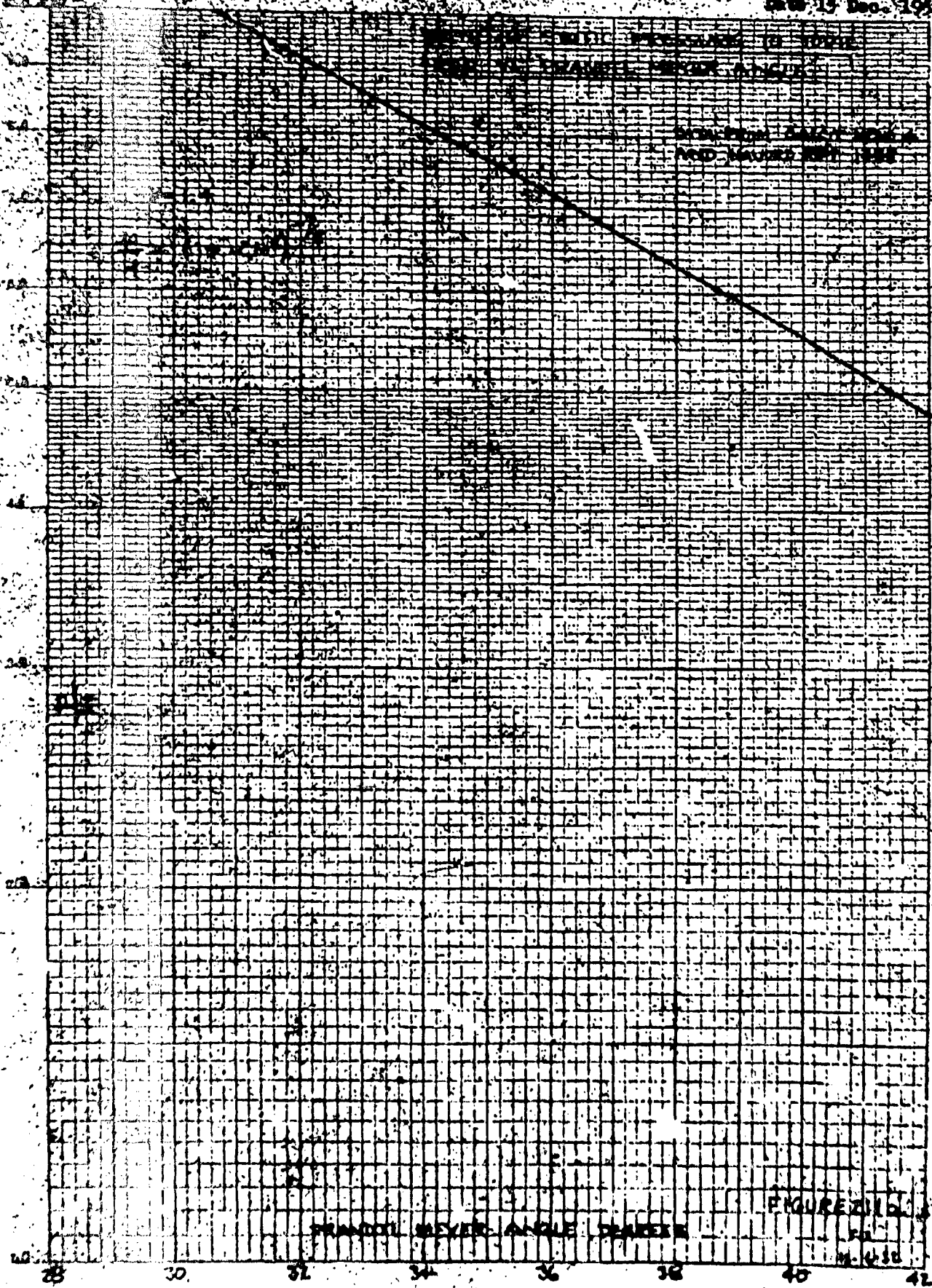
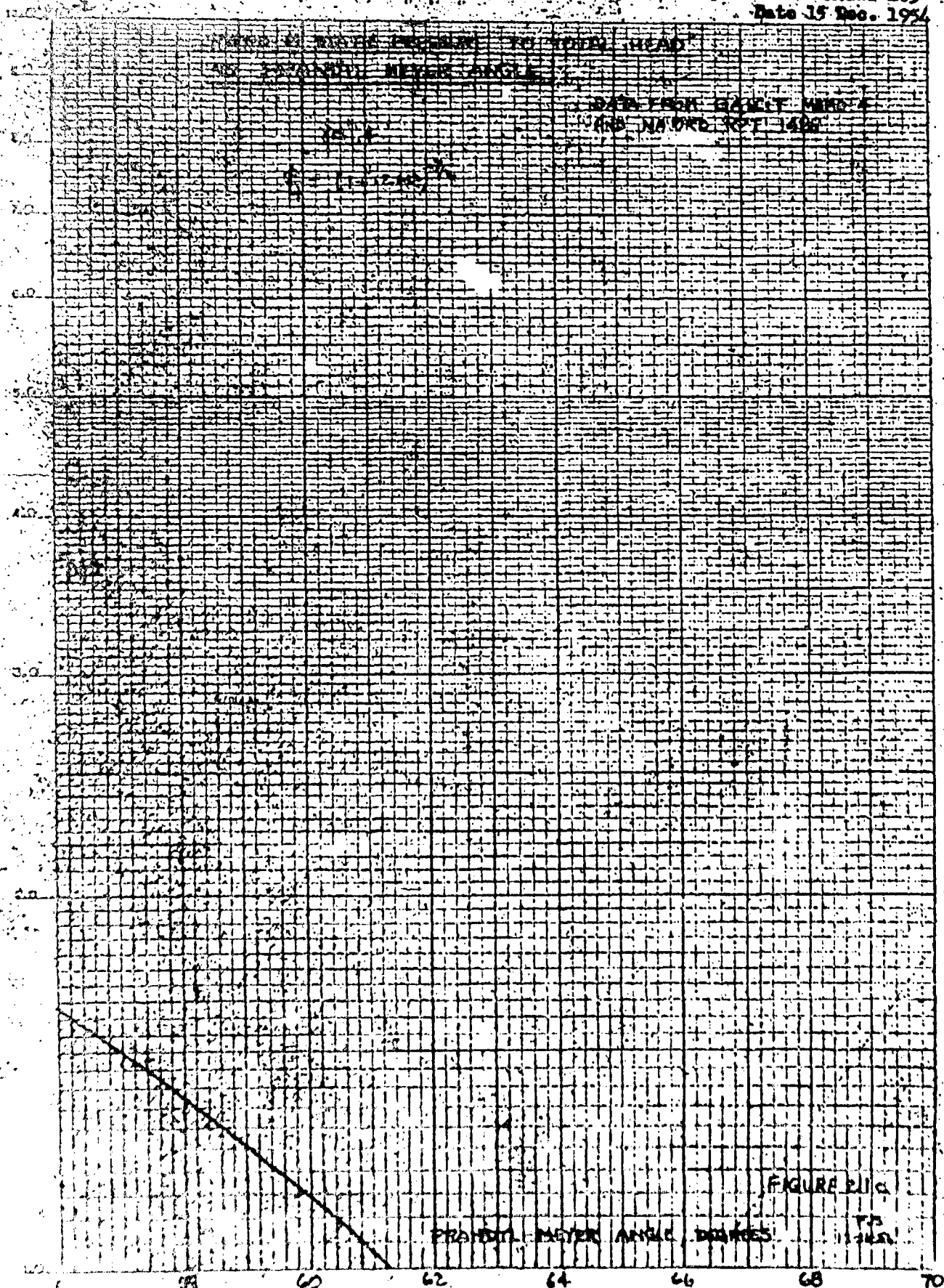


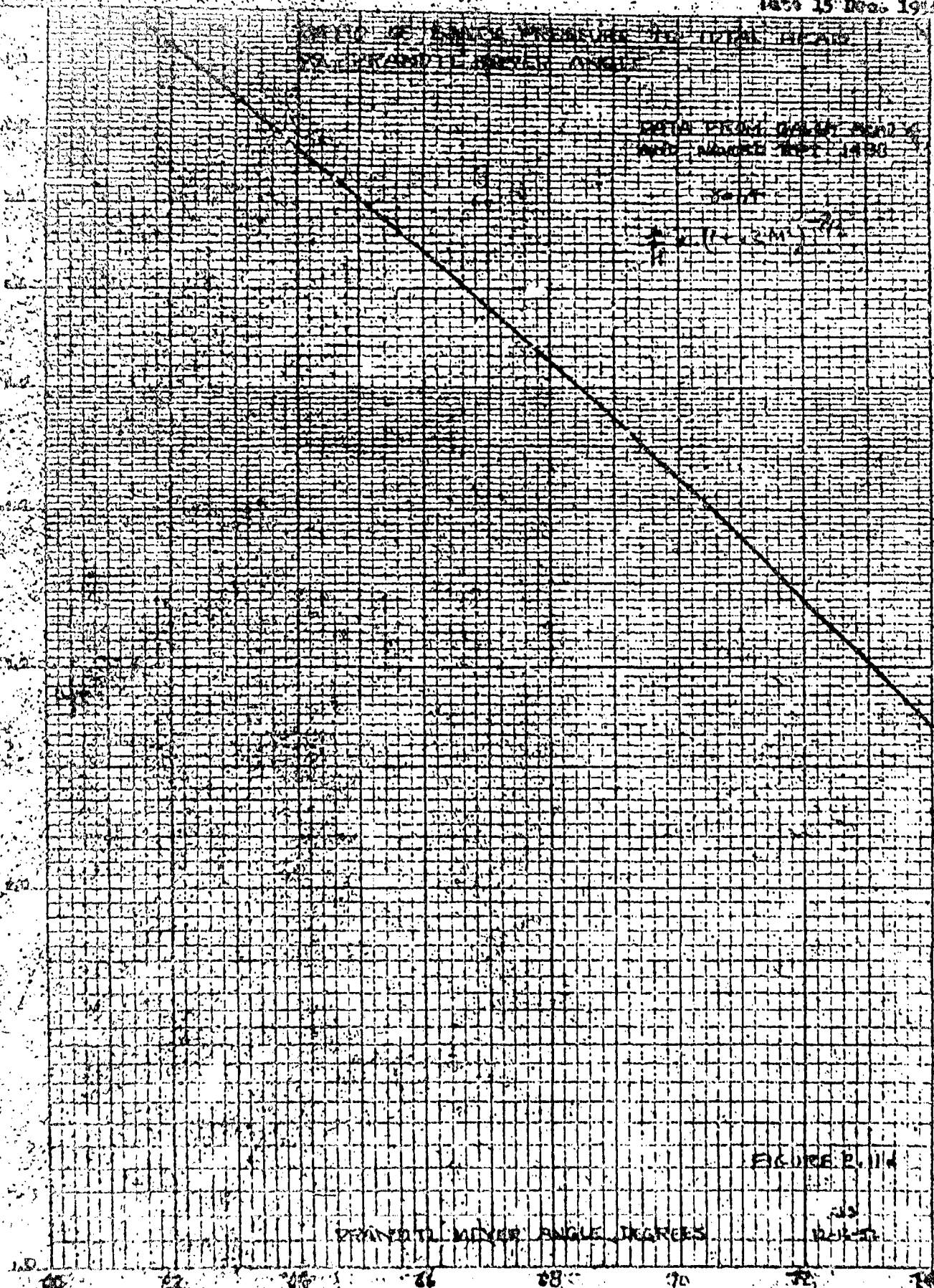
FIGURE 115

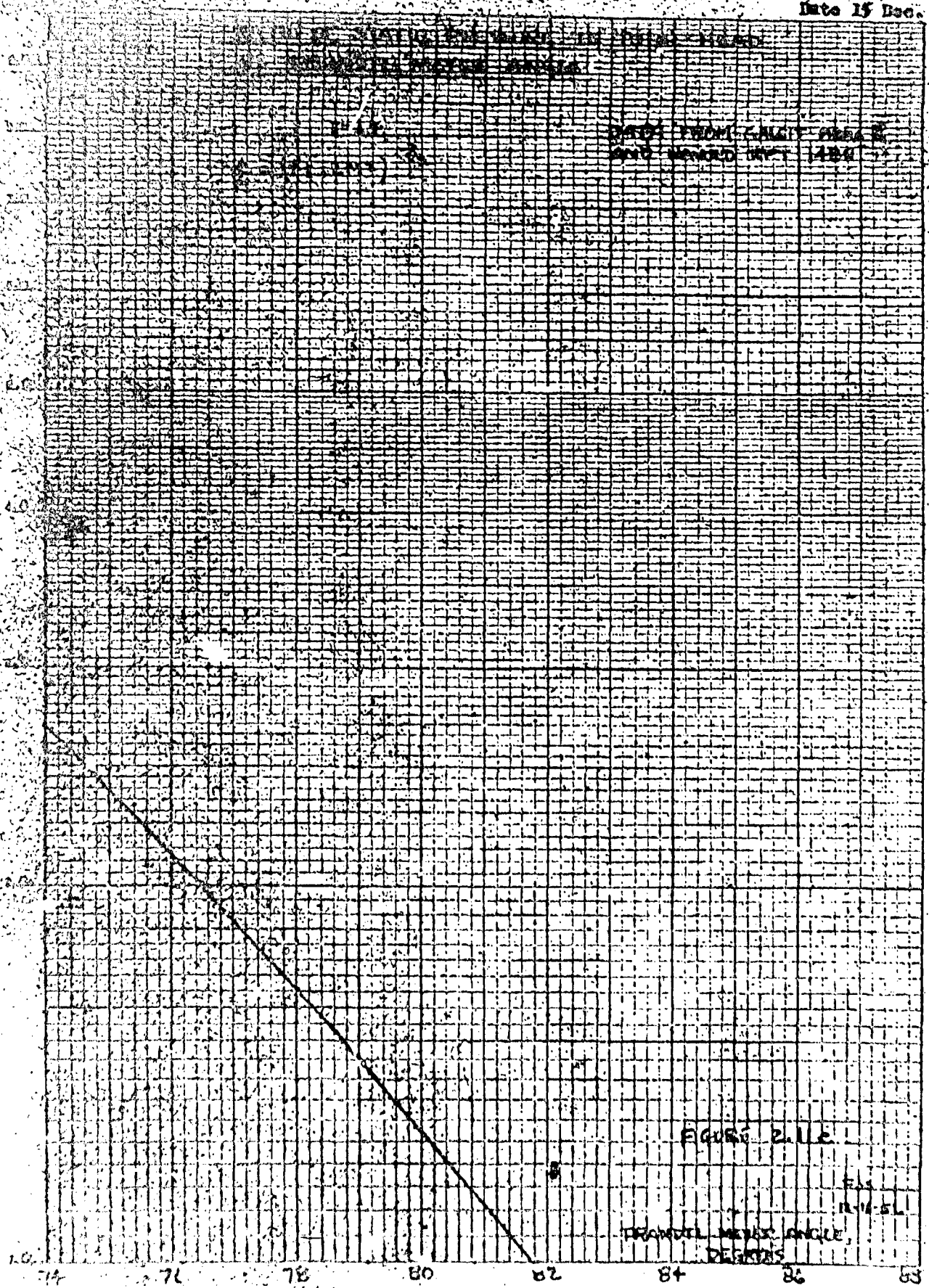
RANDOM MEYER ANGLE DEGREE

M-618









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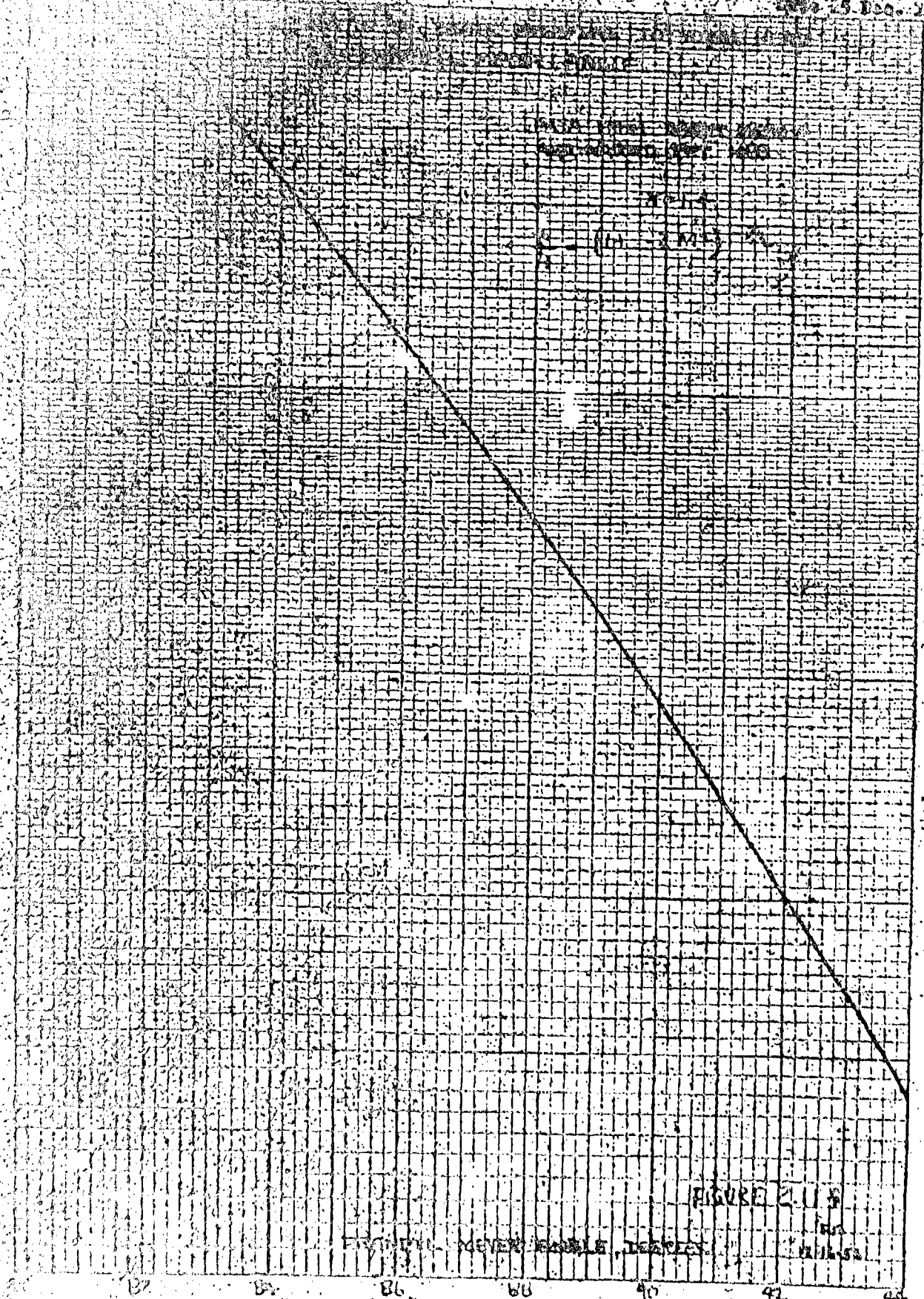
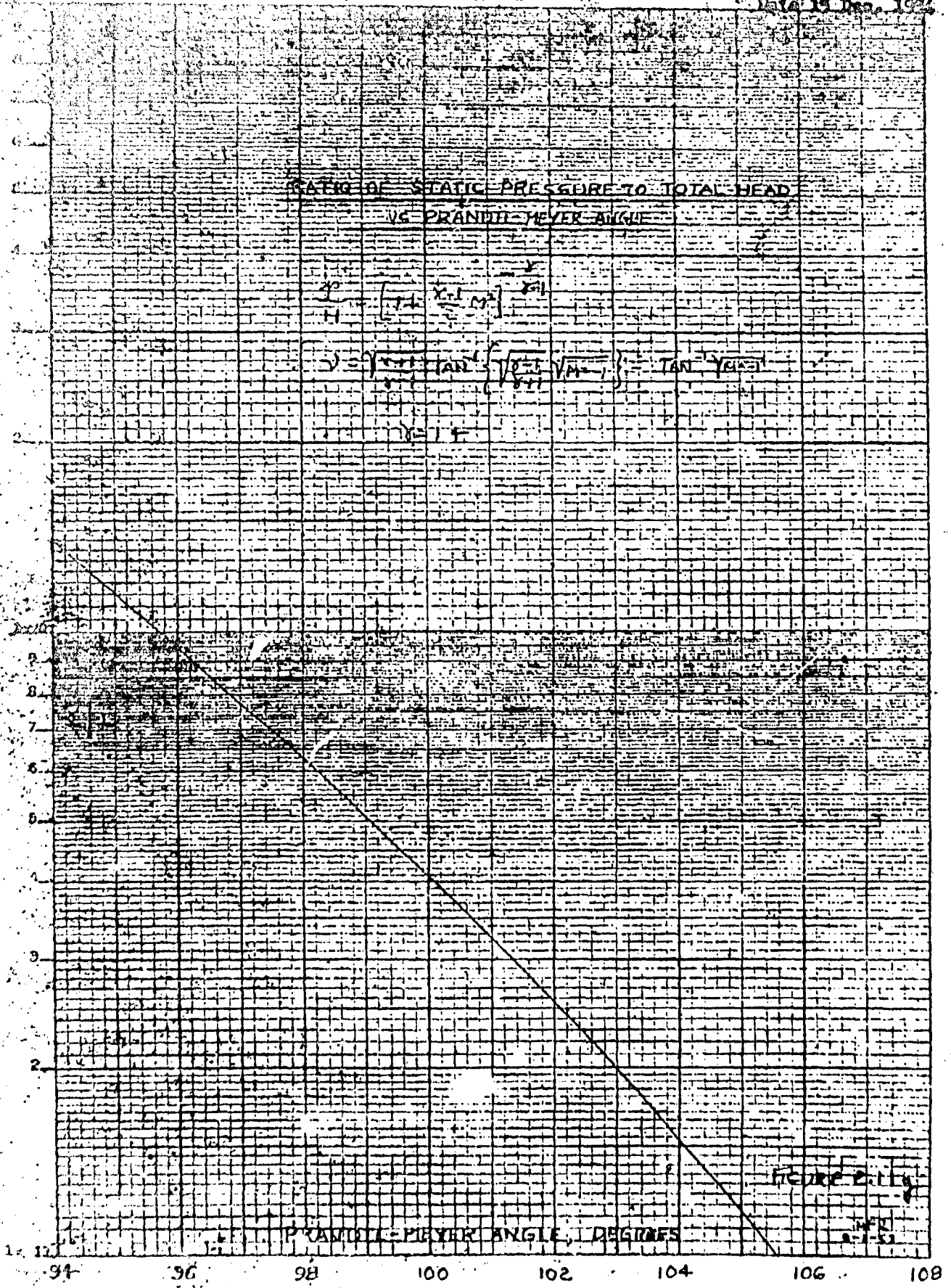


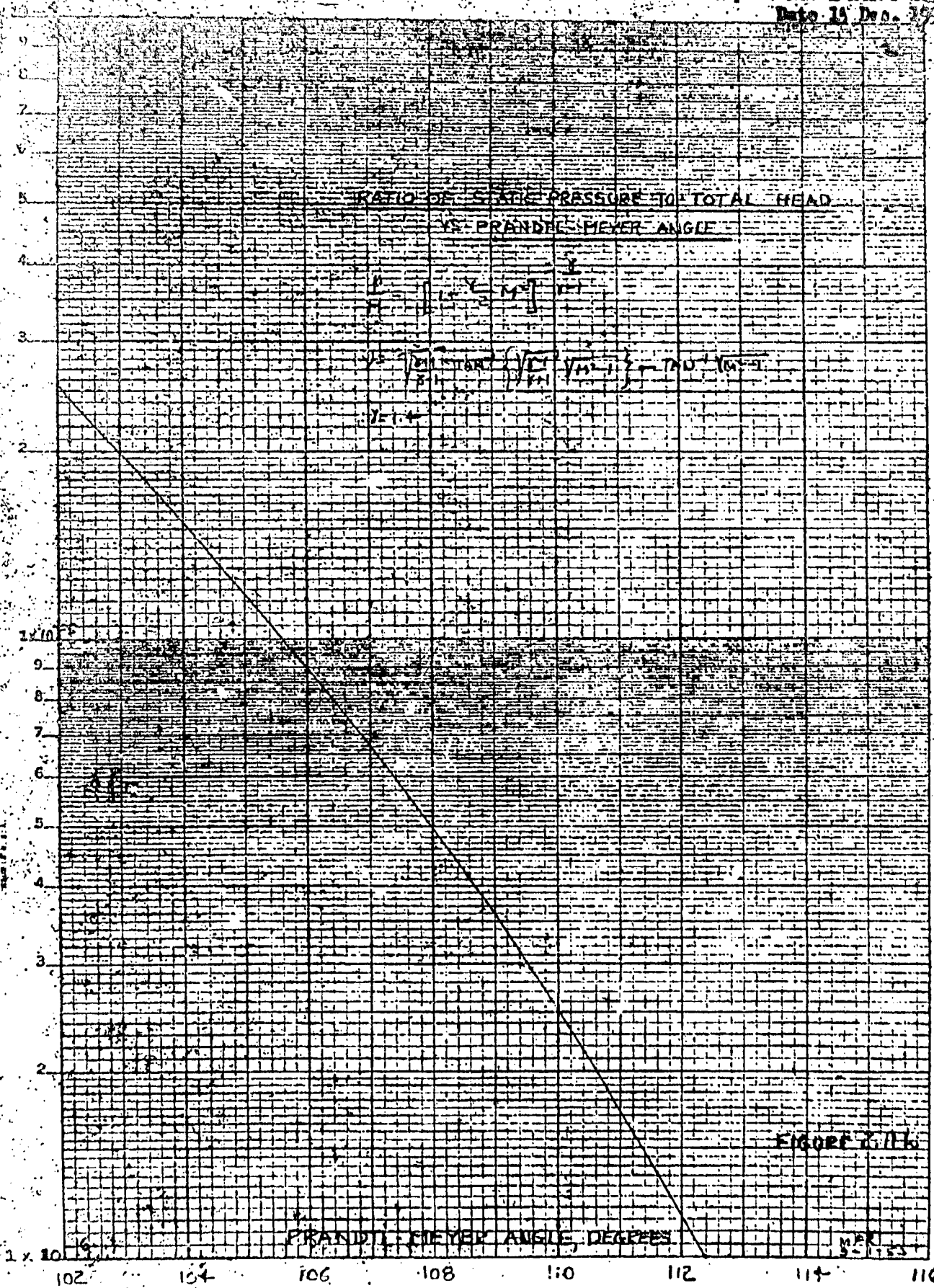
FIGURE 2.1.8

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Sect. 100

FIGURE 10.16

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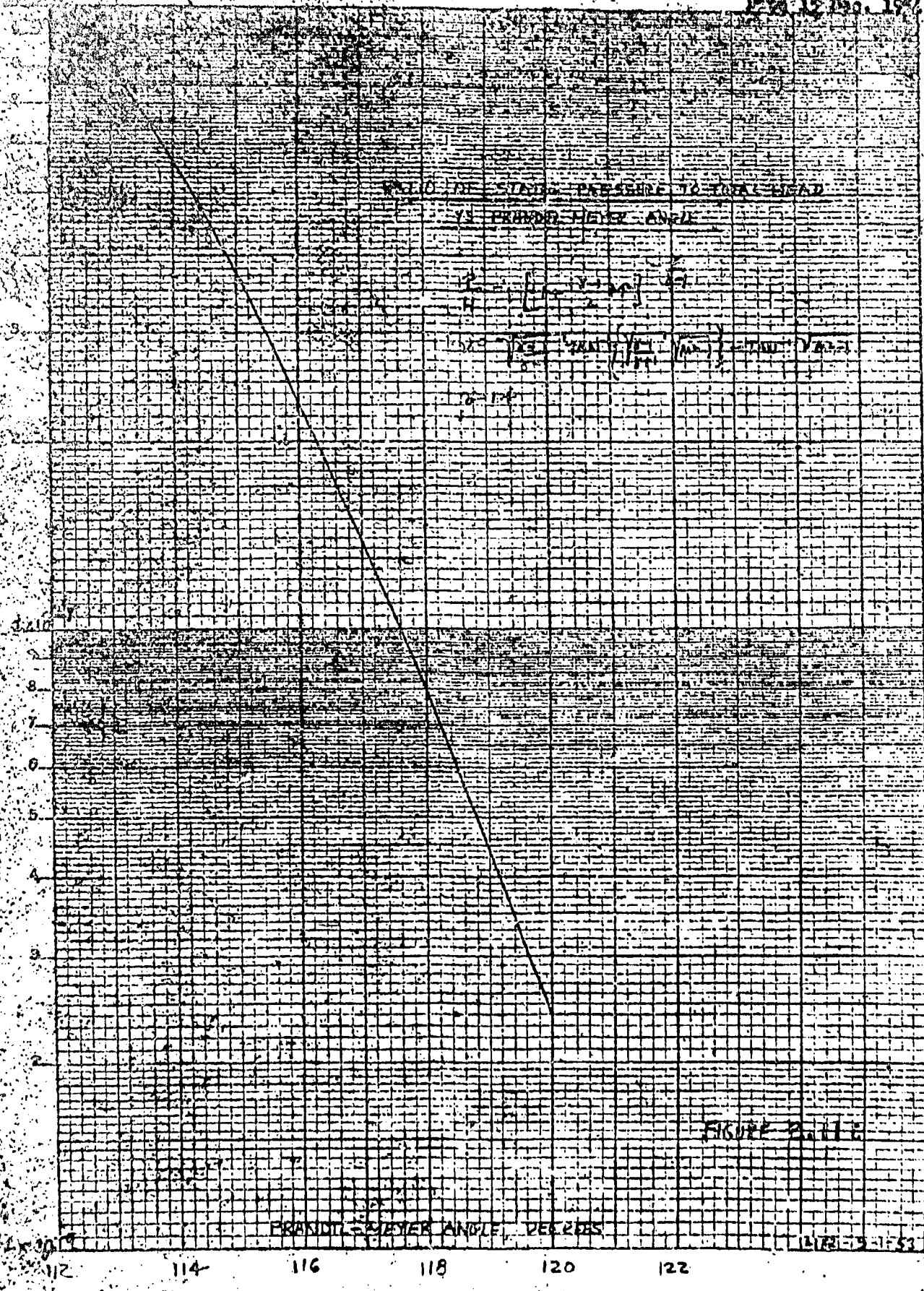
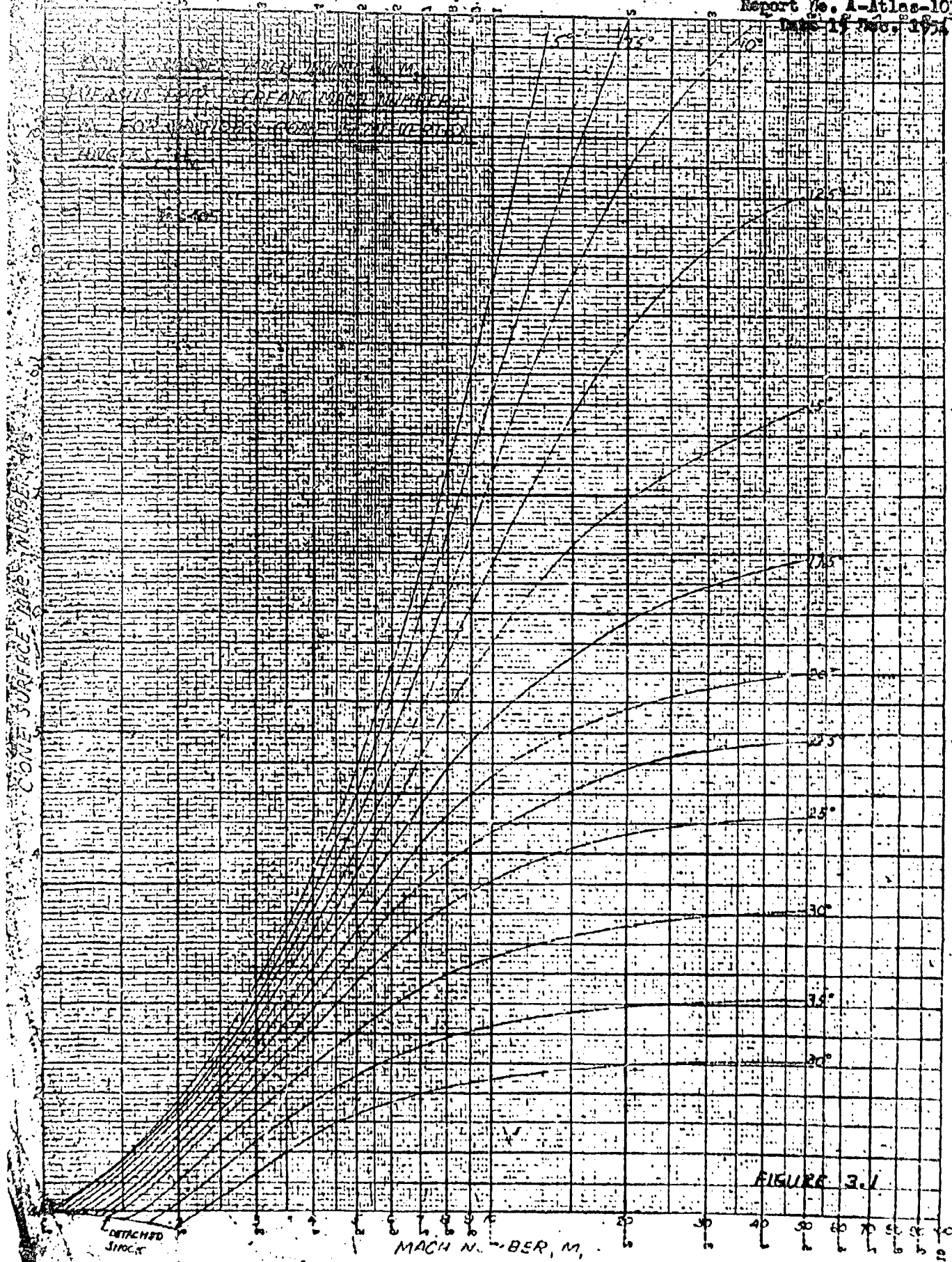


FIGURE 2-11-2

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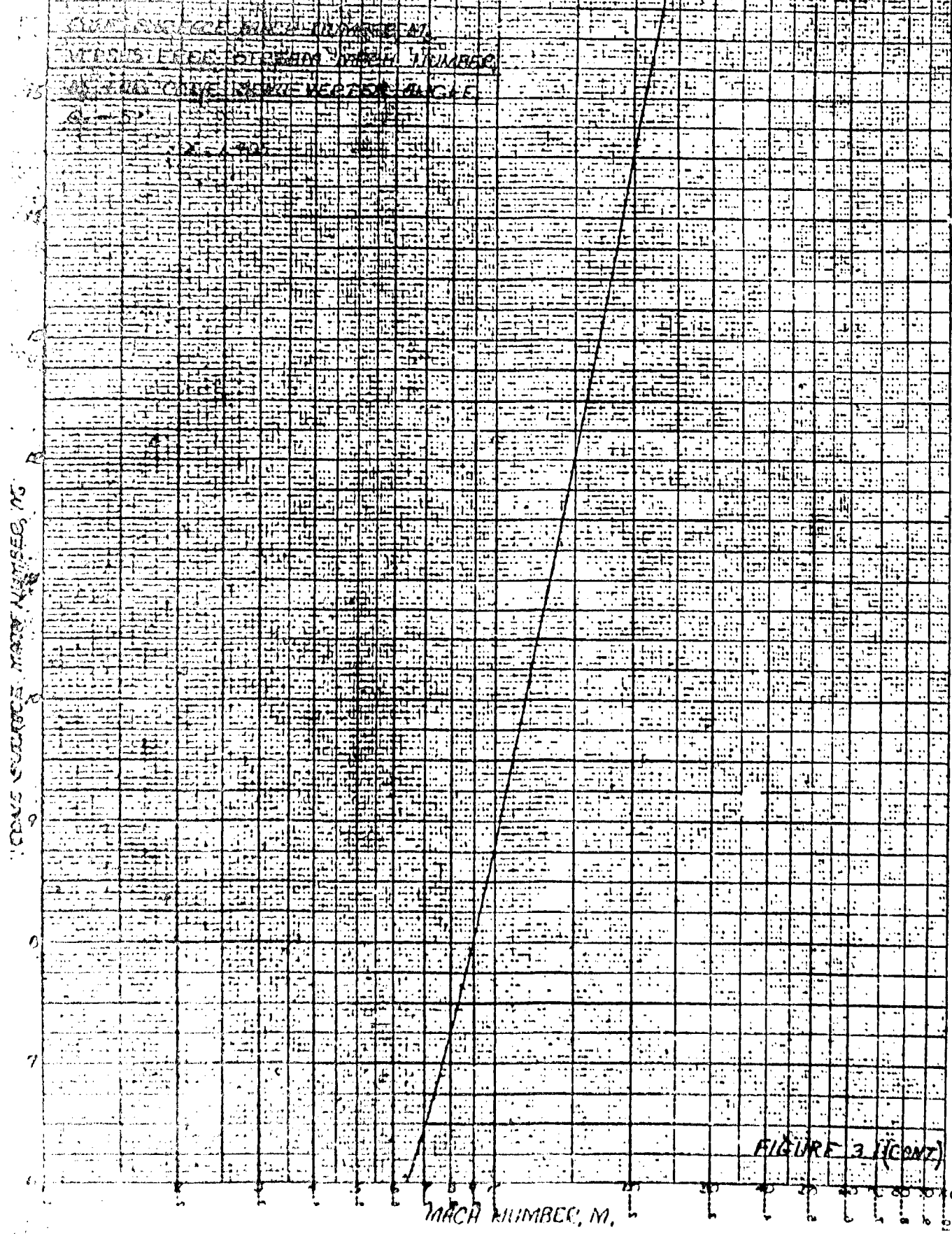


FIGURE 3 (CONT)



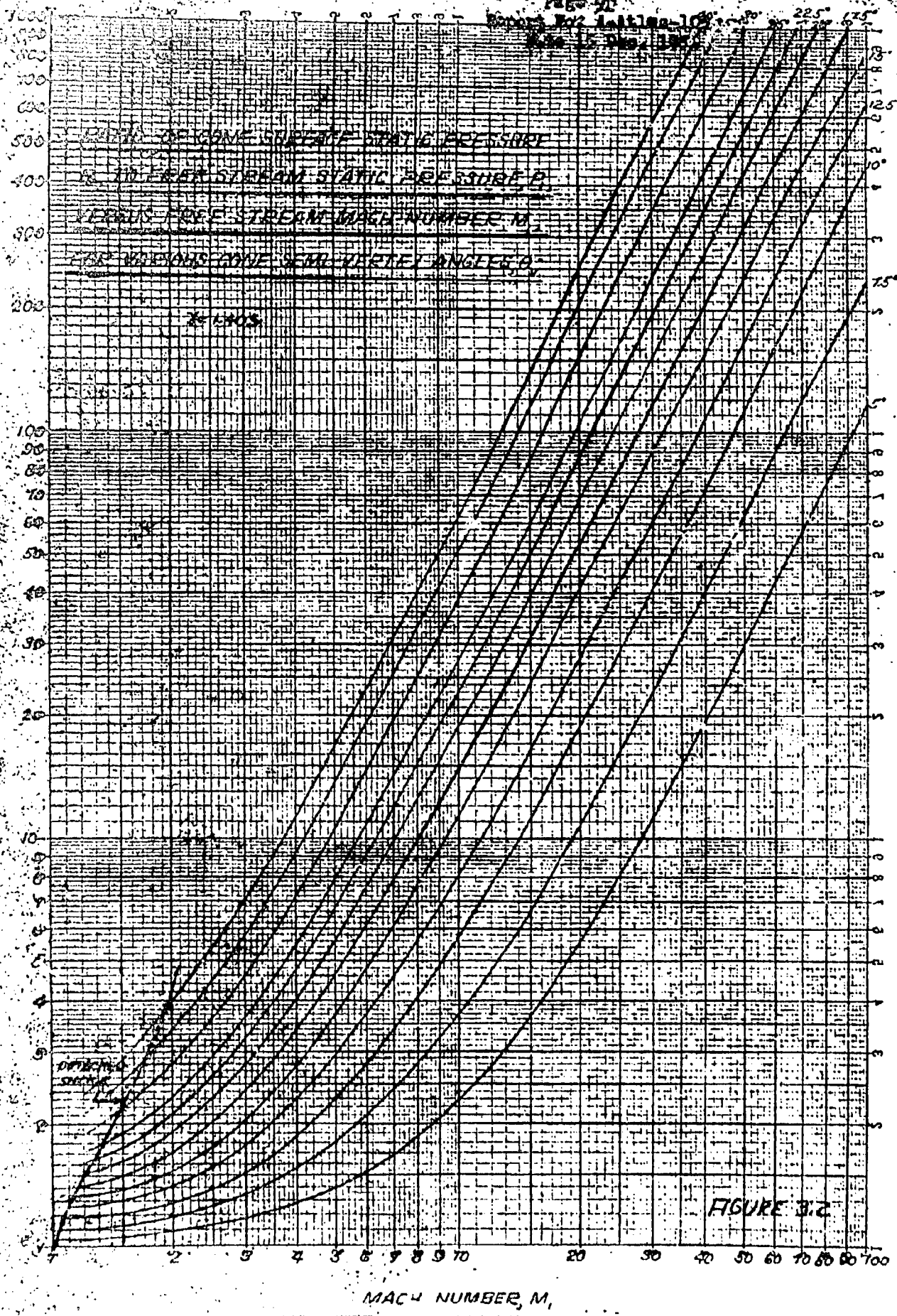


FIGURE 3.2

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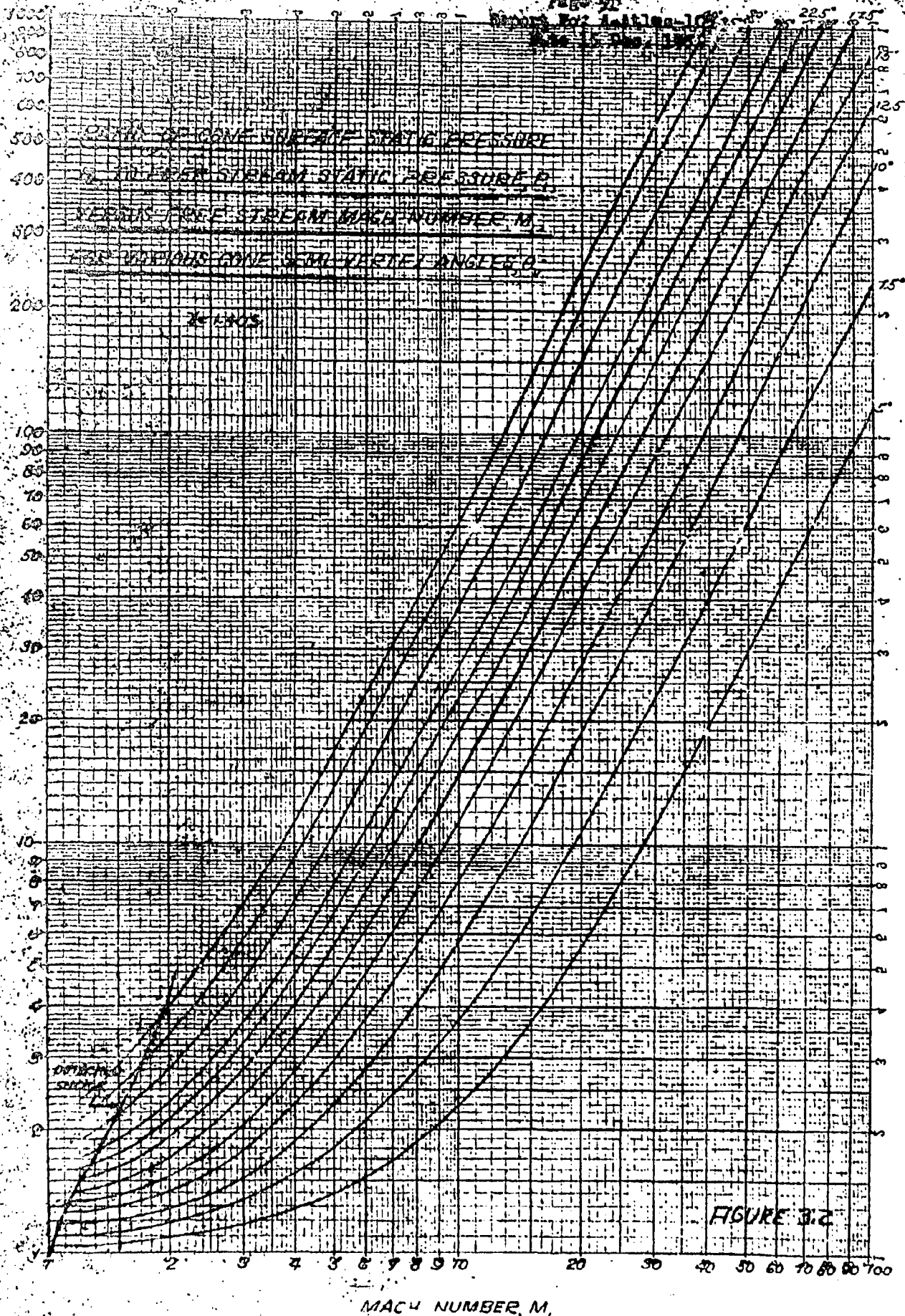
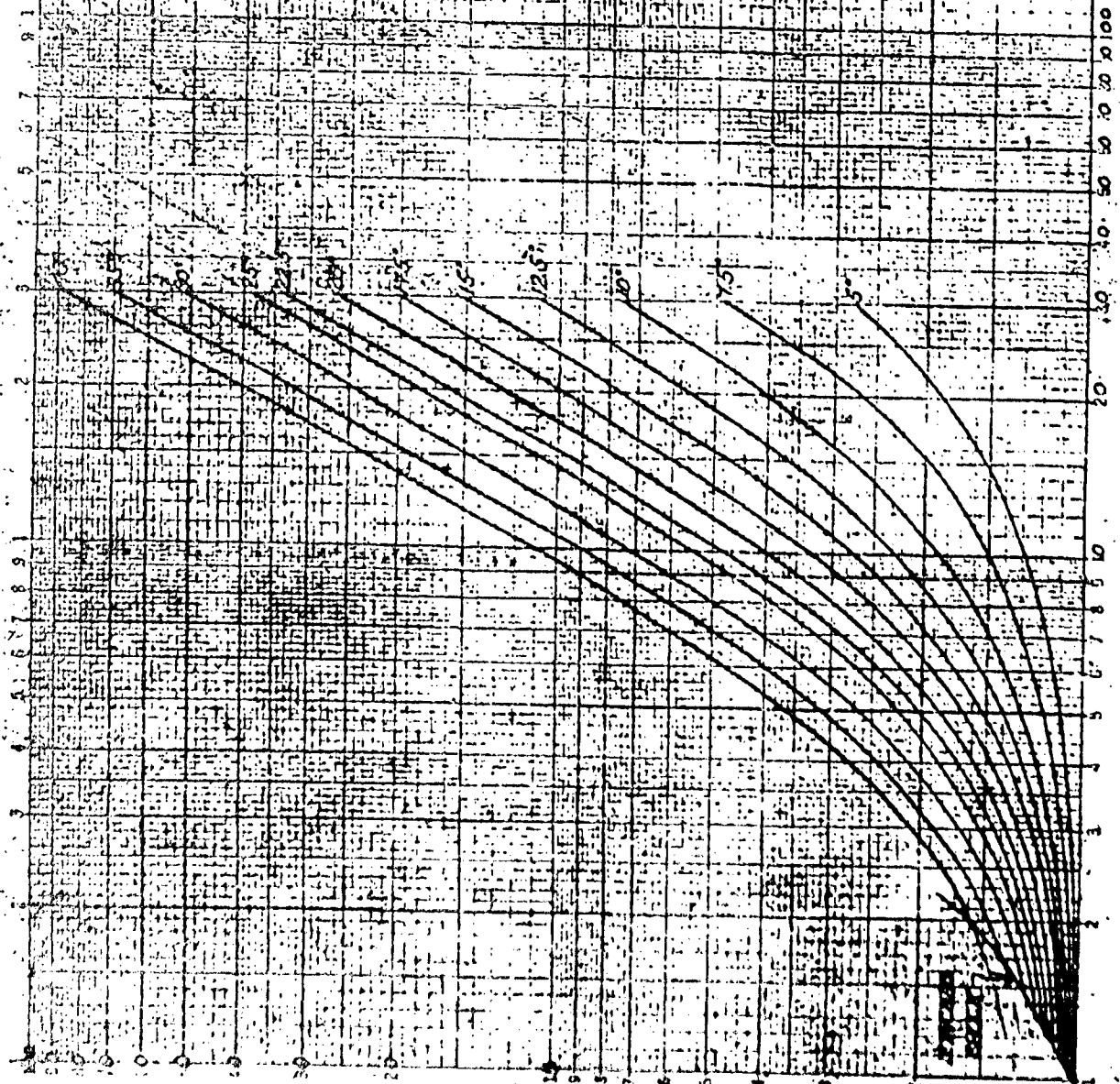


FIGURE 3.2

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FOR CONE 50°  
 FREE STREAM STATIC TEMPERATURE  
 PRESSURE & DENSITY  
 FOR VARIOUS CONE SEMI-ANGLE  $\theta = 50^\circ$

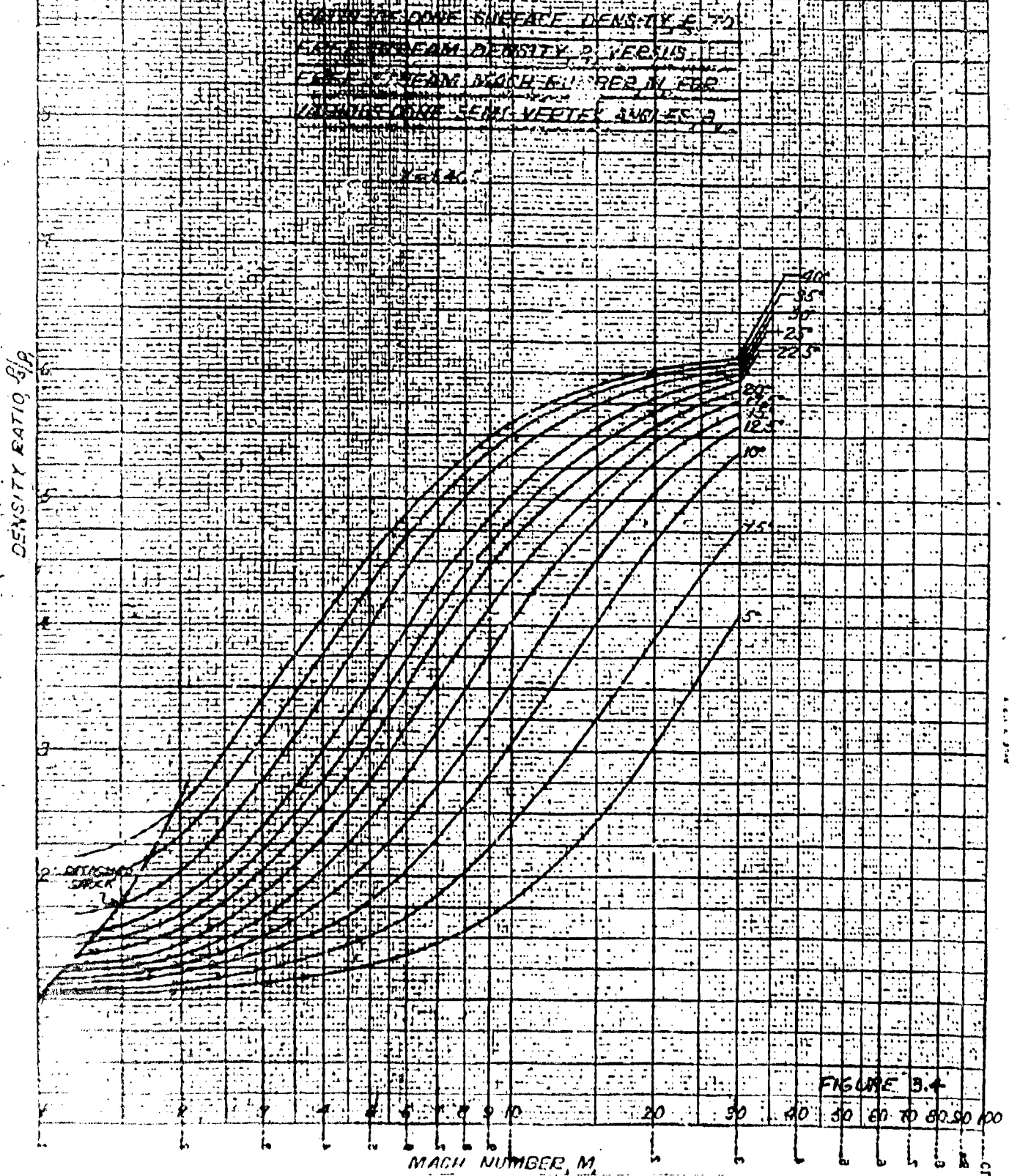
$\gamma = 1.405$



MACH NUMBER,  $M$

STATIC TEMPERATURE RATIO,  $T/T_\infty$

FIGURE 33





CONE SHOCK-WAVE ANGLE,  $\theta_w$ , VERSUS FREE STREAM  
MACH NUMBER,  $M_\infty$ , FOR VARIOUS CONE SEMI-VERTEX  
ANGLES,  $\theta_v$

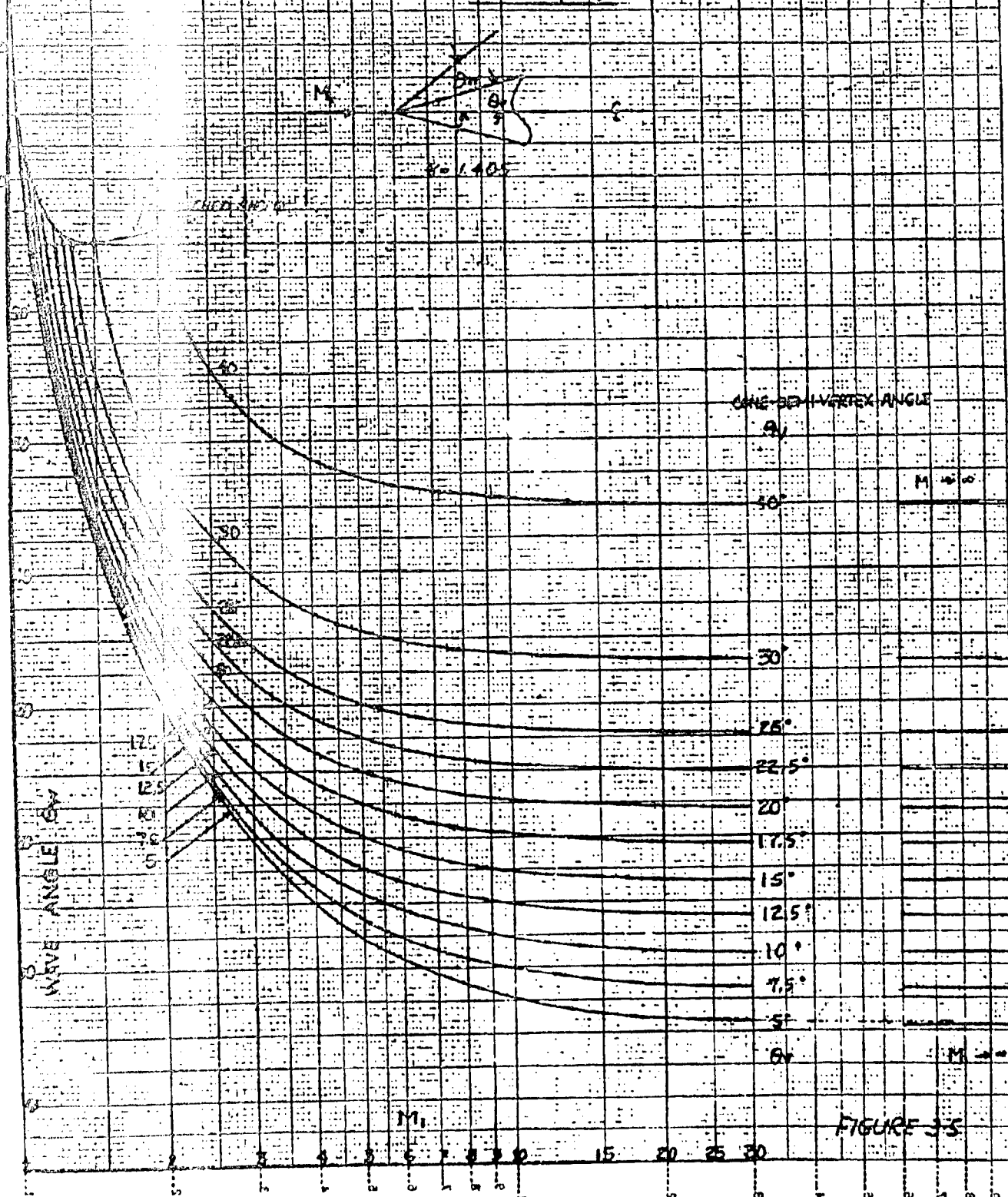
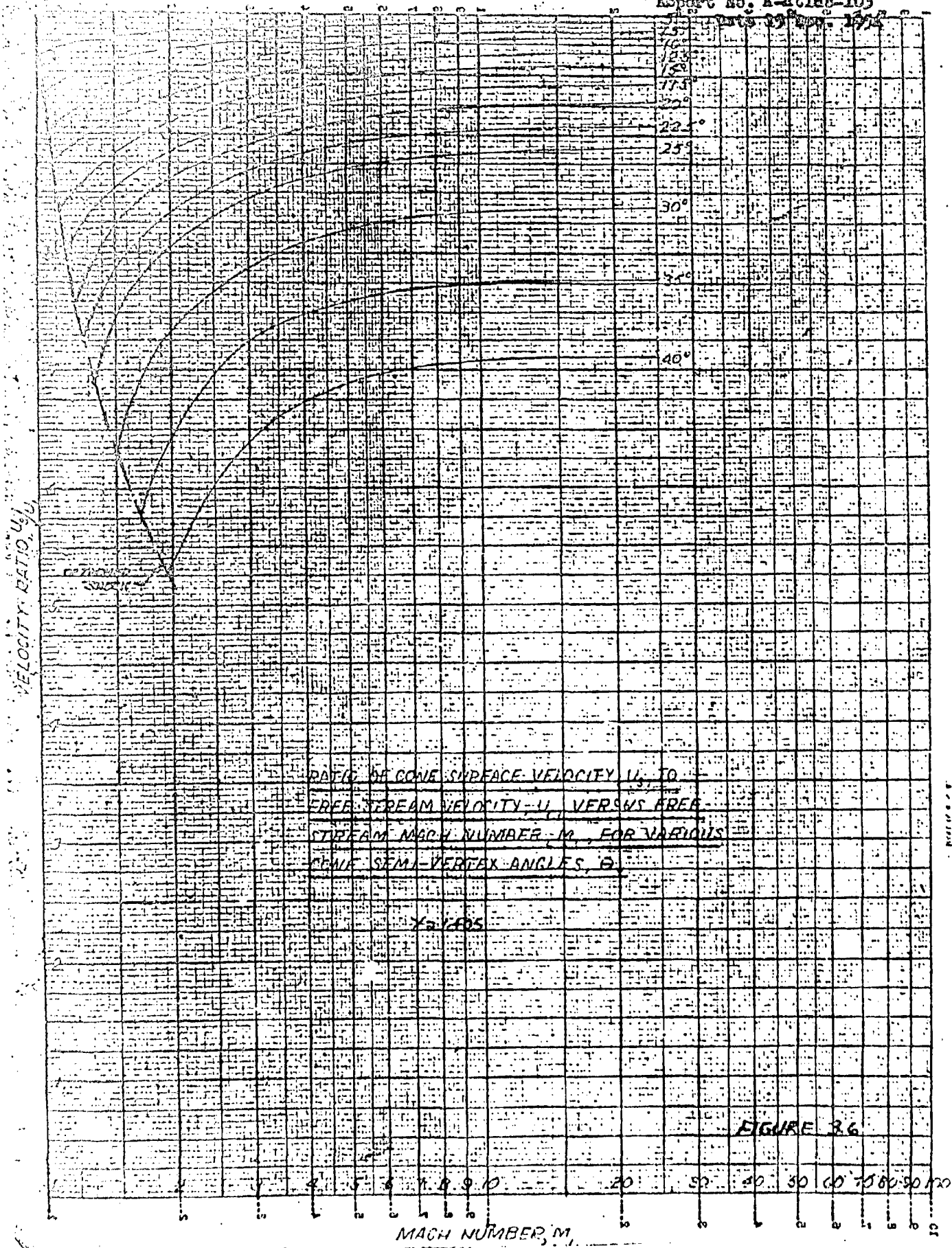


FIGURE 3-5

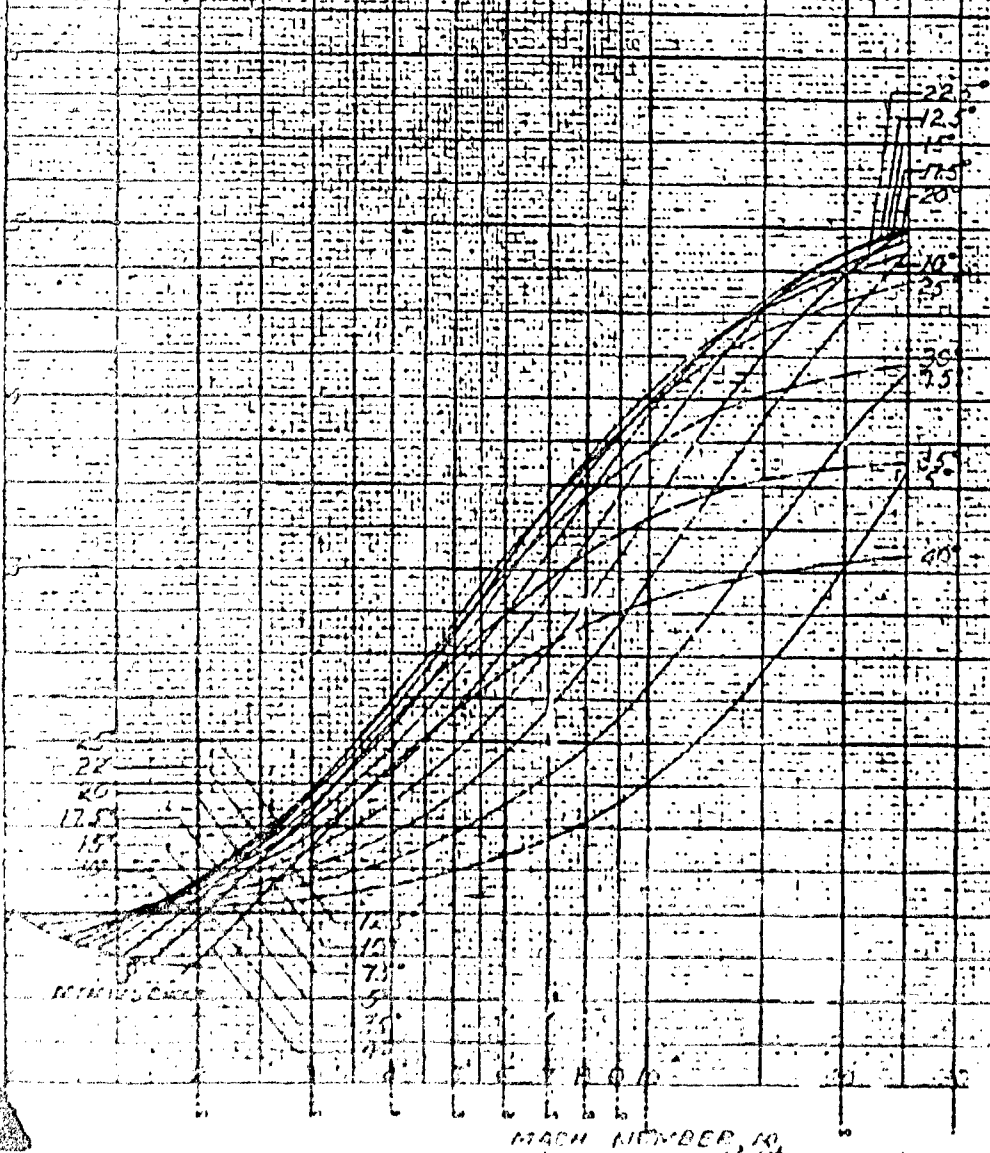


RATIO OF CONE SURFACE VELOCITY,  $U_s$ , TO  
 FREE STREAM VELOCITY,  $U_\infty$ , VERSUS AREA-  
 STREAM MACH NUMBER,  $M$ , FOR VARIOUS  
 CONE SEMI-VERTEX ANGLES,  $\theta$

FIGURE 3.6

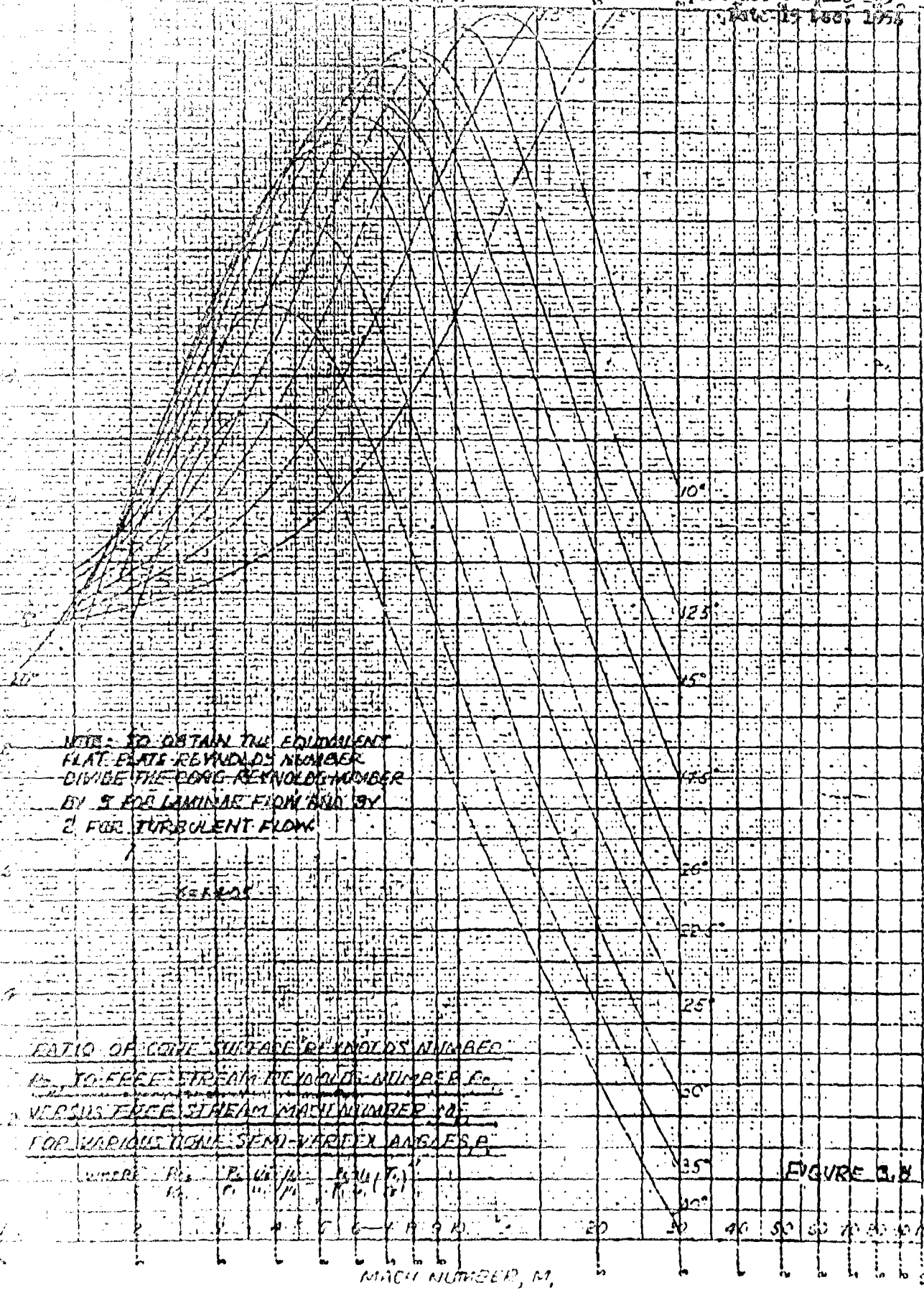
RATIO OF CONE SURFACE DYNAMIC PRESSURE  
TO FREE-STREAM DYNAMIC PRESSURE,  
VERSUS FREE-STREAM MACH NUMBER,  $M_\infty$ ,  
FOR CONES OF SEMI-VERTEX-ANGLE  $\theta$

DYNAMIC PRESSURE RATIO,  $\frac{q_c}{q_\infty}$



MACH NUMBER,  $M_\infty$

REYNOLDS NUMBER RATIO,  $R_c/R_\infty$





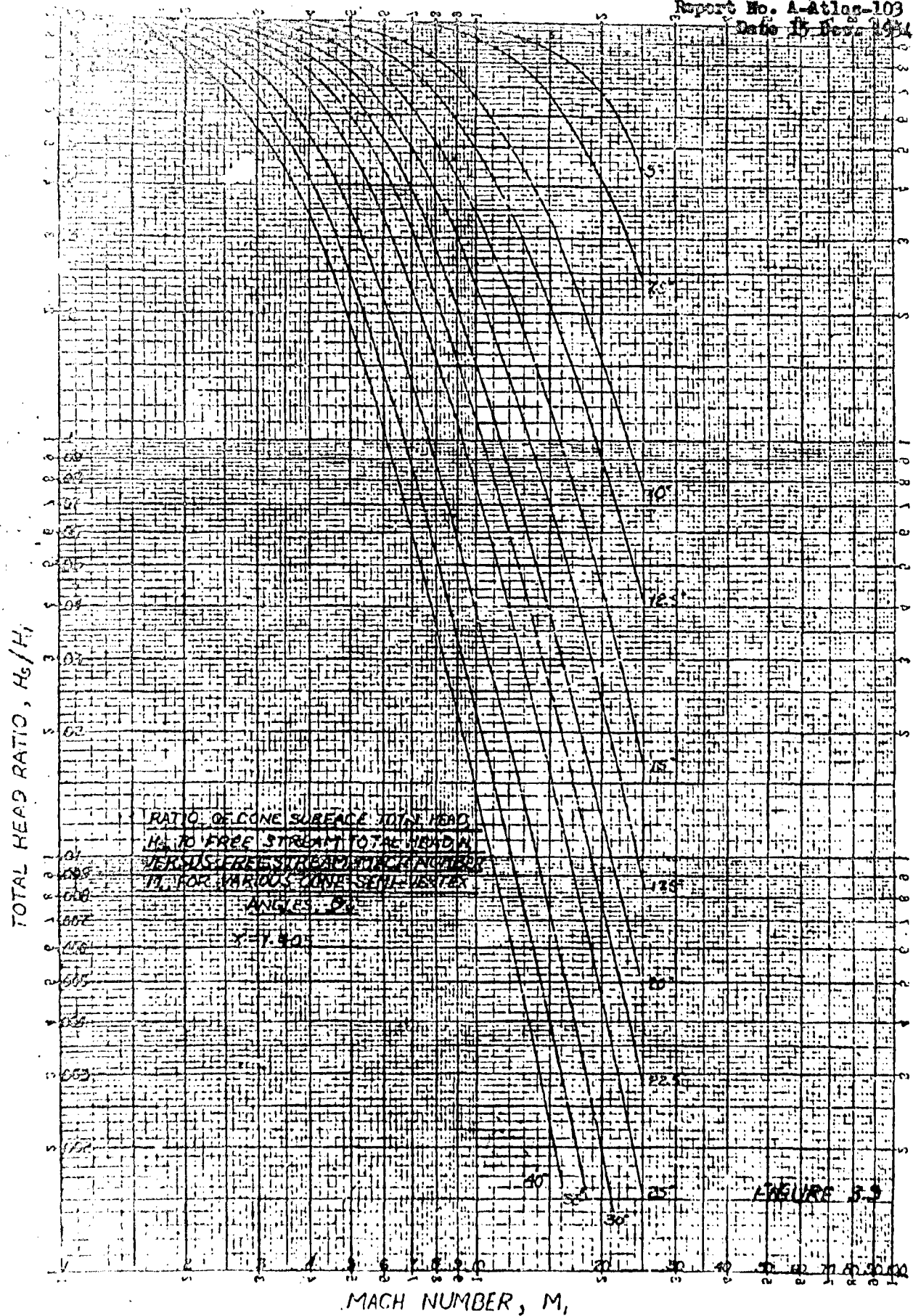


FIGURE 3.9

# WAVE DRAG COEFFICIENT OF A CONE

VS. CONE HALF-ANGLE

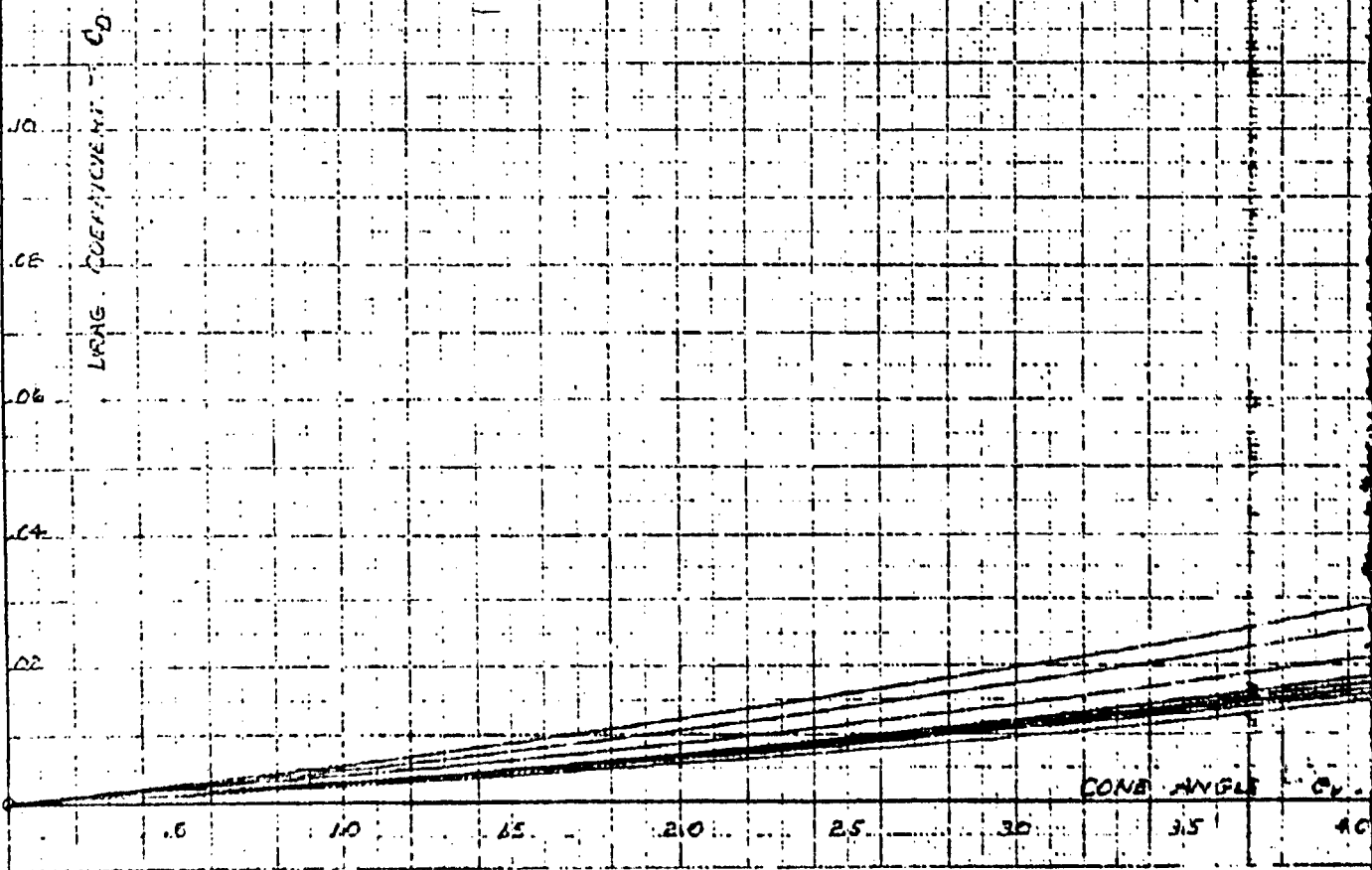
TAYLOR-MACCOLL VALUES

$A_{REF}$  = BASE AREA OF CONE

FOR MACH. NUMBERS 1.5-8.0

$\phi = 0-2.5^\circ$

$\chi = 1.405$



$C_D$  vs  $\theta_v$

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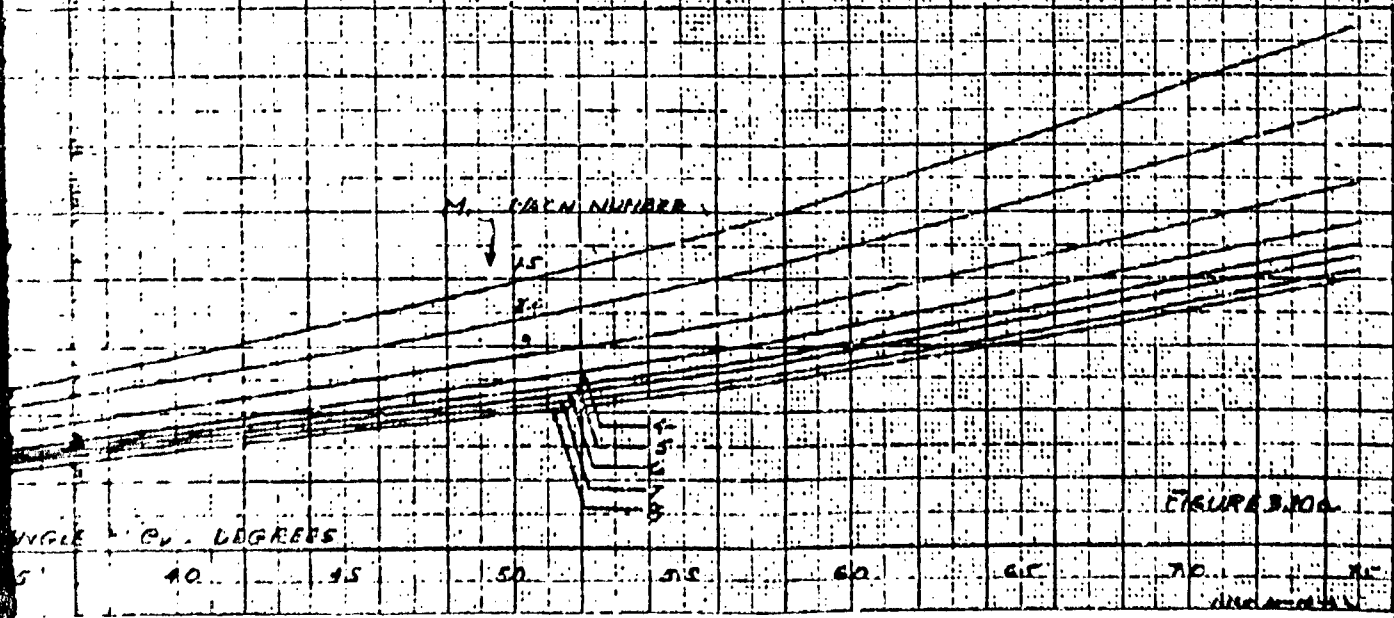


FIGURE 3.100

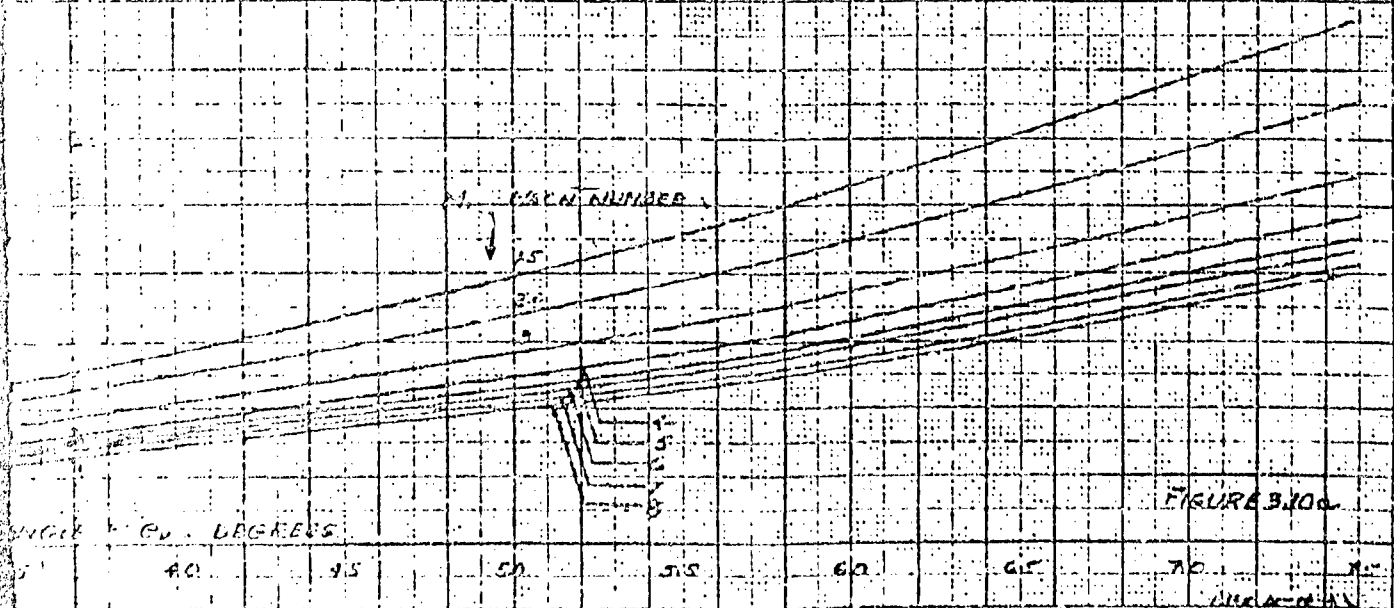
B

$C_D$  vs  $\Theta_v$

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B

# WAVE DRAG COEFFICIENT OF A CONE

CONE HALF-ANGLE

THORPE-EMSCOTT VALUES

BASE = BASE WAVE OF CONE

FOR WHICH NUMBERS 1.5-2.0

$\alpha = 7.5^\circ - 15^\circ$

$\gamma = 1.405$

DRAG COEFFICIENT  $C_D$

24

22

20

18

16

14

12

10

8

6

4

CONE ANG

$\alpha$

75

80

85

90

95

100

105

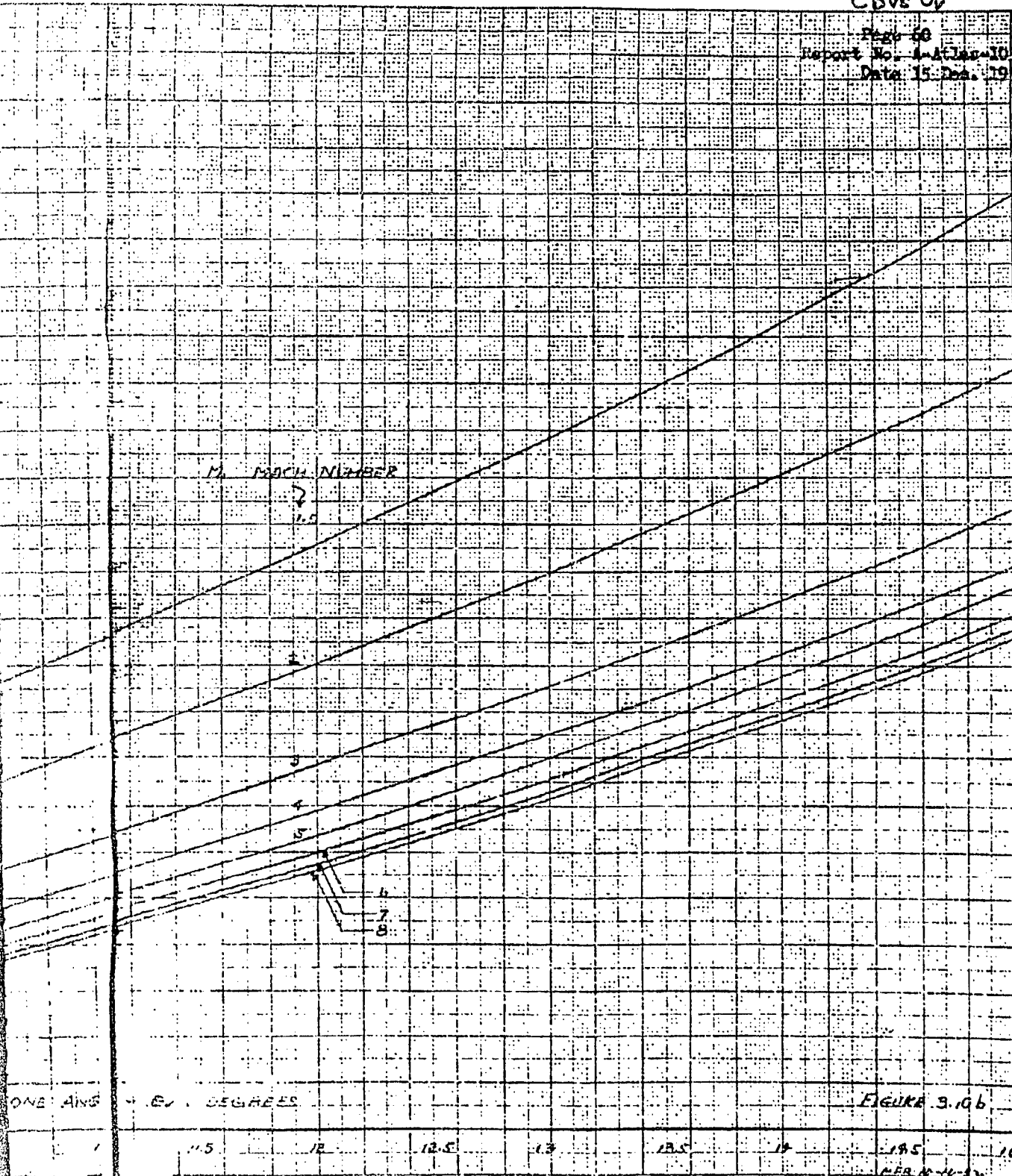
110

115

A

Covs' @

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Date 15 Dec. 1954

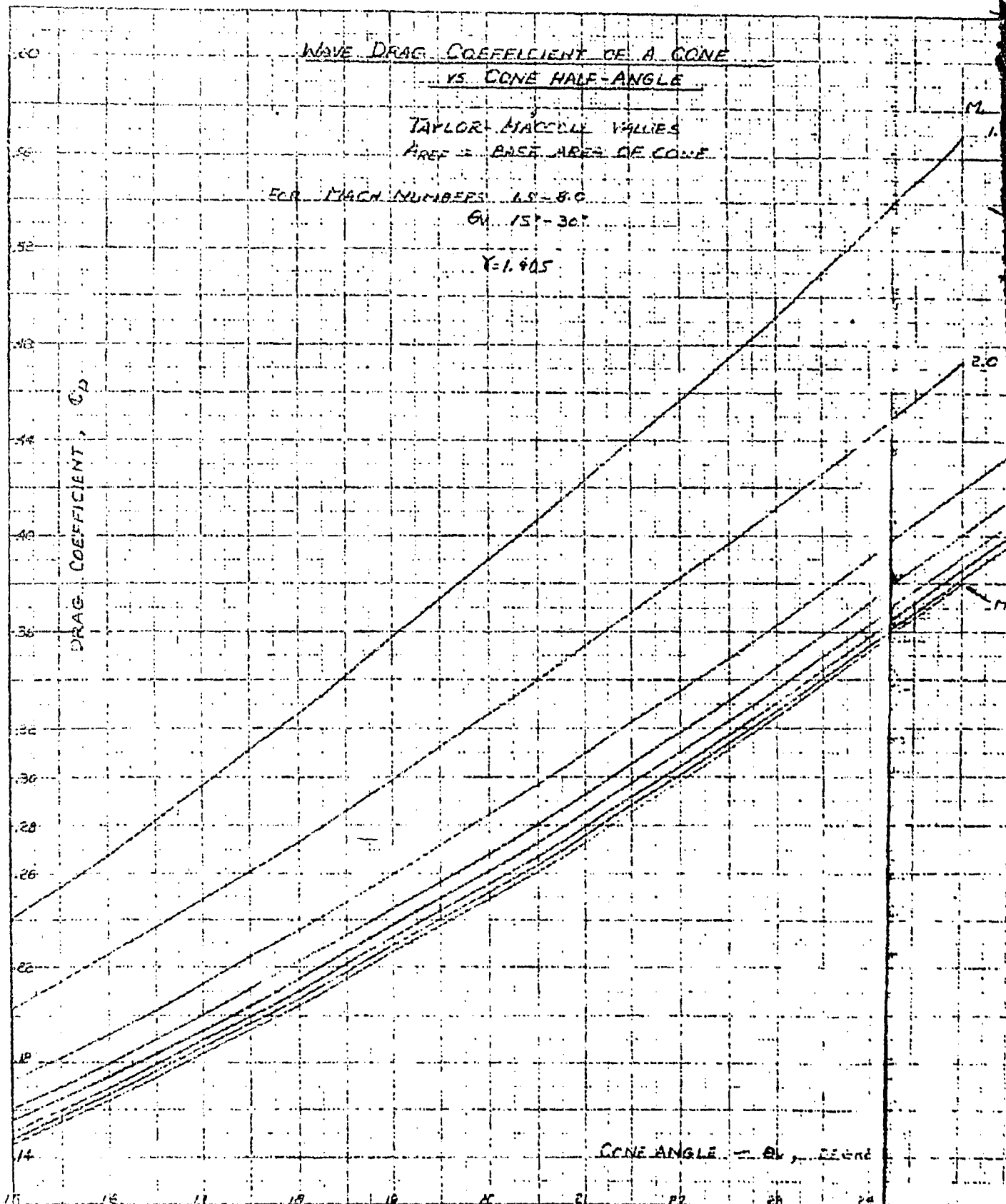




$$Y = 1.945$$

DRAG COEFFICIENT,  $C_D$

CONE ANGLE =  $2\pi$  rad





$C_D$  vs  $\theta_v$

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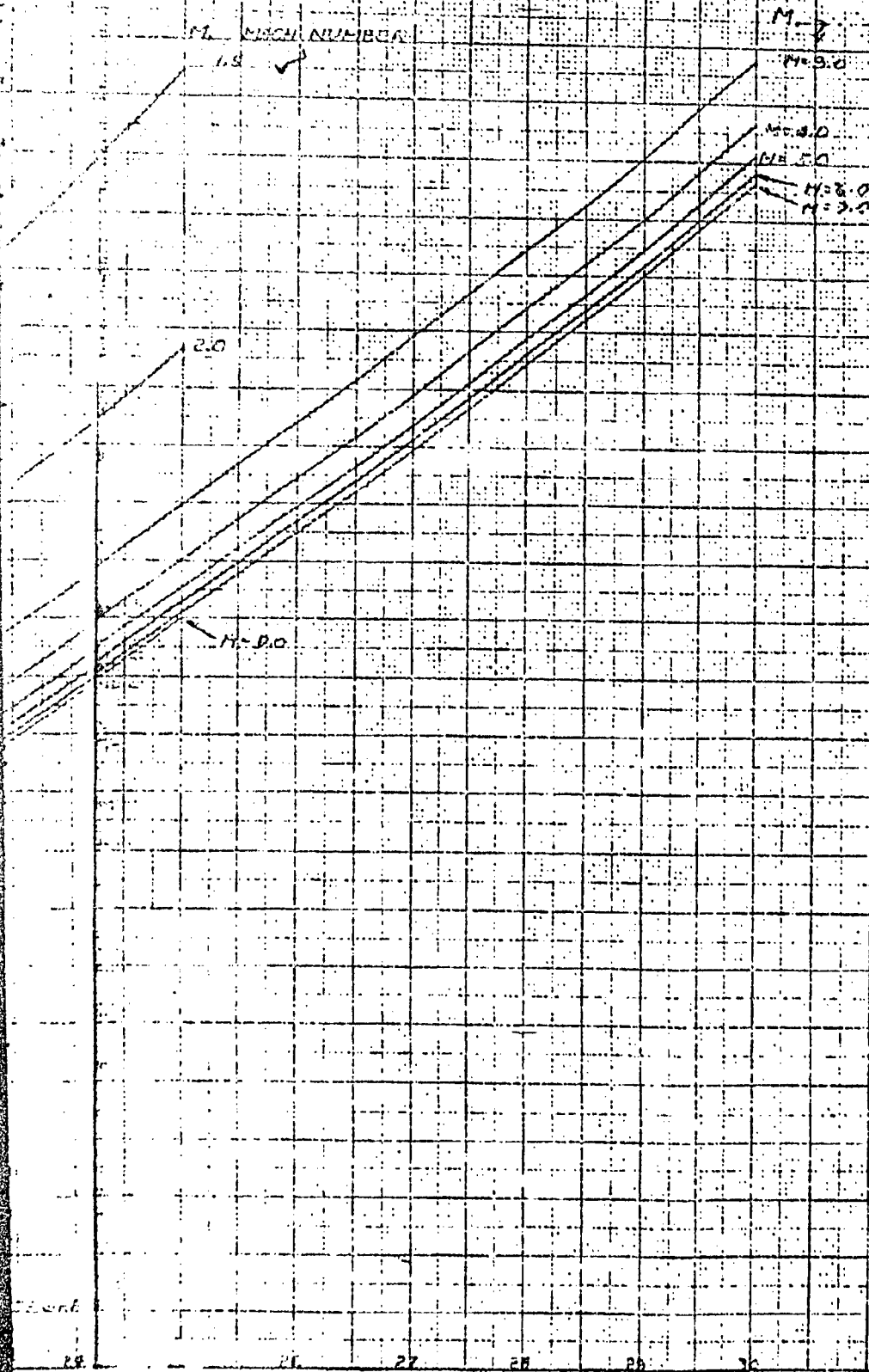


FIGURE 3.10c

DPA-10-20-51

B

WAVE DRAG COEFFICIENT OF A CONE  
VS. CONE HALF-ANGLE

TAYLOR - MACCOLL VALUES

$H_{max}$  = BASE AREA OF CONE

REYNOLDS NUMBERS 10, 12.5, 15

$\theta = 0-15^\circ$

$\delta = 1.405$

DRAG COEFFICIENT  $C_D$

16

14

12

10

08

06

04

02

0

C

1

2

3

4

5

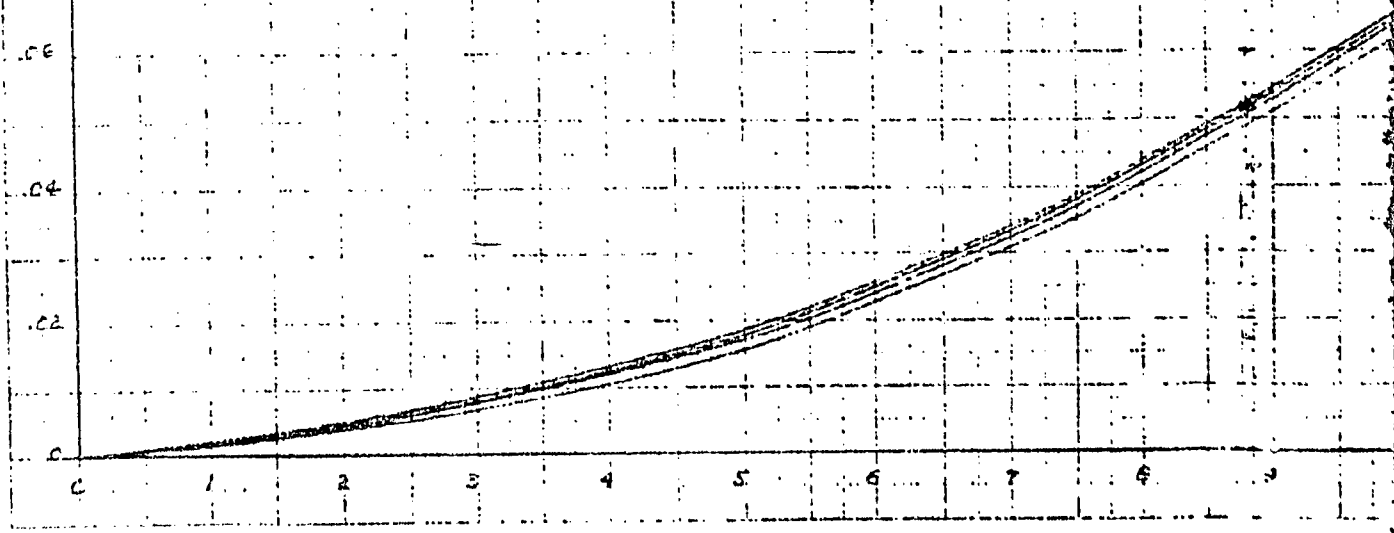
6

7

8

9

A



CD 43 01

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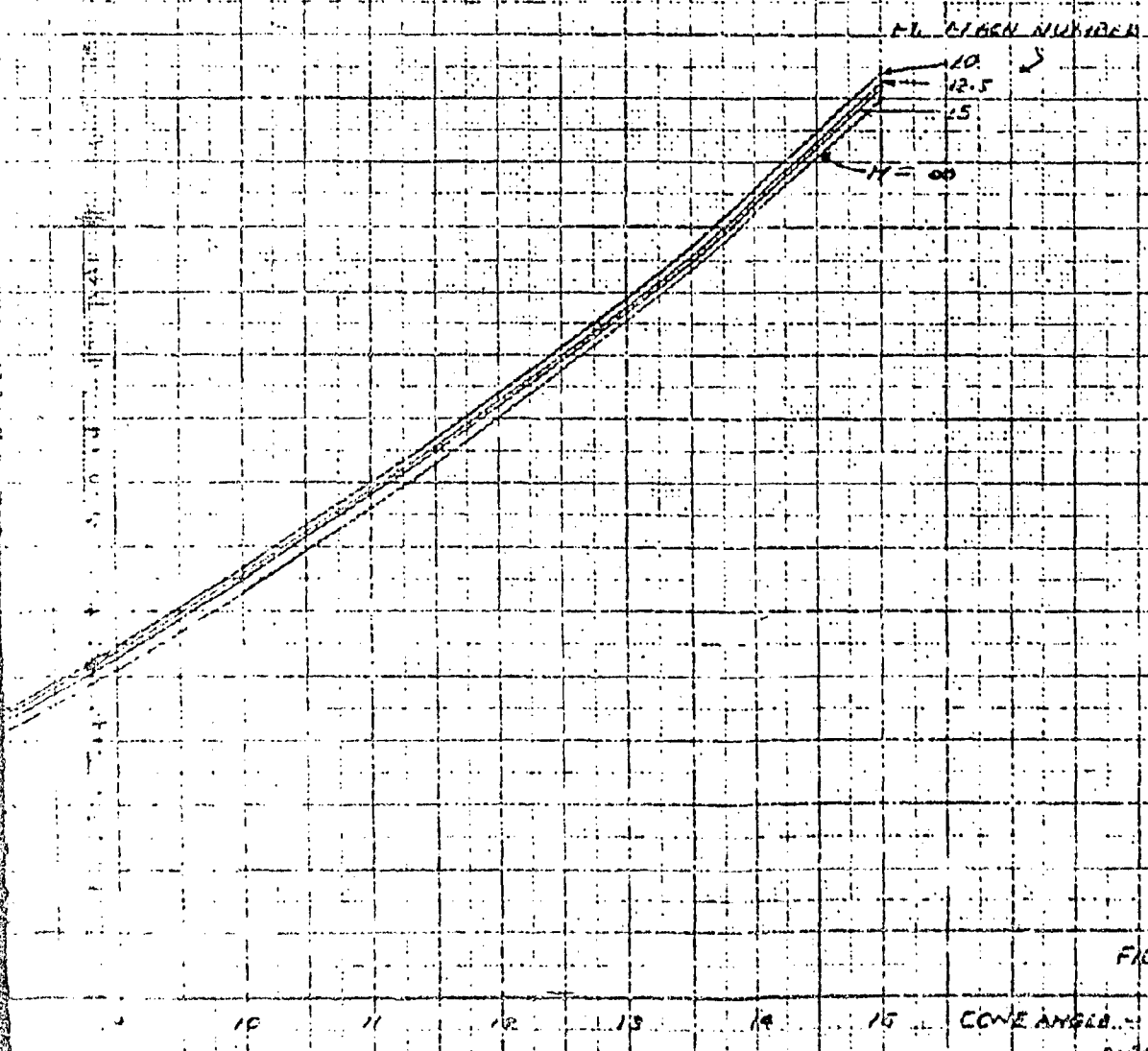


FIGURE 9.104

GU - REGRILL  
FIG. 9.104

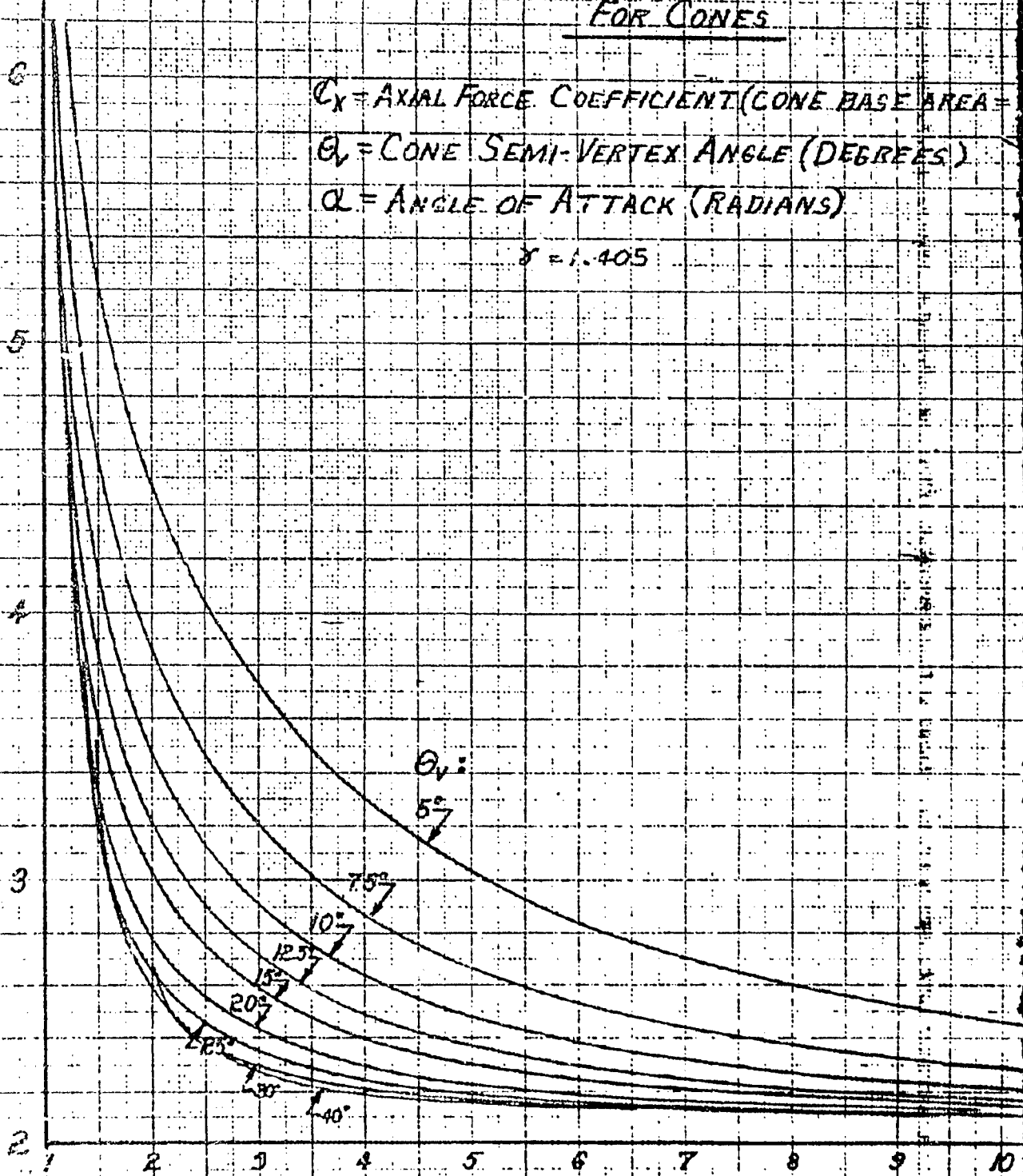
B

# INITIAL AXIAL FORCE COEFFICIENT/SIN FOR CONES

$C_x$  = AXIAL FORCE COEFFICIENT (CONE BASE AREA =  
 $\theta_v$  = CONE SEMI-VERTEX ANGLE (DEGREES)  
 $\alpha$  = ANGLE OF ATTACK (RADIAN)

$$\gamma = 1.405$$

INITIAL AXIAL FORCE COEFFICIENT/SIN  $\theta_v$



MACH NUMBER

$C_x / \sin^2 \theta$  vs MACH NUMBER

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(BASE AREA = REFERENCE AREA)

(DEGREES)

(VS)

NOTE:  $C_{x_{\alpha=0}} / \sin^2 \theta = 2.0$  FOR NEWTONIAN FLOW

$$[C_{x_{\alpha=0}} / \sin^2 \theta] = 2.091$$

$M = \infty$

9 10 11 12 13 14 15 16 17 18 19

NUMBER

FIGURE 3.11

FJD  
5-22-53

B

INITIAL NORMAL FORCE COEFFICIENT SLOPE /  $\cos^2 \theta_v$   
FOR CONES

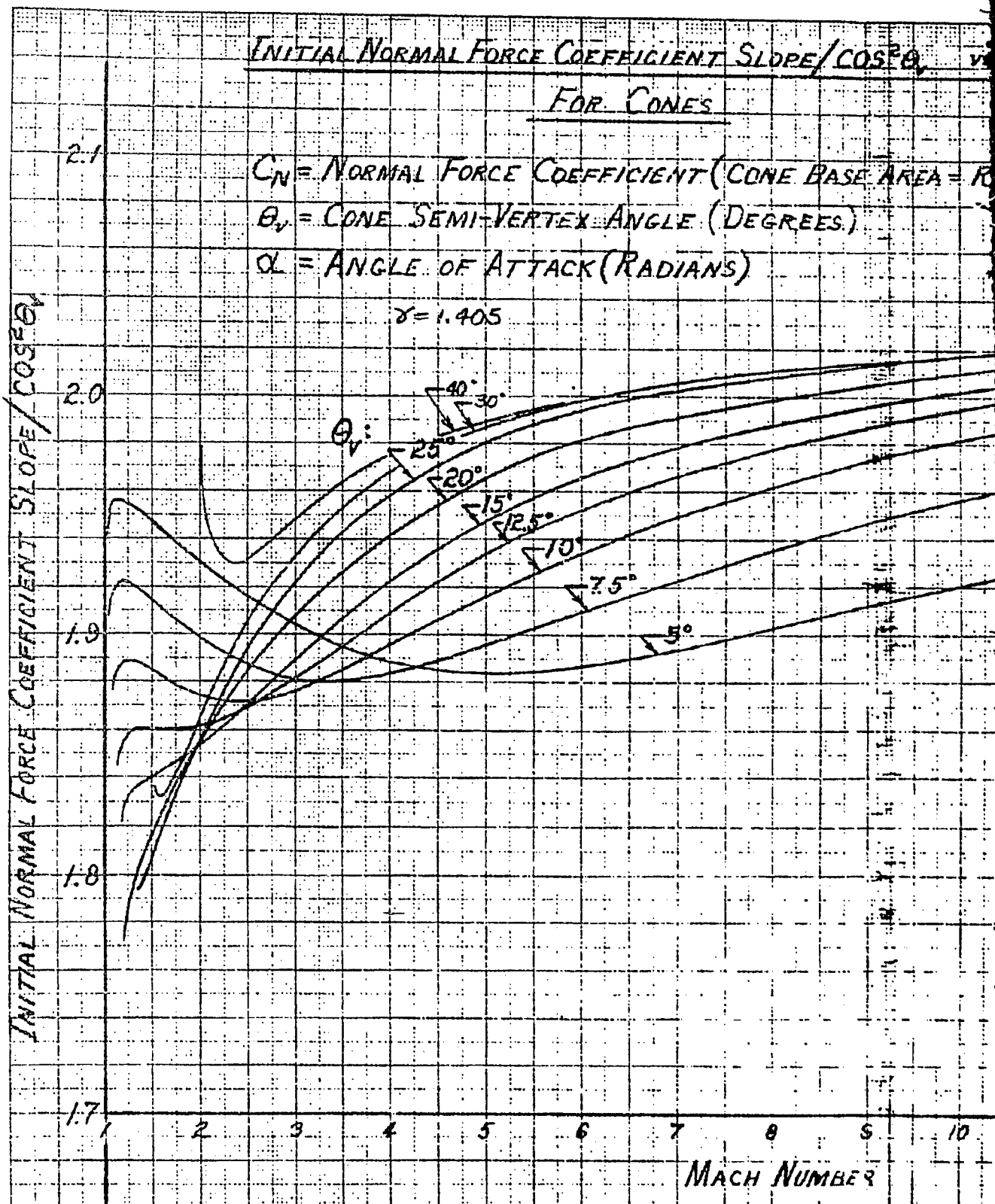
$C_N$  = NORMAL FORCE COEFFICIENT (CONE BASE AREA =  $R^2$ )

$\theta_v$  = CONE SEMI-VERTEX ANGLE (DEGREES)

$\alpha$  = ANGLE OF ATTACK (RADIAN)

$$\gamma = 1.405$$

INITIAL NORMAL FORCE COEFFICIENT SLOPE /  $\cos^2 \theta_v$



MACH NUMBER

A

$$\left(\frac{dC_N}{d\alpha}\right)_{\alpha=0} / \cos^2 \theta_v \text{ vs } M_1$$

$1/\cos^2 \theta_v$  vs MACH NUMBER

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(BASE AREA = REFERENCE AREA)

(ES)

$$\left[\left(\frac{dC_N}{d\alpha}\right)_{\alpha=0} / \cos^2 \theta_v\right]_{M=\infty} = 2.037$$

NOTE:  $(dC_N/d\alpha)_{\alpha=0} / \cos^2 \theta_v = 2.0$   
FOR NEWTONIAN FLOW

9 10 11 12 13 14 15 16 17 18 19

HER

FIGURE 3.12

FID  
3-22-54

B



# ADDITIONAL AXIAL FORCE COEFFICIENT DUE TO ANGLE OF FOR CONES FOR $\alpha \leq \theta_v$

$C_x$  = AXIAL FORCE COEFFICIENT (CONE BASE AREA = REFERENCE AREA)

$\theta_v$  = CONE SEMI-VERTEX ANGLE (DEGREES)

$\alpha$  = ANGLE OF ATTACK (RADIAN)

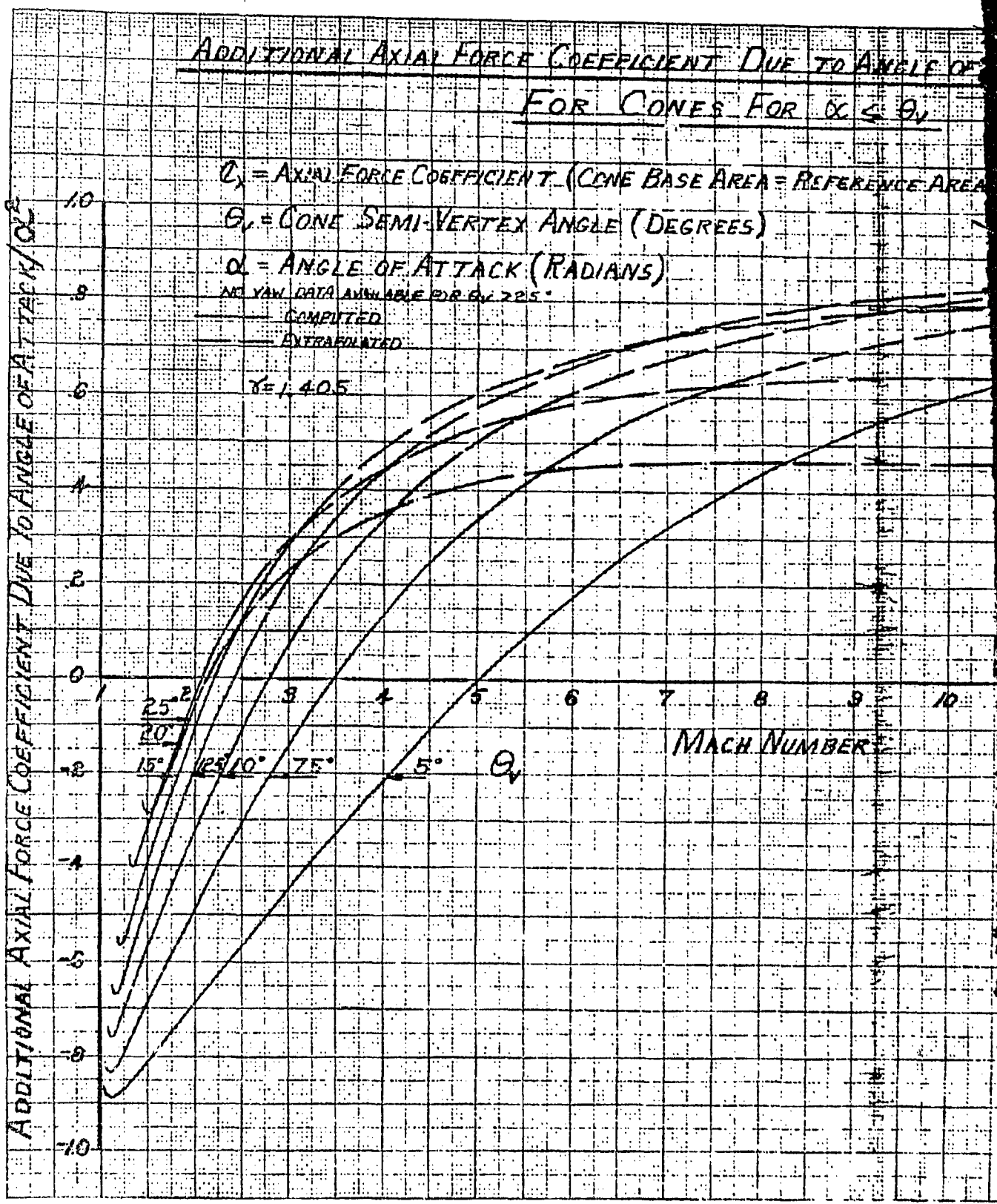
NO YAW DATA AVAILABLE FOR  $\theta_v \geq 22.5^\circ$

— COMPUTED

— EXTRAPOLATED

$\gamma = 1.405$

ADDITIONAL AXIAL FORCE COEFFICIENT DUE TO ANGLE OF ATTACK  $\alpha^2$



4

$\Delta C_x / \alpha^2$  vs  $M_1$

NO. 10-15 OF 1-10-1956 /  $\alpha^\circ$  vs MACH NUMBER

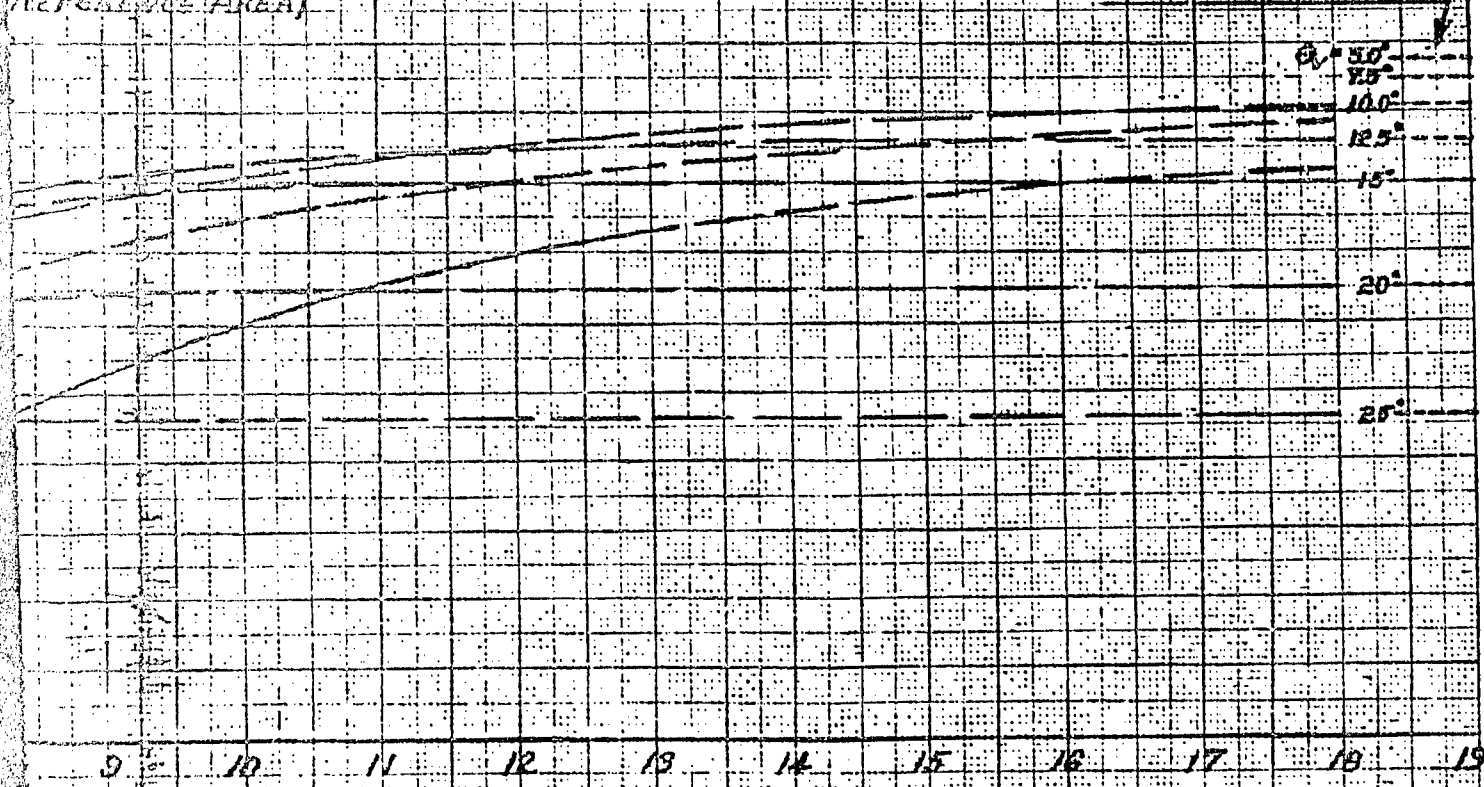
$\alpha \leq 20^\circ$

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NEWTONIAN FLOW:

$\Delta C_x / \sin^2 \alpha = 1.35 \sin^2 \alpha$

REFERENCE AREA



LIBERTY

FIGURE 3.13

B

1.0  
0.8  
0.6  
0.4  
0.2  
0.0  
-0.2  
-0.4  
-0.6  
-0.8  
-1.0

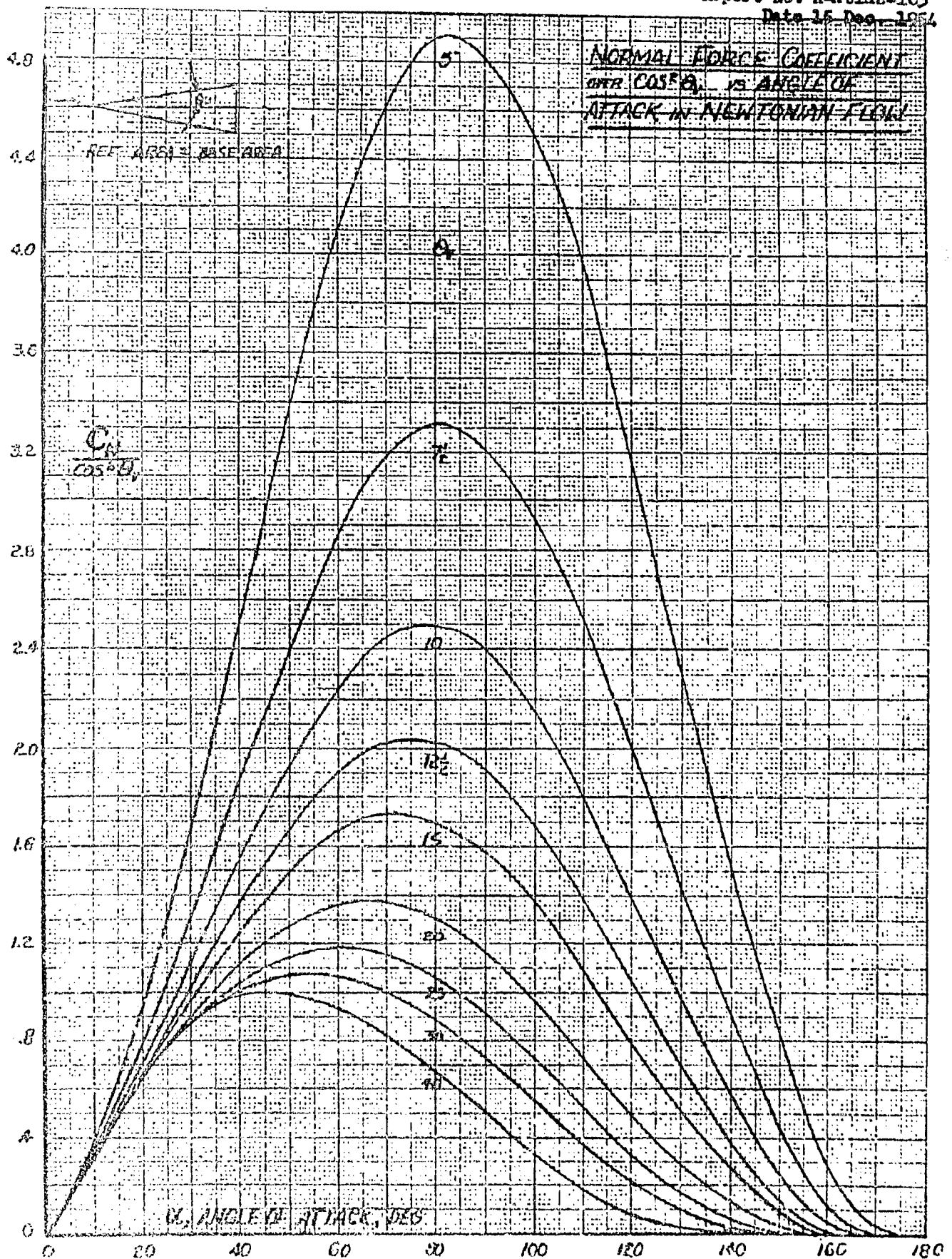


FIGURE 3.14

8

1. The normal force coefficient  $C_N$  is defined as the ratio of the normal force to the dynamic pressure  $q$  and the reference area  $A$ .

2. The normal force coefficient  $C_N$  is a function of the angle of attack  $\alpha$  and the shape of the body.

3

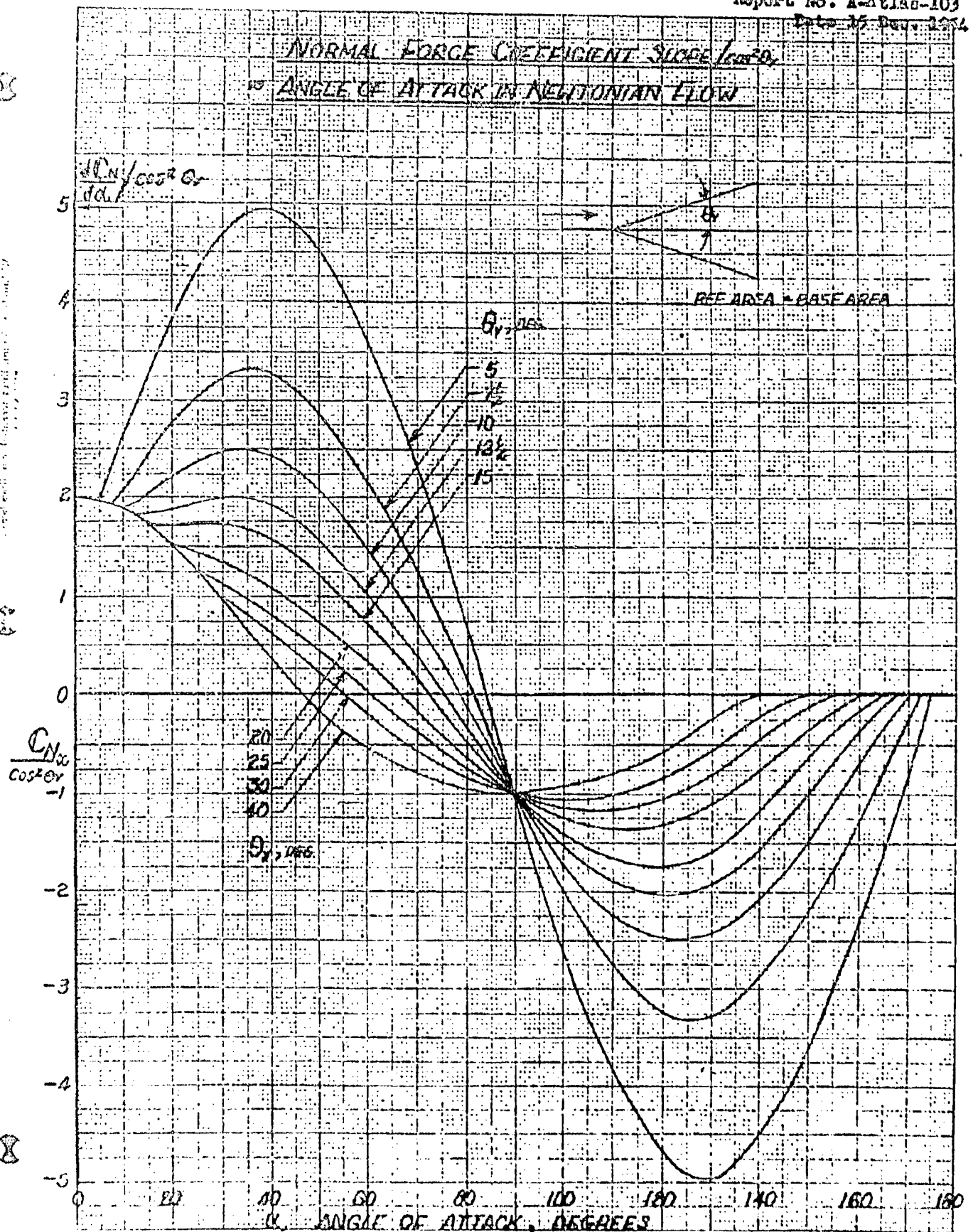


FIGURE 3.15

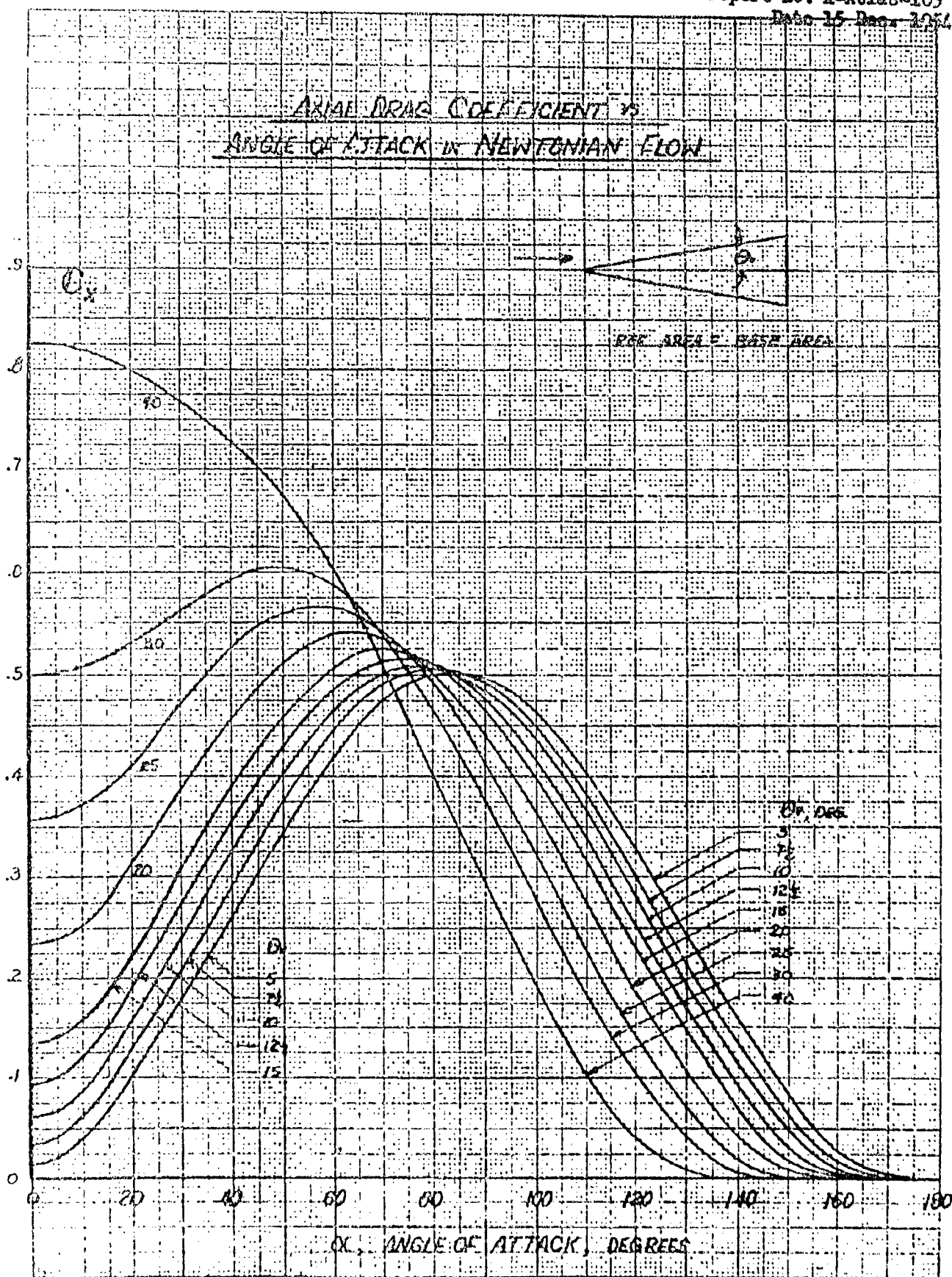
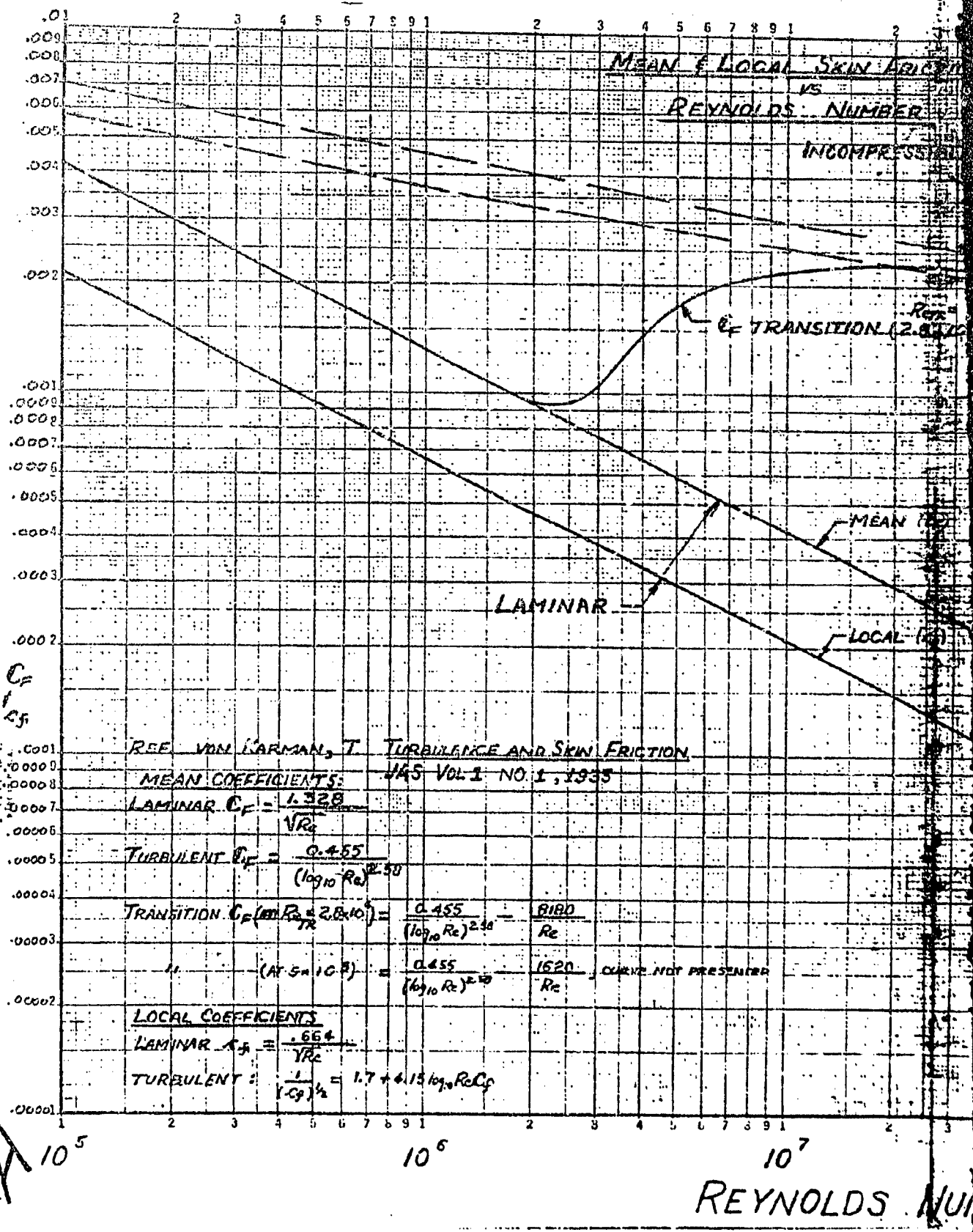


FIGURE 3.16







$C_F$  &  $C_f$  vs  $Re$

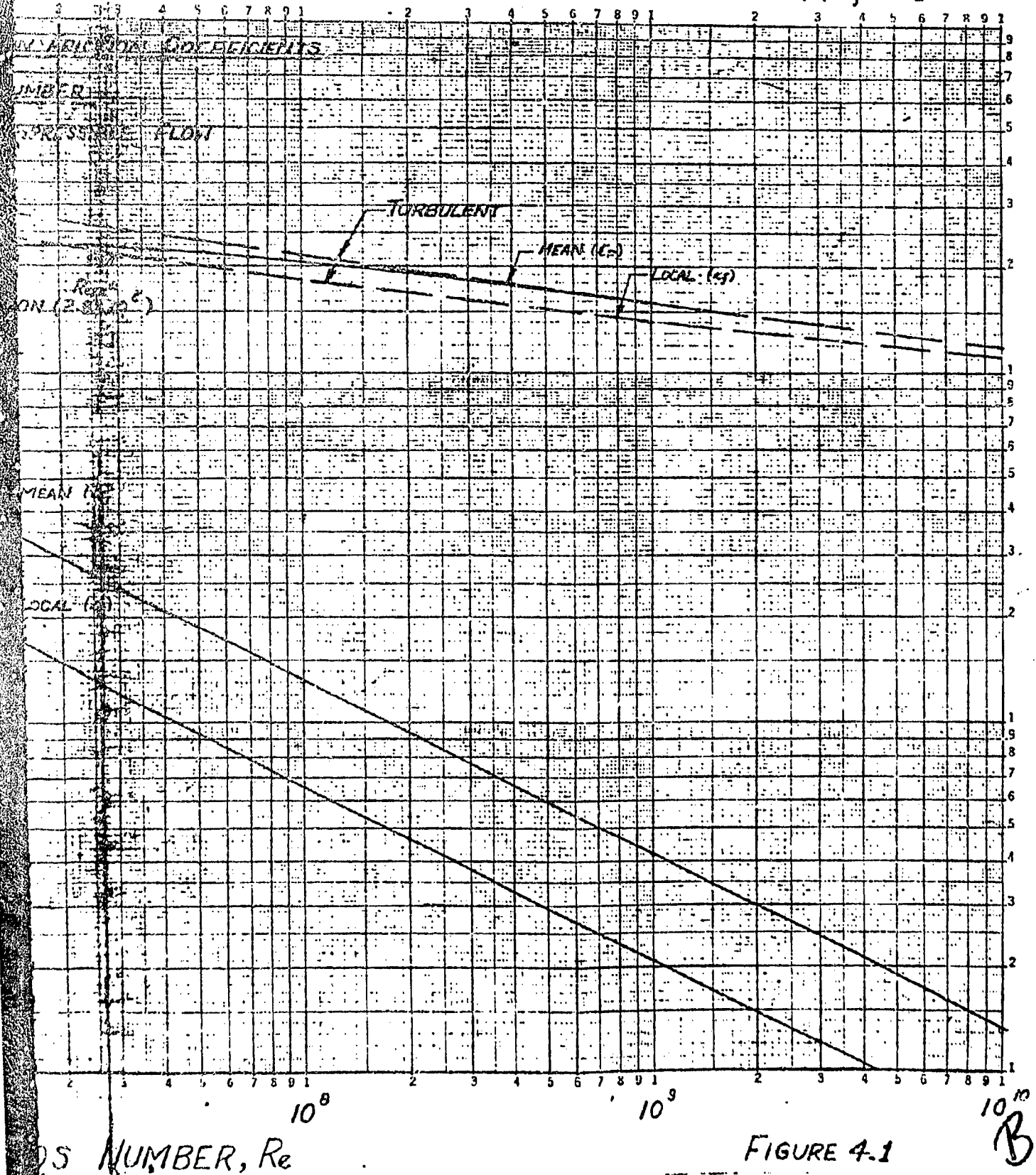


FIGURE 4.1

10<sup>10</sup>  
 B

NOTED  
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 REPORT  
 BY THE  
 RESEARCH  
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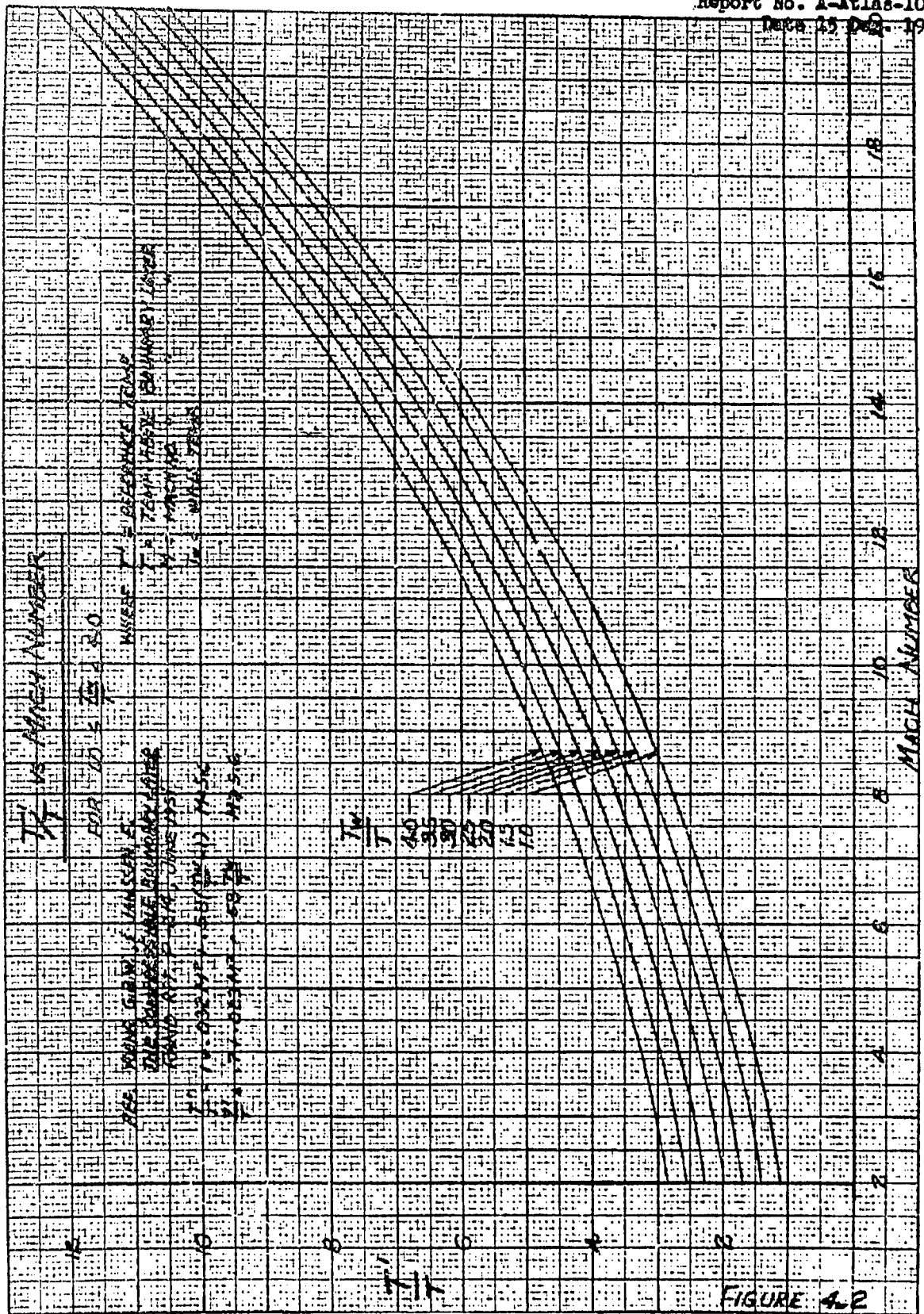
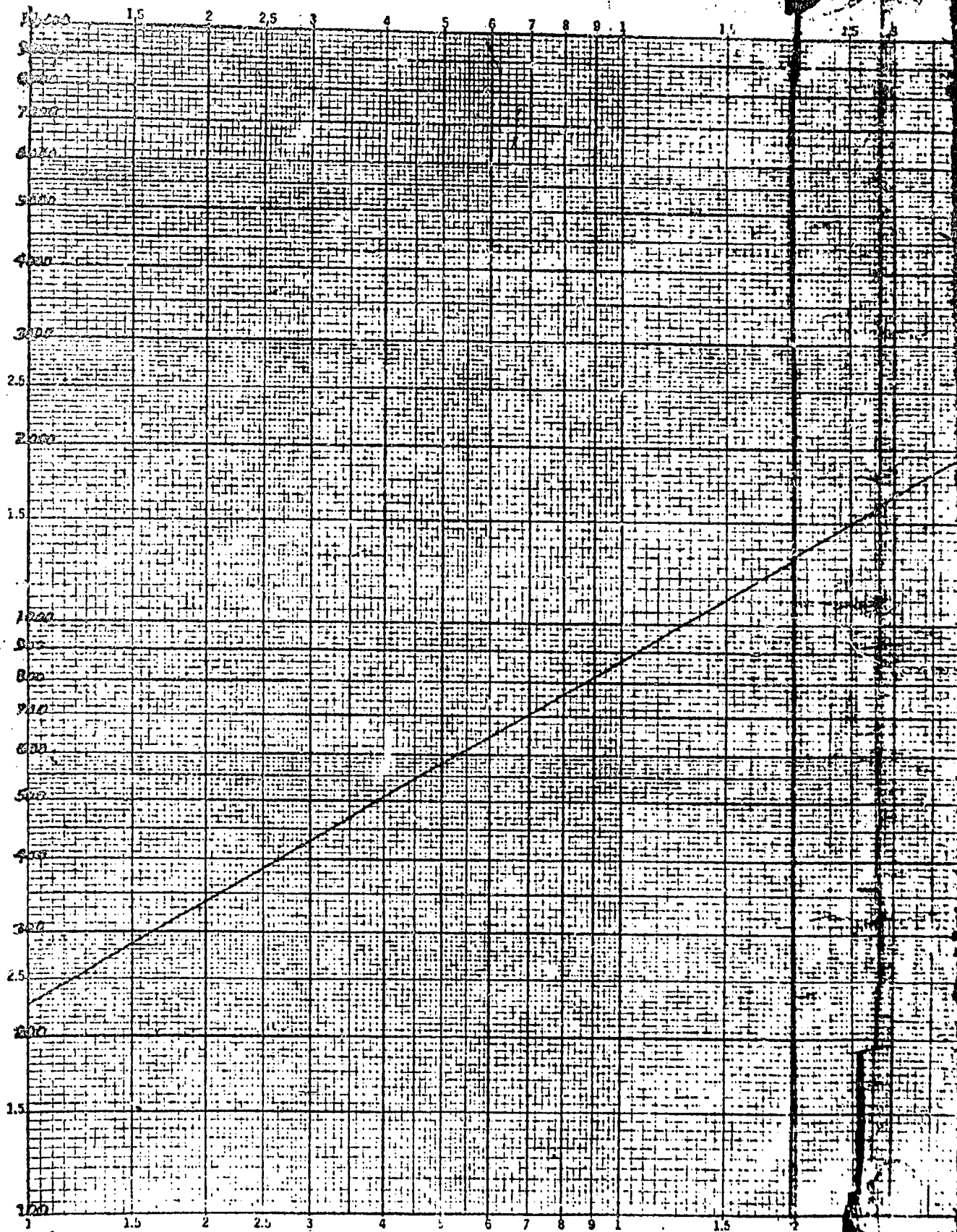


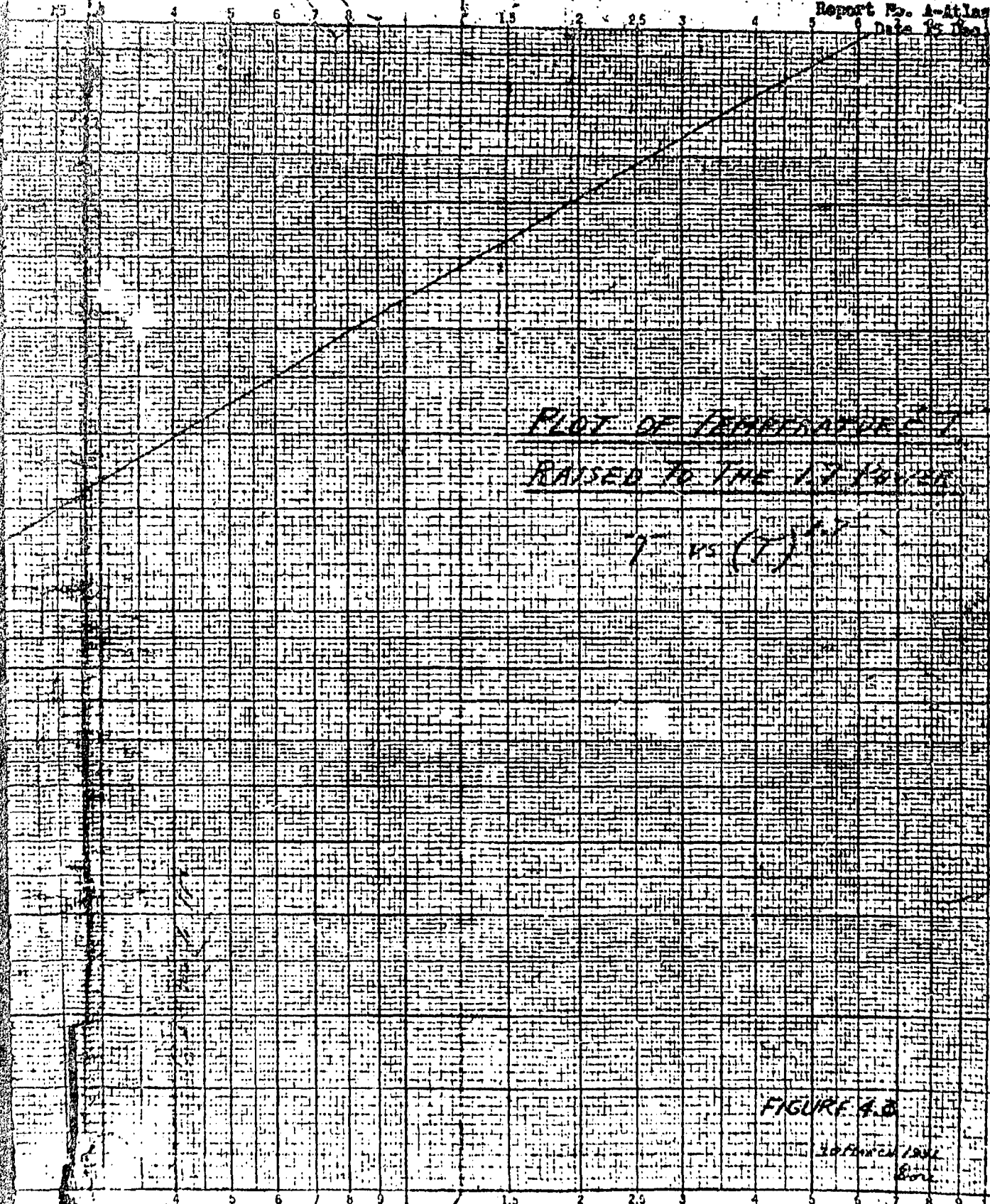
FIGURE 4.2

TEMPERATURE, T

REVUELL & ESSER CO., N.Y. NO. 204-112  
Inventors of the "Coulter"



105



PLOT OF TEMPERATURE  $T$   
RAISED TO THE 1/2 POWER

$T$  vs  $(T)^{1/2}$

FIGURE 4.3

30 JAN 1955  
B



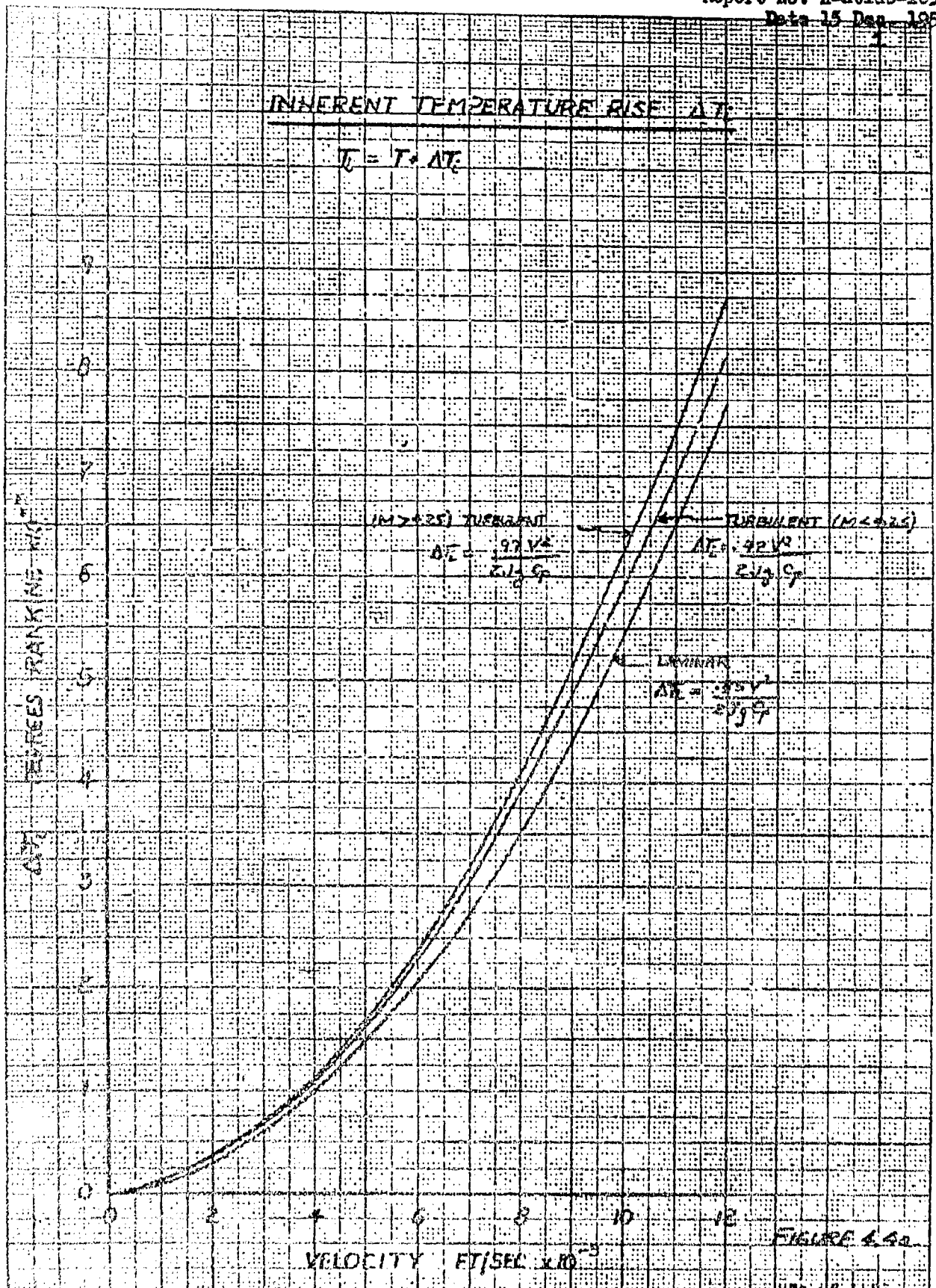
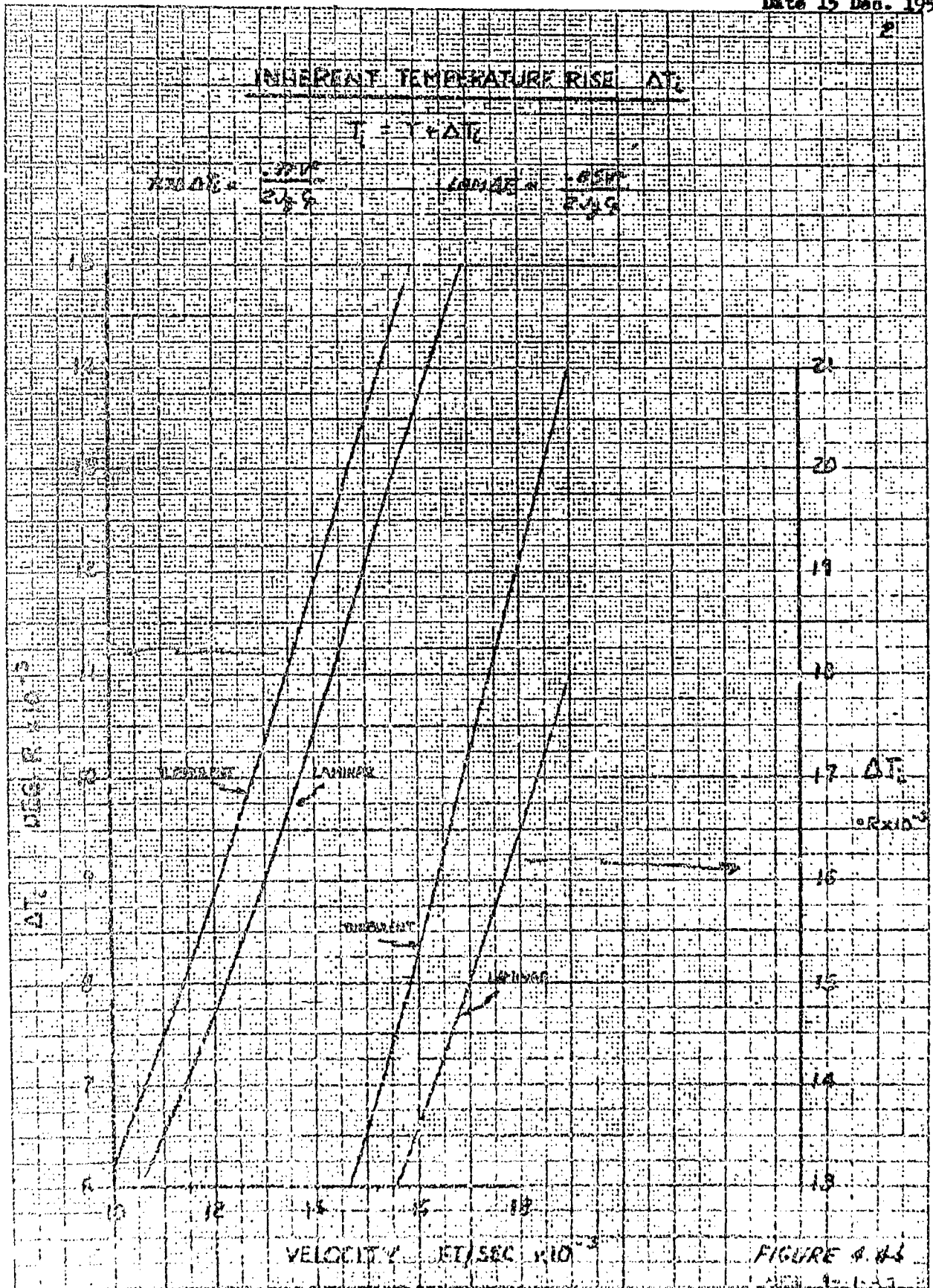
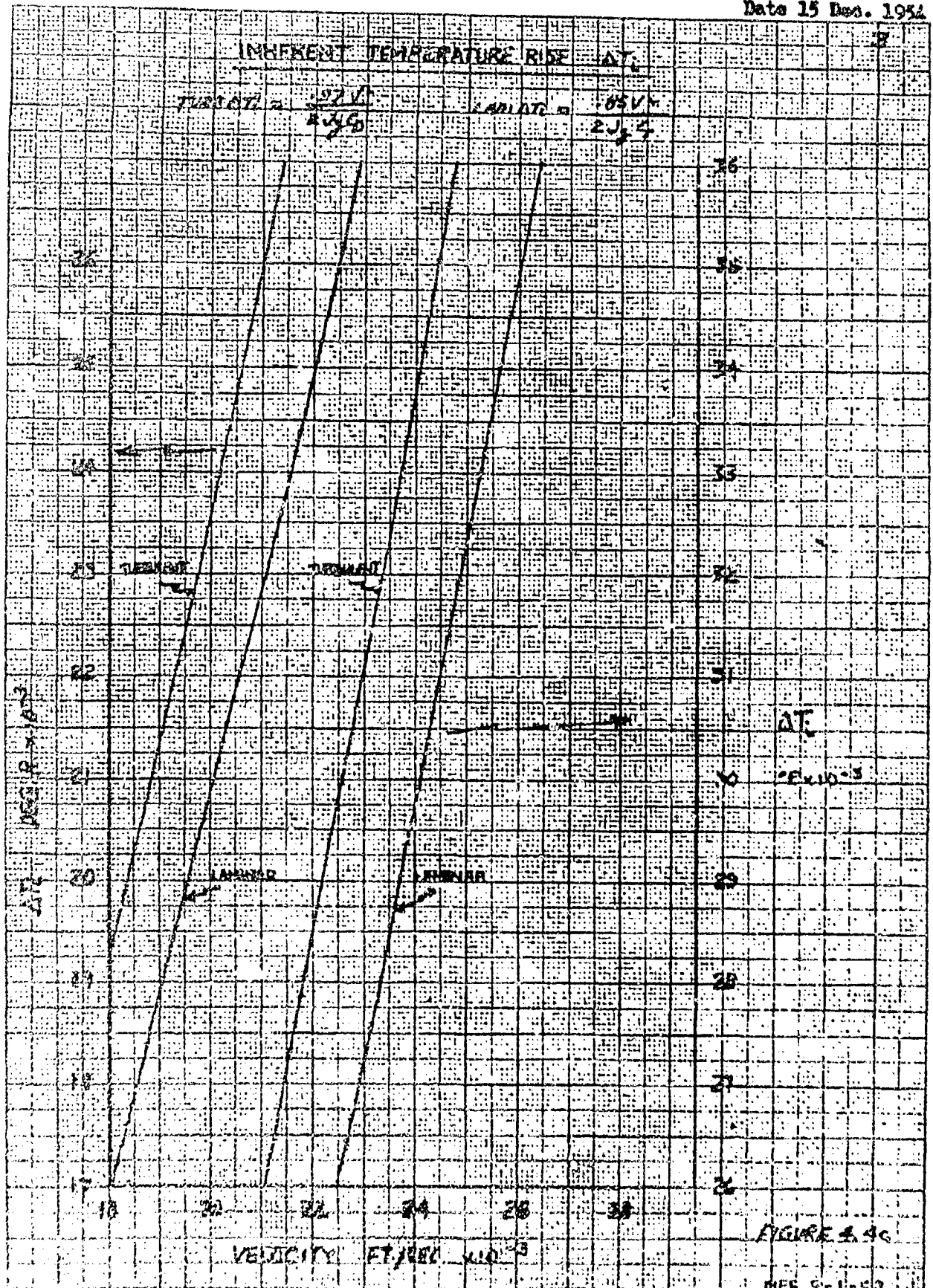


FIGURE 4.4a

REF-10-1-1







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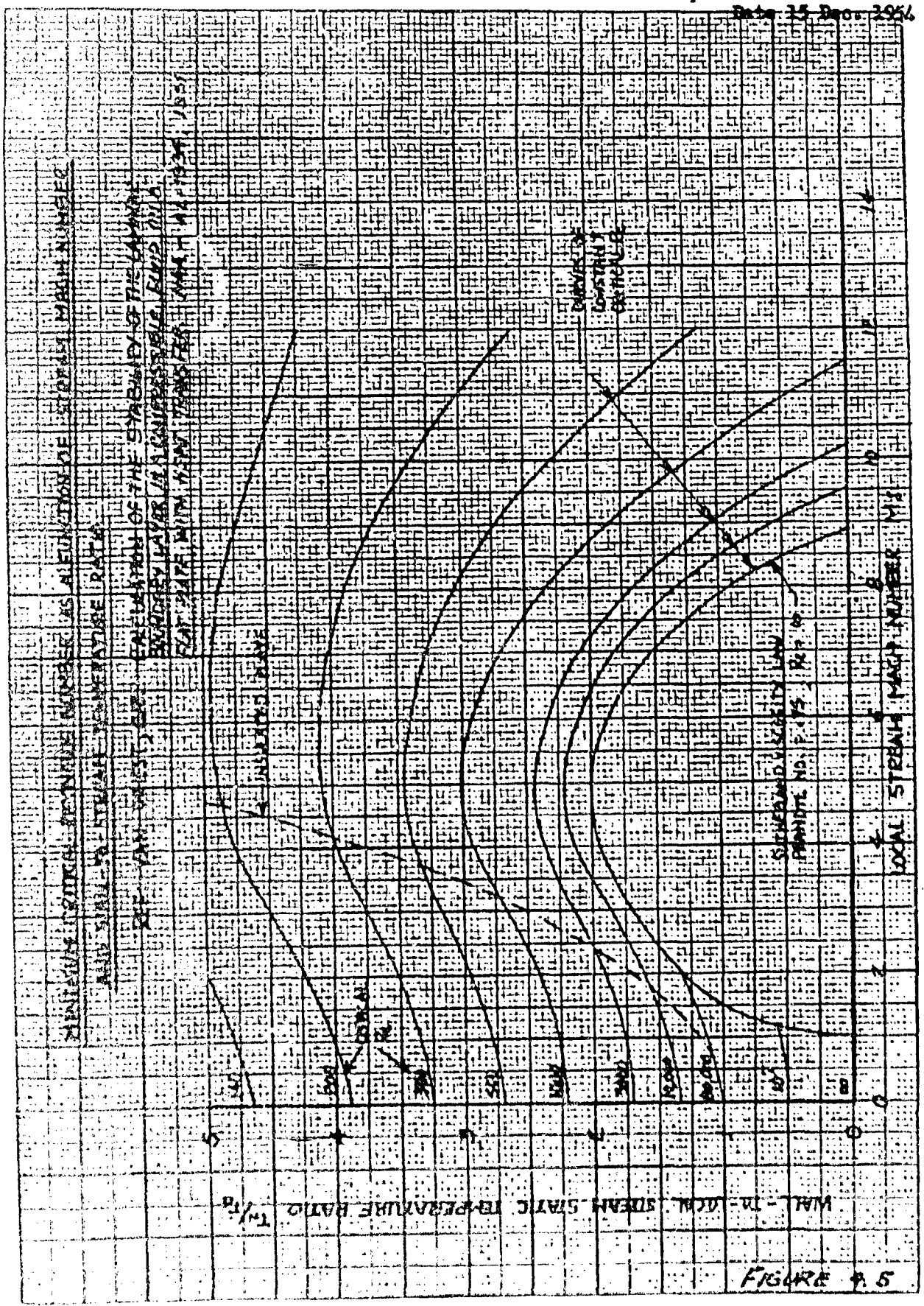


FIGURE 4.5

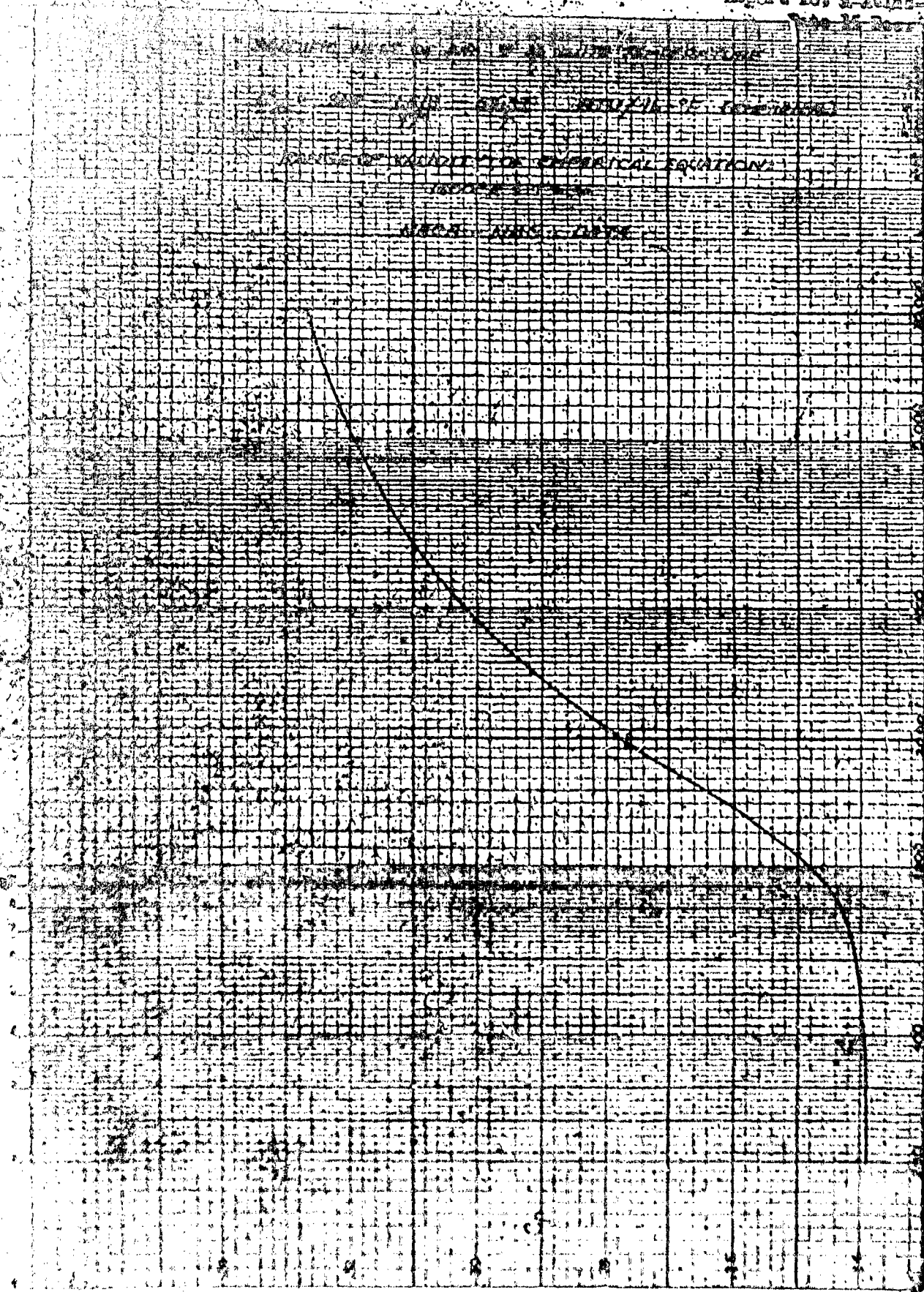
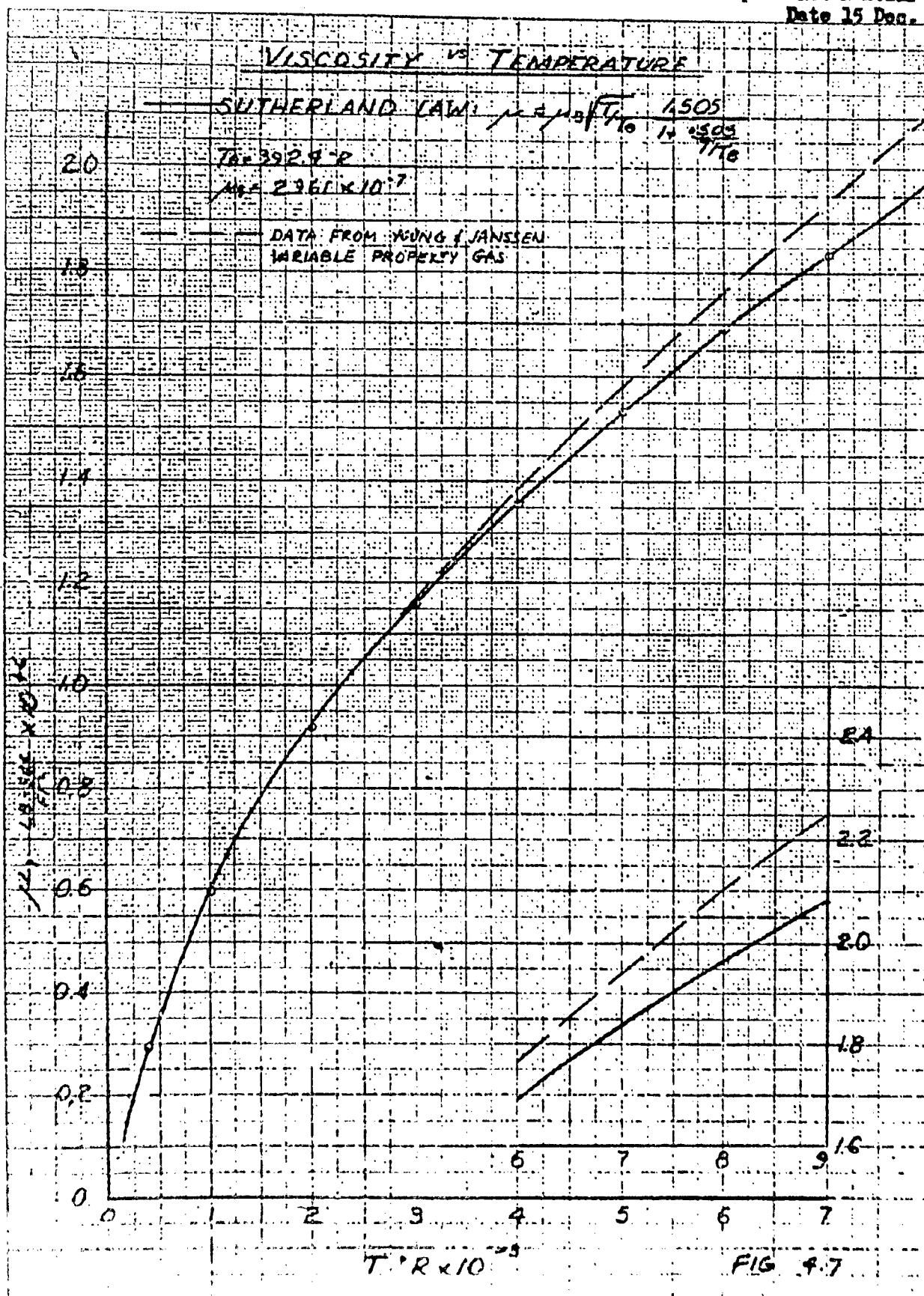


FIGURE 4.6



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 GEORGETOWN, DELAWARE